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CALCULATED HEAT TRANSFER AND COOLING SYSTEM PERFORMANCE, VOLUME II

LIFT FAN FLIGHT RESEARCH AIRCRAFT PROGRAM

CONTRACT NUMBER DA44-177-TC-713

GENERAL & ELECTRIC

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CALCULATED HEAT TRANSFER AND COOLING SYSTEM PERFORMANCE

Volume II

XV-5A LIFT FAN
FLIGHT RESEARCH AIRCRAFT PROGRAM
Contract DA 44-177-TC-715

June 1965

ADVANCED ENGINE AND TECHNOLOGY DEPARTMENT GENERAL ELECTRIC COMPANY CINCINNATI, OHIO 45215

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PREFACE

This report presents calculated heat transfer and cooling system performance for the U.S. Army XV-5A Lift Fan Research Aircraft. The report is submitted in two volumes, and this is Volume II.

Volume I contains the results of analysis and presents heat transfer and cooling performance characteristics. Volume II contains supporting data including test results providing the basis for estimates of external airframe heating, methods used in calculation of cooling system performance and an analysis of structural protection systems.

CONTENTS (Contained in Volume II)

9.0	APPI	ENDIX			259
	9.1	Referen	ices		259
	9.2	Symbols	s and Abbre	viations	261
	9.3	Cooling	System Ana	alysis	281
		9.3.1	Method o	f Approach	281
		9.3.2		Loss Analysis	282
			9.3.2.1 9.3.2.2	General Incompressible	282
			0.0.2.2	Flow	282
			9.3.2.3	Compressible Flow	284
		9.3.3	Boundary	Layer Bleed Duct	
			Aft Flap	per Position	286
		9.3.4	Tailpipe	Ejector Analysis	288
		9.3.5	Cooling A	Air Flow Between	
			the Nose	Fan and Wing Fan	
			Cavities	During Convention-	
			al Flight	Mode	288
		9.3.6	Cooling 1	Fan Outlet Total	
			Pressure		289
		9.3.7	Cooling A	Air Weight Flow	289
	9.4	Therma	l Analysis		383
		9.4.1	Cockpit A	Air Temperature	383
		9.4.2	Cooling 1	Fan Compartment	
			Inlet Por	t Air Temperature	
			- Turboj	et Mode	384
		9.4.3	Cooling I	Fan Compartment	
			Air Tem	perature	385
		9.4.4	Air Tem	perature Rise	
			Across th	ne Cooling Fans	386
		9.4.5		r Air Temperature	388
		9.4.6		perature Rise	
				ne Hydraulic Oil	
			Cooler		000

CONTENTS (Contained in Volume II)

	9.4.7	Engine Bay Inlet Air	
		Temperature	390
	9.4.8	Center Fuselage Air Tem-	
		perature Analysis - Lift Fan	
		Mode	391
	9.4.9	Lift Fan Cavity Air Tem-	
		perature - Turbojet Mode	400
	9.4.10	Wing Fan Ejector Air Tem-	
		perature During Forward	
		Fan Flight	401
	9.4.11	Engine Bay Heat Transfer	
		Analysis	406
	9.4.12	Aft Fuselage Heat Transfer	
		Analysis	423
9.5	Structur	al Protection Analysis	437
	9.5.1	Insulation of Nose Fan	
		Thrust Reverser Door Up-	
		per Closure Longeron As-	
		sembly Part No. 143F003	437
	9.5.2	Insulation Requirements for	
		Local Aircraft Surface Areas	
		9.5.2.1 Method of Analysis	442
9.6	NAGA-A	mes Test Data for Full Scale	
0.0		Model Test 177	458
	76.0 071. 7	70001 1050 177	
	9.6.1	Run Schedule	458
	9.6.2	Installation and Model	
		Photographs	466
	9.6.3	Engine Data	471
	9.6.4	Test 177 Summary - Ther-	
		mocouple Locations and	
		Identification	510
	9.6.5	Reduced Temperature Data	516
	9.6.6	Aircraft Temperatures	
		During XV-5A Model Tests	531
	9.6.7	40×80 Wind Tunnel and	
		Aircraft Operational Con-	
		trol Duta	550

TABLES (Contained in Volume I

	(Contained in Volume II)	
TABLE		PAGE
9.1	Cooling Air Duct Definition - Boundary Layer Bleed Duct to Engine Bay	290
9.2	Cooling Air Duct Definition - Large Cooling	
	Fan to Boundary Layer Bleed Duct	291
9.3	Cooling Air Duct Definition - Engine Bay to Tailpipe Ejector	292
9.4	Cooling Air Duct Definition - Small Cooling	
	Fan to Electronic Compartment	293
9.5	Cooling Air Duct Definition - L. H. Large	
	Cooling Fan to Center Fuselage	294
9.6	Cooling Air Duct Definition - R. H. Large	
	Cooling Fan to Center Fuselage	295
9.7	Cooling Air Duct Definition - Electronic Com-	
	partment to Nose Fan Air Ejector	296
9.8	Cooling Air Duct Definition - Center Fuselage	
1	to Flap Actuator Compartment	297
9.9	Cooling Air Duct Definition - Center Fuselage	
	to Wing Fan Air Ejectors	298
9.10	ARDC Standard Day and ANA Bulletin 421 Hot	
	Day Altitude Conditions Referenced to ARDC	
	Standard Day Sea Level	299
9.11	Insulation System Study Summany	444

RYAN 64B017

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LIST OF FIGURES (Contained in Volume II)

FIGURE		
9.1	Cooling Air Duct Definition - Boundary Layer	
	Bleed Duct to Engine Bay	300
9.2	Cooling Air Duct Definition - Large Cooling	
	Fan to Boundary Layer Bleed Duct	300
9.3	Cooling Air Duct Definition - Engine Bay to	
	Tailpipe Ejector	301
9.4	Cooling Air Duct Definition - Small Cooling Fan	
	to Electronic Compartment	302
9.5	Cooling Air Duct Definition - L. H. Large Cool-	
	ing Fan to Center Fuselage	303
9.6	Cooling Air Duct Definition - R. H. Large Cool-	
	ing Fan to Center Fuselage	303
9.7	Cooling Air Duct Definition - Electronic Com-	
	partment to Nose Fan Air Ejectors	304
9.8	Cooling Air Duct Definition - Center Fuselage	
	to Flap Actuator Compartment	304
9.9	Cooling Air Duct Definition - Center Fuselage	
	to Wing Fan Air Ejectors	305
9.10	Duct Pressure Loss - Cockpit to Cooling Fan	000
0.10	Compartment Vs Cooling Air Flow - Standard	
	Day Sea Level, and Hot Day 2,500 Ft.	306
9.11	Duct Pressure Loss - Cockpit to Cooling Fan	300
0.11	Compartment Vs Cooling Air Flow - Standard	
	Day 10,000 and 20,000 Ft., and Hot Day 10,000	
	Ft.	307
9.12	Duct Pressure Loss - Fuselage Ports to Cool-	301
0.12	ing Fan Compartment Vs Cooling Air Flow -	
	Standard Day Sea Level, and Hot Day 2,500 Ft.	308
9.13	Duct Pressure Loss - Fuselage Ports to Cool-	300
0.10	ing Fan Compartment Vs Cooling Air Flow -	
	Standard Day 10,000 and 20,000 Ft, and Hot	
	Day 10,000 Ft.	200
9.14	Duct Pressure Loss - Small Cooling Fan to	309
0.14	Generator Vs Cooling Air Flow - Standard Day	
		010
0 15	Sea Level, and Hot Day 2,500 Ft.	310
9.15	Duct Pressure Loss - Small Cooling Fan to	
	Generator Vs Cooling Air Flow - Standard Day	
0.10	10,000 and 20,000 Ft., and Hot Day 10,000 Ft.	311
9.16	Duct Pressure Loss - Small Cooling Fan to	
	Electronic Compartment Vs Cooling Air Flow -	010
	TENTON MALINIU WAS LAND ON A LIST TIME OF EAR THE	

FIGURE	(Contained in Volume 11)		
9.17	Duct Pressure Loss - Small Cooling Fan to Electronic Compartment Vs Cooling Air Flow - Standard Day 10,000 and 20,000 Ft., and Hot		
	Day 10,000 Ft.	313	
9.18	Duct Pressure Loss - L. H. Large Cooling Fan		
	to Center Fuselage Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.	314	
9.19	Duct Pressure Loss - L. H. Large Cooling Fan	014	
	to Center Fuselage Vs Cooling Air Flow -		
	Standard Day 10,000 and 20,000 Ft., and Hot		
	Day 10,000 Ft.	315	
9.20	Duct Pressure Loss - R. II. Large Cooling Fan		
	to Center Fuselage Vs Cooling Air Flow -	010	- International
9.21	Standard Day Sea Level, and Hot Day 2,500 Ft. Duct Pressure Loss - R. H. Large Cooling Fan	316	
	to Center Fuselage Vs Cooling Air Flow -		
	Standard Day 10,000 and 20,000 Ft., and Hot		
	Day 10,000 Ft.	317	4.3
9.22	Duct Pressure Loss - Large Cooling Fan to		
	Tailpipe Ejector Vs Cooling Air Flow - Stand-	hore harming	•
9.23	ard Day Sea Level, and Hot Day 2,500 Ft.	318	1
0.23	Duct Pressure Loss - Large Cooling Fan to Tailpipe Ejector Vs Cooling Air Flow - Stand-		
	ard Day 10,000 and 20,000 Ft., and Hot Day		
	10,000 Ft.	319	6
9.24	Duct Pressure Loss - Electronic Compartment		- 3
	to Nose Fan Ejector Vs Cooling Air Flow -		
	Standard Day Sea Level, and Not Day 2,500 Ft.	320	
9.25	Duct Pressure Loss - Center Fuselage to Flap		
	Actuator Compartment Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.	321	
9.26	Duct Pressure Loss - Center Fuselage to Wing		- 112
	Fan Ejector Vs Cooling Air Flow - Standard		
	Day Sea Level, and Hot Day 2,500 Ft.	322	1
9.27	Boundary Layer Bleed Duct Aft Flapper Posi-		
	tion Vs Aircraft Mach No Standard Day, Sea		
0.00	Level, 100% RPM	323	
9.28	Tailpipe Ejector Weight Flow Ratio Vs Primary		-
	and Secondary Pressure Ratio, $P_p/P_0 = 1.1$ to 1.5	324	
			0.

LIST OF FIGURES (Contained in Volume II)

FIGURE		
9.29	Tailpipe Ejector Weight Flow Ratio Vs Primary and Secondary Pressure Ratio, $P_D/P_O = 1.5$ to	
	3.2	325
9.30	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea	
0.00	Level, 100% RPM and Mach No. = 0, 0.1 and 0.2	326
9.31	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 100% RPM and Mach No. = 0.3, 0.4 and	007
0.00	0.5	327
9.32	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 100% RPM and Mach No. = 0.6, 0.7 and	
	0.8	328
9.33	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day	
	10,000 Ft., 100% RPM and Mach No. = 0, 0.2	
0.04	and 0.4	329
9.34	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day 10,000 Ft., 100% RPM and Mach No. = 0.6 and	
	0.8	330
9.35	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Hot Day, 10,000	
	Ft., 100% RPM and Mach No. = 0, 0.2 and 0.4	331
9.36	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Hot Day 10,000	
	Ft., 100% RPM and Mach No. = 0.6 and 0.8	332
9.37	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea	
9.38	Level, 95% RPM and Mach No. = 0, 0.1 and 0.2 Tailpipe Ejector Secondary Weight Flow Vs	333
	Secondary Pressure Ratio - Standard Day Sea	
	Level, 95% RPM and Mach No. = 0.3, 0.4 and 0.5	994
9.39		334
7. 37	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 95% RPM and Mach No. = 0.6, 0.7 and	
	0.8	335
9.40	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea	300
	Level, 85% RPM and Mach No. = 0, 0.1 and 0.2	336

LIST OF FIGURES (Contained in Volume II)

FIGURE		
9.41	Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 85% RPM and Mach No. = 0.3, 0.4 and	
9.42	0.5 Tailpipe Ejector Secondary Weight Flow Vs	337
	Secondary Pressure Ratio - Standard Day Sea Level, 85% RPM and Mach No. = 0.6, 0.7 and 0.8	338
9.43	Wing Fan Region Chordwise Pressure Distri-	339
9.44	Nose Fan Doors - Pressure Distribution and Leakage Area - Mach No. = 0.8	340
9.45	Wing Fan Cavity Pressure Vs Air Flow Between Fan Cavity and Outside and Mach No	
9.46	Standard Day, Sea Level Nose Fan Cavity Pressure Vs Air Flow Be- tween Fan Cavity and Outside, and Mach No	341
9.47	Standard Day, Sea Level Pressure Loss Vs Weight Flow in the Lift Fan	342
9.48	Supply Ducts from the Nose Fan Cavity to the Wing Fan Cavity - Standard Day Sea Level Large Cooling Fan Exhaust Total Pressure Vs	343
	Flow Rate and Plenum Chamber Pressure - Standard Day, Sea Level, 100% RPM	344
9.49	Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure - Hot Day, 2,500 Ft., 100% RPM	345
9.50	Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure - Standard Day, Sea Level, 95% RPM	346
9.51	Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -	0.0
9.52	Standard Day, Sea Level, 85% RPM Large Cooling Fan Exhaust Total Pressure Vs	347
9.53	Flow Rate and Plenum Chamber Pressure - Standard Day, Sea Level, 75% RPM Large Cooling Fan Exhaust Total Pressure Vs	348
	Flow Rate and Plenum Chamber Pressure -	349

LIST OF FIGURES (Contained in Volume II)

FIGURE		
9.54	Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -	
	Standard Day, 20,000 Ft., 100% RPM	35
9.55	Large Cooling Fan Exhaust Total Pressure Vs	00
5.00	Flow Rate and Plenum Chamber Pressure -	
	Hot Day, 10,000 Ft., 100% RPM	35
9.56	Small Cooling Fan Exhaust Total Pressure Vs	00
9.00	Flow Rate and Plenum Chamber Pressure -	
	Standard Day, Sea Level, 100% RPM	35
9.57	Small Cooling Fan Exhaust Total Pressure Vs	00.
3.31	Flow Rate and Plenum Chamber Pressure -	
	Hot Day, 2,500 Ft., 100% RPM	35
9.58	Small Cooling Fan Exhaust Total Pressure Vs	500
0.0 0	Flow Rate and Plenum Chamber Pressure -	
	Standard Day, Sea Level, 95% RPM	354
9.59	Small Cooling Fan Exhaust Total Pressure Vs	00
2,00	Flow Rate and Plenum Chamber Pressure -	
	Standard Day, Sea Level, 85% RPM	35
9.60	Small Cooling Fan Exhaust Total Pressure Vs	
	Flow Rate and Plenum Chamber Pressure -	
	Standard Day, Sea Level, 75% RPM	350
9.61	Small Cooling Fan Exhaust Total Pressure Vs	
	Flow Rate and Plenum Chamber Pressure -	
	Standard Day, 10,000 Ft., 100% RPM	357
9.62	Small Cooling Fan Exhaust Total Pressure Vs	-
	Flow Rate and Plenum Chamber Pressure -	
	Standard Day, 20,000 Ft., 100% RPM	358
9.63	Small Cooling Fan Exhaust Total Pressure Vs	
	Flow Rate and Plenum Chamber Pressure -	
	Hot Day, 10,000 Ft., 100% RPM	359
9.64	Cooling Air Weight Flow - Cockpit to Cooling	
	Fan Compartment Vs Fuselage Pressure and %	
	RPM - Fan Mode, Standard Day, Sea Level	360
9.65	Cooling Air Weight Flow - Fuselage Ports to	
	Cooling Fan Compartment Vs Fuselage Pres-	
	sure and % RPM - Fan Mode, Standard Day,	
	Sea Level	360
9.66	Cooling Air Weight Flow - Small Cooling Fan to	4.
	Electronic Compartment Vs Fuselage Pressure	
	and % RPM - Fan Mode Standard Day See Level	361

FIGURE		
9.67	Cooling Air Weight Flow - Small Cooling Fan to Generator Vs Fuselage Pressure and % RPM -	
	Fan Mode, Standard Day, Sea Level	. 361
9.68	Cooling Air Weight Flow - L. H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea	
	Level	362
9.69	Cooling Air Weight Flow - R. H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level	362
9.70	Cooling Air Weight Flow - Large Cooling Fan to Tailpipe Ejector Vs Fuselage Pressure and	
9.71	% RPM - Fan Mode, Standard Day, Sea Level Cooling Air Weight Flow - Center Fuselage to Wing Fan Cavity Vs Fuselage Pressure and %	363
	RPM - Fan Mode, Standard Day, Sea Level	363
9.72	Cooling Air Weight Flow - Forward Fuselage to Nose Fan Cavity Vs Fuselage Pressure and %	
9.73	RPM - Fan Mode, Standard Day, Sea Level Cooling Air Weight Flow - Wing and Nose Fan Ejectors and Flap Actuator Slot to Outside Vs Fuselage Pressure and % RPM - Fan Mode,	364
	Standard Day, Sea Level	364
9.74	Cooling Air Weight Flow - Balance of Flow Thru The Lower Fuselage Vs Fuselage Pressure and	
9.75	% RPM - Fan Mode, Standard Day, Sea Level Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day, Sea	365
	Level	366
9.76	Cooling Air Weight Flow - Fuselage Ports to Cooling Fan Compartment Vs Fuselage Pres-	
	sure and % RPM - Conventional Mode, Standard	
	Day, Sca Level	366
9.77	Cooling Air Weight Flow - Small Cooling Fan to Electronic Compartment Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day,	
	See Level	387

FIGURE		
9.78	Cooling Air Weight Flow - Small Cooling Fan to Generator Vs Fuselage Pressure and % RPM -	
9.79	Conventional Mode, Standard Day, Sea Level Cooling Air Weight Flow - L.H. Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day, Sea	367
	Level	368
9.80	Cooling Air Weight Flow - R.H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day,	
	Sea Level	368
9.81	Cooling Air Weight Flow - Large Cooling Fan to Tailpipe Ejector Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day, Sea	
	Level	369
9.82	Cooling Air Weight Flow - Wing and Nose Fan Ejectors and Flap Actuator Slot to Outside Vs Fuselage Pressure and % RPM - Conventional	
9.83	Mode, Standard Day, Sea Level Cooling Air Weight Flow Balance of Flow Thru The Lower Fuselage Vs Fuselage Pressure and	369
	% RPM - Conventional Mode, Standard Day,	050
9.84	Sea Level Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day,	370
	Sea Level, Mach No. = 0.2	370
9.85	Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.4	371
9.86	Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard	311
9.87	Day, Sea Level, Mach No. = 0.6 and 0.8 Cooling Air Weight Flow - Fuselage Ports to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode,	371
	Standard Day, Sea Level, Mach No. = 0.2	372

FIGURE		
9.88	Cooling Air Weight Flow - Fuselage Ports to	
	Cooling Fan Compartment Vs Fuselage Pres-	
	sure and % RPM - Conventional Flight Mode,	
	Standard Day, Sea Level, Mach No. = 0.4	373
9.89	Cooling Air Weight Flow - Fuselage Ports to	
	Cooling Fan Compartment Vs Fuselage Pres-	
	sure % RPM - Conventional Flight Mode,	
	Standard Day, Sea Level, Mach No. = 0.6 and	
	0.8	373
9.90	Cooling Air Weight Flow - Small Cooling Fan to	
	Electronic Compartment Vs Fuselage Pressure	
	and % RPM - Conventional Flight Mode, Stand-	
	ard Day, Sea Level, Mach No. = 0.2 and 0.4	374
9.91	Cooling Air Weight Flow - Small Cooling Fan to	
	Electronic Compartment Vs Fuselage Pressure	
	and % RPM - Conventional Flight Mode, Stand-	
	ard Day, Sea Level, Mach No. = 0.6 and 0.8	374
9.92	Cooling Air Weight Flow - Small Cooling Fan to	
	Generators Vs Fuselage Pressure and % RPM -	
	Conventional Flight Mode, Standard Day, Sea	
	Level, Mach No. = 0.2 and 0.4	375
9.93	Cooling Air Weight Flow - Small Cooling Fan to	
	Generators Vs Fuselage Pressure and % RPM -	
	Conventional Flight Mode, Standard Day, Sea	
	Level, Mach No. = 0.6 and 0.8	375
9.94	Cooling Air Weight Flow - L. H. Large Cooling	
	Fan to Center Fuselage Vs Fuselage Pressure	
	and % RPM - Conventional Flight Mode, Stand-	
	ard Day, Sea Level, Mach No. = 0.2 and 0.4	376
9.95	Cooling Air Weight Flow - L. H. Large Cooling	
	Fan to Center Fuselage Vs Fuselage Pressure	
	and % RPM - Conventional Flight Mode, Stand-	
	ard Day, Sea Level, Mach No. = 0.6 and 0.8	376
9.96	Cooling Air Weight Flow - R. H. Large Cooling	
	Fan to Center Fuselage Vs Fuselage Pressure	
	and % RPM - Conventional Flight Mode, Stand-	
	ard Day, Sea Level, Mach No. = 0.2 and 0.4	377

FIGURE		
9.97	Cooling Air Weight Flow - R. H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure	
	and % RPM - Conventional Flight Mode, Stand-	0.55
9.98	ard Day, Sea Level, Mach No. = 0.6 and 0.8	377
9.90	Cooling Air Weight Flow - Large Cooling Fans	
	to Engine Bay Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard	
	Day, Sea Level, Mach No. = 0.2 and 0.4	378
9.99	Cooling Air Weight Flow - Center Fuselage to	310
J. JJ	Wing Fan Air Ejectors Vs Fuselage Pressure	
	and Mach No Conventional Flight Mode,	
	Standard Day, Sea Level	379
9.100	Cooling Air Weight Flow - Center Fuselage to	0.0
	Nose Fan Air Ejectors Vs Fuselage Pressure	
	and Mach No Conventional Flight Mode,	
	Standard Day, Sea Level	379
9.101	Cooling Air Weight Flow - Outside to Nose Fan	
	Cavity Vs Fuselage Pressure and Mach No	
	Conventional Flight Mode, Standard Day, Sea	
	Level	380
9.102	Cooling Air Weight Flow - Balance of Flow Into	
	and Out of the Lower Fuselage Vs Fuselage	
	Pressure and % RPM - Conventional Flight	
	Mode, Standard Day, Sea Level, Mach No.	
	= 0.2 and 0.4	381
9.103	Cooling Air Weight Flow - Balance of Flow Into	
	and Out of the Lower Fuselage Vs Fuselage	
	Pressure and % RPM - Conventional Flight	
	Mode, Standard Day, Sea Level, Mach No.	
	= 0.6 and 0.8	382
9.104	Heating and Cooling Schematic - Center Fuse-	
	lage Ducting	392
9.105	Center Fuselage Air Temperature Rise Vs Re-	
	circulation of Fuselage Air Between Supply	
	Duct and Shroud - Fan Mode	399
0 100	P-P s. s a	404
9.106	Wing Fan Forward Air Ejector - Tc S Vs s a qs	404
	Р -Р	
9.107	Wing Fan Aft Air Ejector - T S Vs S a	405

LIST OF FIGURES (Contained in Volume II)

FIGURE		
9.108	Heating and Cooling Schematic - Engine Bay	407
9.109	Forward and Aft Turbine Casing Heat Flow	
	Schematic	407
9.110	Diverter Valve Heat Flow Schematic	422
9.111	Bellows Heat Flow Schematic	422
9.112	Heating and Cooling Schematic - Aft Fuselage	424
9.113	Heating and Cooling Schematic - Aft Fuselage	
	Sections	425
9.114	Aft Fuselage Heat Balance Schematic	426
9.115	Part No. 143F003 Heat Transfer Model	437
9.116	Heat Transfer Model for Insulated Metal Skin	445
9.117	Comparison of Predicted and Experimental In-	
	sulated Panel Temperatures: 0.545" Min K In-	
	sulation on . 025" Titanium	446
9.118	Comparison of Predicted and Experimental In-	
	sulated Panel Temperatures: 0.723" Min K In-	
	sulation on . 025" Titanium	447
9.119	Skin Temperature-Time Profiles Vs Gas Tem-	
	perature 0.25" Min K Insulation on .025"	
	Titanium	448
9.120	Skin Temperature-Time Profiles Vs Gas Tem-	
	peratures 0.375" Min K Insulation on .025"	
.	Titanium	449
9.121	Skin Temperature-Time Profiles Vs Gas Tem-	
	peratures 0.50" Min K Insulation on .025"	
0.400	Titanium	450
9.122	Skin Temperature Vs Insulation Thickness and	4
0.100	Gas Temperature After 5 Minutes Exposure	451
9.123	Skin Temperature-Time Profiles Vs Gas Tem-	
	perature 0.25" Min K Insulation on .040" Aluminum	450
0.104		452
9.124	Skin Temperature-Time Profiles Vs Gas Tem-	
	perature 0.375" Min K Insulation on .040" Aluminum	453
0.105		403
9.125	Skin Temperature-Time Profiles Vs Gas Tem-	
	perature 0. 500"Min K Insulation on . 040"	464
0.100	Aluminum	454
9.126	Skin Temperature-Time Profiles Vs Gas Tem-	
	perature 0, 625"Min K Insulation on . 040"	451
	Aluminum	455

LIST OF FIGURES (Contained in Volume II)

FIGURE		
9.127	Skin Temperature Vs Insulation Thickness and	
	Exposure Time; Gas Temperature 1000° F, and	
	Aluminum Skin	456
9.128	Skin Temperature Vs Insulation Thickness and	
	Exposure Time; Gas Temperature 715° F,	
	Aluminum Skin	457

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9.0 APPENDIX

9.1 REFERENCES

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9.2 SYMBOLS AND ABBREVIATIONS

Symbol	Description
A	Heat Transfer Area Normal to Direction of Flow
ADUCT	Cross-Sectional Area of Duct Branch in Question
A _f	Radiative Heat Transfer Area Factor
A _F	Wing Fan Area
A _{fF}	Area of Duct Facing Floor of Engine Bay
A Flapper	Area of Boundary Layer Bleed Duct Flapper
A _{fp}	Area of Duct Facing Honeycomb Panel
A _{fw}	Area of Duct Facing Inside Vertical Firewall
A _m	Cross-Sectional Area at Station m of Duct
A _{m+1}	Cross-Sectional Area at Station m+1 of Duct
A _{m+2}	Duct Area at 2nd Section From Section n
A _p	Inside Area Honeycomb Panel
A _R	Duct Area of the R th Section of Duct
A _S	Tailpipe Shroud Area
A _T	Heat Transfer Area of Duct or Turbine Casing
A _{T'}	Area of Tailpipe

Symbol	Description
A _W	Area of Vertical Firewall
A ₃	Heat Transfer Area of Power Distribution Ducting
A ₄	Heat Transfer Area of Fiberglass Shroud
BL	Butt Line: - Lateral Distance from Aircraft Centerline
CG	Center of Gravity
c p	Specific Heat of Hot Duct Gases
c _{pa}	Specific Heat of Air at Constant Pressure
c _p	Specific Heat at Bulk Temperature
c _p	Specific Heat of Insulation at Constant Pressure
c _{pm}	Specific Heat of Metal at Constant Pressure
c _{po}	Specific Heat of Hydraulic Oil at Constant Pressure
CTOL	Conventional Take-off and Landing
d	Characteristic Duct Diameter
D	Duct Diameter or Shroud Diameter
D _H	Hydraulic Diameter of Section n
D _{Hn}	Hydraulic Diameter of Section n of Duct
$\mathbf{D}_{\mathbf{p}}$	Tailpipe Nozzle Exit Diameter
D _S	Tailpipe Shroud Exit Diameter
${^{\mathrm{D}}_{1}}$	Fan Diameter Designations
E	Hydraulic Oil Cooler Effectiveness Factor or a constant

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Symbol	Description
g	Acceleration of Gravity
G	Flow Rais per Unit Area, or a Constant
Gr	Grashof's Number
G.W.	Gross Weight
h	Convective Heat Transfer Coefficient, Height of Wing Fan Above Ground
h _a	Heat Transfer Coefficient Fuselage to Ambient
h _{ac}	Convective Component of h
h ar	Radiative Component of h
h _B	Convective Heat Transfer Coefficient Engine Bay Floor to Center Fuselage Air
h _e	Convective Heat Transfer Coefficient
h _{c3-4}	Convective Heat Transfer Coefficient Between Duct and Shroud
h _{c5-6}	Convective Heat Transfer Coefficient at Fuselage
h/D	Ratio of Lift Fan Height Above Ground Level to Fan Diameter
h _F	Convective Heat Transfer Coefficient Engine Bay Floor to Engine Bay Air
hg	Convective Heat Transfer Gases Pitch Fan Gases to Insulation
h _{gi}	Convective Heat Transfer Coefficient at the Insulation Surface
^h G	Convective Heat Transfer Rate Hot Gas to Tailpipe Wall

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Symbol	Description
h _o	Convective Heat Transfer Coefficient Outside Honeycomb Panel to Outside Air
h p	Convective Heat Transfer Coefficient; Inside Honeycomb Panel Surface to Air
h _r	Radiative Heat Transfer Coefficient
$h_{r_{3-4}}$	Radiative Heat Transfer Coefficient Between Duct and Shroud
h _{r5-6}	Radiative Heat Transfer Coefficient at Fuselage
h _{si}	Convective Heat Transfer Coefficient Shroud to Cooling Air
h _{s1}	Convective Heat Transfer Coefficient Shroud to Fuselage Air
h _T	Convective Heat Transfer Coefficient Shroud to Cooling Air
h _T 1	Convective Heat Transfer Coefficient Tailpipe to Cooling Air
h w	Convective Heat Transfer Coefficient Vertical Firewall to Air
$\mathbf{h_{w_i}}$	Convective Heat Transfer Coefficient Fuselage to Fuselage Air
$\mathbf{h_{w_1}}$	Convective Heat Transfer Coefficient Fuselage to Outside Air
h ₁₋₂	Heat Transfer Coefficient
$\mathbf{h_2}$	Heat Transfer Coefficient Between Fuselage Walls
h _{2c}	Convective Component of h
h _{2r}	Radi ative Component of h ₂
h ₃₋₄	Total Heat Transfer Coefficient Between Duct and Shroud

Symbol	Description
h ₅₋₆	Heat Transfer Coefficient
i	Ficticious Slab Interface
j	Time Increment
k	Ratio Specific Heat
К	General Pressure Loss Coefficient
k _b	Thermal Conductivity at Bulk Temperature
$k_{c_{3-4}}$	Thermal Conductivity of Air Between Duct and Shroud
K _G	Geometrical Pressure Loss Coefficient
K(G)	Geometrical Pressure Loss Coefficient
K _{Gn}	Geometrical Pressure Loss Coefficient
k _i	Thermal Conductivity of Insulation
K _n	Pressure Loss Coefficient for Section n of Duct
K n+1	Pressure Loss Coefficient for the Section of Duct Following Section n of the Duct
K _{n+2}	Pressure Loss Coefficient for the Second Section of Duct Following Section n of the Duct
K _P	Effective Thermal Conductivity of Honeycomb Panel
K _R	Pressure Loss Coefficient for the R th Section of a Duct
K _T	Total Value of Pressure Loss Coefficient
$\kappa_{\mathbf{T_n}}$	Total Value of Pressure Loss Coefficient at Cross-Section n of Duct
k ₂₋₃	Thermal Conductivity Power Distribution Ducting

Symbol	Description
k ₄₋₅	Thermal Conductivity Fiberglass Shroud
1	Length of Duct Under Consideration or Thickness of Honeycomb Panel
L	Length Used for Convective Heat Transfer Coefficient
L _n	Length of Section n of duct
M	Mach Number
M _A	$M_A = c_{p_2} - \gamma_i (\Delta X_i)^2 / k_i \Delta \theta$
Mach	Mach Number: - Ratio of Actual Speed to Speed of Sound
N _A	Defined by Equation $N_A = h_{g_i} \Delta X_i / k_i$
N _F	RPM or % RPM of Wing Fans
N _{FL}	RPM or % RPM of Left Wing Fan
N _{FR}	RPM or % RPM of Right Wing Fan
N _{Gr}	Grashof's Number
N _P	RPM or % RPM of Pitch Control Fan
P	Local Pressure
Pa	Ambient Pressure
Pamb	Ambient Pressure
P _i	Cooling Fan Inlet Pressure
Po	Ambient Air Temperature
P _p	Total Pressure of Primary Air in Ejector
Pr	Prandt'ls Number

Symbol	Description
PREF	Reference Pressure
P _S	Static Pressure at Flapper, Total Pressure Secondary Airflow in Ejector, Static Surface Pressure
P _{S1}	Static Pressure Boundary Layer Bleed Duct at Flapper
P _{S2}	Static Pressure Large Cooling Fan Duct at Flapper
P _{S3}	Static Pressure Following Mixing of Boundary Layer Bleed Air and Large Cooling Fan Air Downstream of Flapper
$\mathbf{P_{T}}$	Total Pressure
P _{T1}	Total Pressure Boundary Layer Bleed Duct at Flapper
P_{T2}	Total Pressure Large Cooling Fan Duct at Flapper
PTI	Inlet Total Pressure
PTI _m	Total Pressure at Inlet of Section m of Duct
PTIn	Inlet Total Pressure at Cross-Section n of Duct
PTI _{n+1}	Inlet Total Pressure at Cross-Section n+1 of Duct
PTO	Outlet Total Pressure
PTO _n	Outlet Total Pressure at Cross-Section n
PTO _{n+1}	Outlet Total Pressure at Cross-Section n+1 of Duct
P ₁	Absolute Pressure of Inlet Air to Blower
P_{2}	Absolute Pressure of Outlet Air From Blower
q	Dynamic Pressure, or Rate of Heat Flow
Q	Volume Rate of Air Flow
q AIR	Heat Transfer Rate to Hydraulic Oil Cooler Cooling Air

Symbol	Description
q _C B-W	Convective Heat Transfer Rate Fuselage Air to Fuselage
q _C _{F-A}	Convective Heat Transfer Rate Engine Bay Floor to Engine Bay Air
q _C F-B	Convective Heat Transfer Rate Engine Bay Floor to Center Fuselage Air
q _{CO}	Convective Heat Transfer Rate Outside Aircraft Surface
q _C _{P-A}	Heat Transfer Rate Honeycomb Panel to Engine Bay Air
q crew	Rate of Heat Addition Due to Heat Load From Crew
q _C S-A	Convective Heat Transfer Rate Shroud to Cooling
q _C s-B	Convective Heat Transfer Rate From Shroud to Fuselage Air
q _C	Heat Transfer Rate From Turbine Casing or Wall to Engine Bay Air
^q C _{T-8}	Convective Heat Transfer From Shroud to Cooling Air
q _C T'-8	Convective Heat Transfer From Tailpipe to Cooling Air
q _S W-A	Convective Heat Transfer Vertical Firewalls to Engine Bay Air
q _{g-a}	Net Convective Heat Transfer Hot Gases to Outside Air
q _{g-1}	Convective Heat Transfer Hot Gases to Insulation
$\mathbf{q}_{\mathbf{G}}$	Heat Addition Rate from Generator to Cooling Air; or Hot Gas Heat Transfer Rate to Tailpipe

Symbol	Description
$\mathbf{q_{G_i}}$	Total Energy Input to Generator
q _i	Rate of Heat Transfer from Hot Gases to Turbine or Duct Wall
$^{ m q}_{ m KP}$	Heat Transfer Rate Across Honeycomb Panel
$\mathbf{q}_{\mathbf{m}}$	Dynamic Pressure at Duct Station m
q _o	Free Stream Dynamic Pressure at Aircraft Speed
$\mathbf{q}_{\mathbf{oil}}$	Heat Transferred from Hydraulic Oil in Cooler
$\mathbf{q_{N'}} \mathbf{q_{N_1}}$, $\mathbf{q_{N_1}}$, $\mathbf{q_{N_2}}$	Wing Lift Fan Stream Dynamic Pressures
$\mathbf{q}_{\mathbf{NP}}$	Pitch Fan Stream Dynamic Pressure
q _{RB}	Radiative Heat Transfer Rate Engine Bay Floor to Center Fuselage
$\mathbf{q}_{\mathbf{RF}}$	Radiative Heat Transfer Turbine Casing or Duct to Engine Bay Floor
q _{RO}	Radiative Heat Transfer Outside Aircraft Surface to Environment
$^{ m q}_{ m RP}$	Radiative Heat Transfer Turbine Casing or Duct Wall to Honeycomb Panel
$\mathbf{q}_{\mathbf{R_{S-W}}}$	Radiative Heat Transfer Shroud to Fuselage
q_{RT}	Radiant Heat Transfer Rate Tailpipe to Shroud
$q_{R_{T-S}}$	Radiative Heat Transfer Tailpipe to Shroud
q _{RW}	Radiative Heat Transfer Rate Turbine Casing or Duct to Vertical Firewall
q _{RX}	Radiative Heat Transfer Rate Turbine Casing Axially Along Engine Bay

Symbol	Description
$\mathbf{q^s}$	Effective Fan Stream Dynamic Pressure
q s	Dynamic Pressure of Air Stream Running Along the Ground
q Solar	Rate of Heat Addition Due to Solar Heat Load
\mathbf{q}_{1}	Dynamic Pressure Boundary Layer Bleed Duct Air at Flapper
q ₁₋₂	Heat Transfer Rate Across Insulation
$\mathbf{q}_{2}^{}$	Dynamic Pressure Large Cooling Fan Duct Air at Flapper
$\mathbf{q}_{\mathbf{2-3}}^{}$	Heat Transfer Rate Across Fuselage Wall
^q 3-а	Heat Transfer Rate Fuselage Wall to Environment
r	Recovery Factor
R	Gas Constant
R_{e_d}	Reynolds Number for Flow Inside Ducting
Re	Reynolds Number for Flow Over Flat Plate
RE	Arithmatic Mean Reynolds Number
RPM	Revolution per Minute
S	Distance from Tailpipe Nozzle Plane to Shroud Exit Plane
STA	Aircraft Station
$egin{array}{c} \mathbf{s_1} \ \mathbf{s_2} \end{array}$	Distance Between Fans
t	Temperature
T	Absolute Temperature
^t a	Ambient or Outside Air Temperature

Symbol	Description
TA	Engine Bay Air Absolute Temperature
Tair in	Inlet Air Temperature to Hydraulic Oil Cooler
Tair out	Outlet Air Temperature from Hydraulic Oil Cooler
TAM	Mean Shroud Air Temperature
t AMB	Ambient Air Temperature
tAMB ₁	Ambient Air Temperature During Test
tAMB ₂	100° F
TAMB	Absolute Ambient Air Temperature
^t B	Temperature Boundary Layer Bleed Air
T _B	Absolute Temperature Engine Bay
t _c	Temperature Cockpit Air to Cooling Fan Plenum
TC	Thermocouple
TC _{MAX}	Thermocouple Number With the Maximum Reading
T _c s	Fan Stream Thrust Coefficient
T _D	Absolute Temperature Gas Power Distribution Ducting
t _F	Temperature Air from Large Blower at Flapper
$\mathbf{T}_{\mathbf{F}}$	Fuselage Air Temperature, Absolute Temperature Engine Bay Floor
tg	Pitch Fan Exhaust Gas Temperature
t g, j	Gas Temperature at Time Increment j
t g, j+1	Gas Temperature at j+1 th Time Increment

Symbol	Description
$^{\mathbf{t}}\mathbf{g}_{1}$	Wing Fan Exhaust Gas Temperature During Test
$\mathbf{t_{g}_{2}}$	Wing Fan Exhaust Gas Temperature at 100% Power
^t G	Generator Outlet Air Temperature
$\mathbf{T}_{\mathbf{g}}$	Absolute Total Temperature of Gas Stream in Duct
T _G	Absolute Temperature Duct Gases
t _H	Hot Gas Temperature from Wing or Diverter Valve Leakage
t _i	Temperature of Air to Engine Bay from Flapper
t _{i, j}	Temperature at i th interface at j th Time Increment
t _{i, j+1}	Temperature at i th Interface at j+1 th Time Increment
t _{i-1, j}	Temperature at i-1 th Interface at j th Time Increment
t i+1, j	Temperature at i+1 th Interface at j th Time Increment
^t L	Temperature of Duct Leakage
t _m	Mean or pitch fan Temperature of Cooling Fan Plenum Air or Wing Air
T _m	Absolute Mean Temperature of Cooling Fan Plenum Air
t _{MAX}	Maximum Landing Gear Environmental Temperature
t _{M1}	Measured Temperature of Landing Gear Environment During Test
t _{n, j}	Temperature at Insulation-Metal Plate Interface at j th Time Increment
t n, j+1	Temperature at Insulation-Metal Plate Interface at j+1 th Time Increment
t _o	Outside Air Temperature

Symbol	Description
T _o	Absolute Temperature Outside Air
Toil in	Temperature Inlet Oil to Hydraulic Oil Cooler
Toil out	Temperature Outlet Oil from Hydraulic Oil Cooler
t _{o, j}	Temperature Gas-Insulation Interface at j th Time Increment
t 0, j+1	Temperature Gas-Insulation Interface at j+1 th Time Increment
T	Wing Fan Lift at $\beta_v = 0$, $M = 0$, and $\beta_s = 0$
t _p	Temperature Inlet Air at Fuselage Port or Pitch Fan Inlet Air
T _p	Absolute Temperature of Primary Air of Ejector
T_{p_i}	Absolute Temperature Inside Surface Honeycomb Panel
t _{p, j+1}	Assumed Temperature Insulated Plate Temperature at j+1 th Time Increment
TPO	Absolute Temperature Outside Surface Honeycomb Panel
T _{REF}	Reference Temperature
T _s	Absolute Temperature of Secondary Air of Ejector, Fiberglass Shroud or Cooling Air Temperature
T _{SO}	Absolute Temperature of Outside Fiberglass Shroud
$\mathbf{T_{T}}$	Absolute Temperature Turbine Casing or Duct or Shroud Temperature
T _{T'}	Tailpipe Temperature
$^{\mathbf{T}}\mathbf{w}$	Absolute Temperature Vertical Firewall
T _X	Absolute Temperature Honeycomb Panel Aft of Turbine Casing

Symbol	Description
too	Hot Gas-Insulation Interface Temperature
t ₁	Temperature Gases to Fan Scrolls
T ₁	Absolute Temperature Insulation Surface
t ₂	Temperature Surface q
$\mathbf{T_2}$	Absolute Temperature Air Leaving Blower, or Absolute Temperature of Surface 2
t ₃	Temperature of Fuselage Surface
T ₃	Absolute Temperature of Fuselage Surface
t _{5.1}	X353-5B Gas Generator Exhaust Gas Temperature
t ₆	Temperature Fuselage Air
u _i	Overall Heat Transfer Coefficient Hot Gases to Turbine Case or Duct Surface
U _o	Overall Heat Transfer Coefficient Across Insulation and Fuselage
$\mathbf{v}_{_{\mathbf{D}}}$	Velocity of Boundary Layer Bleed Air
$\mathbf{v}_{\mathbf{F}}$	Wing Lift Fan Air Velocity
$\mathbf{v}_{\mathbf{g}}$	Pitch Fan Exhaust Gas Velocity Over Insulation
v_{m}	Mean Velocity in Duct
v _o	Aircraft Free Stream Velocity
v _p	Aircraft Flight Speed
w	Weight Rate of Airflow From Flapper to Engine Bay
Wa	Weight Rate of Airflow

Symbol	Description
$\mathbf{w}_{\mathbf{B}}$	Weight Rate of Boundary Layer Bleed Air to Flapper
w _c	Weight Rate Airslow Out of Cockpit
w _F	Weight Rate of Large Blower Cooling Air to Flapper or in Fuselage
$\mathbf{w}_{\mathbf{g}}$	Weight Rate of Hot Gas Flow
$\mathbf{w}_{\mathbf{G}}$	Weight Rate Airflow Through Generator
W _H	Weight Rate of Hot Gas Flow from Wing or Diverter Valve Leakage
$\mathbf{w}_{\mathbf{L}}$	Weight Rate of Duct Leakage
w _o	Weight Rate Oil Flow Through Hydraulic Oil Cooler or Outside Airflow
w _p	Weight Rate of Flow of Ejector Primary Air, Weight Rate of Air to Cooling Fan Plenum Through Fuselage Ports; or Weight Flow from Pitch Fan Area
w _s	Weight Rate of Flow of Ejector Secondary Air
$\mathbf{w_1}$	Weight Rate of Airflow in Boundary Layer Bleed Duct at Flapper
\mathbf{w}_2	Weight Rate of Airflow in Large Cooling Fan Duct at Flapper
\mathbf{w}_3	Weight Rate of Airflow Downstream of Flapper
WL	Aircraft Water Line
x	Distance
$\mathbf{x}_{\mathbf{i}}$	Thickness of Insulation
x _m	Thickness of Metal

Symbol	Description
Xs	Ground Distance From Lift Fan Center
$\mathbf{x_t}$	Correlating Temperature Difference Ratio
x ₂₋₃	Thickness Power Distribution Ducts
X ₄₋₅	Fiberglass Shroud Thickness
Y =	γ^2 g β C μ k For Air
α	Aircraft Angle of Attack
β	Volumetric Expansion Factor
β _{AP}	Apparent Turning Angle of Fan Turbine Exhaust
β _s	Stagger Angle of Lift Fan Louvers
$\boldsymbol{\beta}_{\mathbf{v}}$	Vector Angle of Lift Fan Louvers
β _{V₁}	Louver Vector Angle of Lift Fan No. 1
$^{eta}v_{2}^{}$	Louver Vector Angle of Lift Fan No. 2
γ	Density of Duct Gases
$\gamma_{\mathbf{i}}$	Density of Insulation
$\gamma_{\mathbf{m}}$	Density of Metal
Y REF	Reference Specific Weight
δ	Air Gap Thickness of Fuselage Section
Δ	Difference Symbol
$^{\delta}\mathbf{F}$	Flap Angle Setting
€	Surface Emissivity
° В	Surface Emissivity of Center Fuselage Bay
F F	Emissivity of Engine Bay Floor

Symbol	Description
ϵ pi	Inside Surface Emissivity Honeycomb Panel
€ po	Outside Surface Emissivity Honeycomb Panel
$\epsilon_{\mathbf{si}}$	Surface Emissivity of Inside of Shroud
$\epsilon_{ m sl}$	Surface Emissivity of Outside of Shroud
$\epsilon_{ m t}$	Surface Emissivity Turbine Casing or Duct
€ W	Surface Emissivity of Vertical Firewall
$\epsilon_{ m wi}$	Surface Emissivity of Inside Fuselage Skin
$\epsilon_{ m wl}$	Surface Emissivity of Outside of Fuselage Skin
ϵ_3	Emissivity of Power Distribution Ducting
ϵ_4	Emissivity of Inside Shroud Surface Gold Plated
€ 5	Emissivity of Outside Shroud Surface
€ 6	Emissivity of Fuselage Inside Surface
0	Angle of Boundary Layer Bleed Duct Flapper With Respect to Airflow; Time
μ	Viscosity of Duct Gases
μ _b	Viscosity of Air at Bulk Temperature
$^{\mu}$ wsi	Viscosity of Cooling Air at Inside Shroud Temperature
μ wt	Viscosity of Cooling Air at Outside Tailpipe Temperature
ρ	Density of Hot Gases in Ducts
σ	Stephan-Boltzman Constant 1730 x 10^{-12} as Used in This Report
Σ	Summation of Terms Symbols

Symbols	Description
Ф	Summation of Pressure Loss Coefficients Related to a Given Section of a Duct
ΔΡ	Pressure Difference
$^{\Delta P}_{ ext{DUCT}}$	Pressure Loss at Duct of Varying Shape, Cross-Section, etc.
$\Delta P_{ extbf{f}}$	Pressure Drop Due to Wall Friction
$^{\Delta P}_{G}$	Pressure Drop Due to Geometrical Factors
ΔP_n	Pressure Drop Across Section n of Duct
ΔP_{total}	Sum of ΔP_f and ΔP_G
$\Delta P_{\overline{T}}$	Incremental Total Pressure
$^{\Delta q}_{G}$	Rate of Heat Rejection by Generator
Δt	Temperature Rise Across Cooling Fan Blowers
ΔΤ	Temperature Difference Shroud to Fuselage Air
ΔT_{A}	Temperature Rise of Air Across Hydraulic Oil Cooler
Δt_{AH}	Incremental Temperature Due to Aerodynamic Heating
ΔT _{AH}	Incremental Absolute Temperature Due to Aerodynamic Heating
Δt _{FusAir}	Temperature Rise Due to Air Recirculation Between Shroud and Duct
$\Delta t_{f G}$	Temperature Rise of Generator Cooling Air
$^{\Delta t}_{ extbf{M}}$	Incremental Temperature of Landing Gear Environment
ΔT _{oil}	Temperature Change

Symbol	Description
$\Delta t_{\mathbf{p}}$	Temperature Drop Across Metal Plate
Δt_{sc}	Cockpit Air Temperature Rise Due to Solar and Crew Heat Loads
ΔT_{T}	Total Temperature Increment $\Delta t_T = \Delta t_{AH} + \Delta t_{SC}$
ΔX_{i}	Thickness of Slab of Figure 13.2 Defined as X _{i/n}
Δθ	Time Increment Defined from Equation for MA

9.3 COOLING SYSTEM ANALYSIS

9.3.1 Method of Approach

The cooling system analysis of this section establishes the balanced cooling air flow rates through the various flow passages of the aircraft. Thermal performance of the cooling system is considered in Section 9.4. The general procedure to establish the balanced flow rates consists of the following steps:

- 1. Definition of flow passages and their geometrical factors affecting flow rates; (See Tables 9. 1 through 9. 10 and Figures 9. 1 through 9. 9.)
- 2. Selection of pressure loss factors for the flow path components at the appropriate ranges of geometrical factors and estimated flow rates from Reference 12, (See Tables 9.1 9.10.)
- 3. Establishment of terminal conditions for each flow passage in terms of aircraft operation.
- 4. Calculation of pressure losses in each flow passage by a digital computer program for a matrix of input-output conditions of flow rate, inlet pressure and outlet pressure. This program is presented in Section 9.3.2.3.
- 5. Establishment of cooling fan performance at off-design conditions based on vendor and unpublished test data using conventional equations and procedures derived from fan similarity laws.
- 6. Generally balanced flow was established by a series of iterations in three steps: (a) the upper fuselage section was balanced assuming a series of lower fuselage compartment pressures; (b) the lower fuselage section was balanced based on the same series of compartment pressures used in 6(a); and (c) the upper and lower fuselage sections were balanced at the compartment pressure-flow rate interface.
- 7. The above steps were carried out for specific conditions of the aircraft speed-altitude envelope, operating mode, and

for ARDC standard and ANA Bulletin 421 hot day conditions. Although somewhat lengthy and tedious, once the procedure was established, balanced flow rates were established in a routine manner.

8. Since point by point coverage of the wide range of aircraft operating conditions was impractical, approximate methods were developed by analysis, which were verified by spot checks at terminal and mid-point conditions, and used to establish intermediate data by interpolation.

9.3.2 Pressure Loss Analysis

9.3.2.1 General

Since the airflow rate in a given duct is established only when the pressure drop available for flow is equal to the pressure drop required for the given flow, detailed knowledge of those factors affecting pressure loss estimates is needed. In the subject studies, compressible flow equations were used, unless otherwise specified. Pressure loss may be considered in two parts: frictional and geometrical components. The frictional component is expressed as

$$\Delta P_{f} = \left(\frac{4fl}{D}\right) q$$

The geometrical component (effective for changes in duct direction, shape, or cross-section) is given by:

$$\Delta P_G = K_G q$$

where ${\rm K}_G$ may be the product or sum of several factors depending upon the methods of data correlation. The total loss in pressure is the sum of these two components, or

$$\Delta P_{\text{Total}} = \Delta P_f + \Delta P_G = (4f^1/D + K_G) q$$

9.3.2.2 Incompressible Flow

An attractive advantage of incompressible flow is the ease with which ducting losses are analyzed and related in terms of one section of a duct passage. For example, in the equation

$$\Delta P_{\text{Duct}} = \Phi q_{\text{n}}$$

where

$$\Phi = K_n + K_{n+1} \left(\frac{A_m}{A_{m+1}}\right)^2 + K_{n+2} \left(\frac{A_m}{A_{m+2}}\right)^2 + \dots \cdot K_R \left(\frac{A_m}{A_R}\right)$$

both frictional and geometrical effects can be included in K without significant error and the study is referenced to any convenient cross-section n.

Incompressible flow was assumed for the following three branches - cockpit to cooling fan compartment, fuselage ports to cooling fan compartment, and the small cooling fan to the generator.

Flow from Cockpit to Cooling Fan Compartment

Standard Day, Sea Level

$$Q = 589 \sqrt{\Delta P_{in} H_2 O}$$
 $Ft^3/min.$

Hot Day, 2500 feet

$$Q = 633 \sqrt{\Delta P_{in} H_2 O} \qquad Ft^3/min.$$

The plots of Q vs $\triangle P$ for various day and altitudes are presented in Figures 9.10 and 9.11

Flow from Fuselage Ports to Cooling Fan Compartment

Standard Day, Sea Level

$$Q = 868 \sqrt{\Delta P_{in} H_2 O} \qquad \text{Ft}^3/\text{min.}$$

Hot Day, 2500 feet

$$Q = 934 \sqrt{\triangle P_{in} H_{2O}} \qquad Ft^{3}/min$$

The plots of Q vs ΔP for various days and altitudes are presented in Figures 9. 12 and 9. 13.

Flow from Small Cooling Fan to Generator

Standard Day, Sea Level

$$Q = 60.9 \sqrt{\Delta P_{in} H_2 O} \qquad Ft^3/r_{in}$$

Hot Day, 2500 feet

$$Q = 65.7 \sqrt{\Delta P_{in} H_2O} \qquad Ft^3/min$$

The plots of Q vs $\triangle P$ for various days and altitudes are presented in Figures 9.14 and 9.15.

9.3.2.3 Compressible Flow

All ducts except those mentioned in Section 9.3.2.2 were analyzed with compressible flow. The duct characteristics are presented in Figures 9.1 - 9.9 and Tables 9.1 - 9.9. Pressure loss analysis of the various ducts utilized an IBM 704 computer program requiring information on the following fluid and duct characteristics: temperature, viscosity, specific heat ratio, molecular weight, geometric K-factor, length, hydraulic diameter, duct station areas, and weight flow. An option was available to include a table of geometric K-factor vs Reynolds No. for any section where the Reynolds Number demonstrated a large effect.

The computer calculated and printed out the following:

1. Reynolds Number, RE

$$RE = \frac{12 D_{H} W}{\left(\frac{A_{m} + A_{m+1}}{2}\right) M}$$

2. Mach Number, M

M was obtained by iteration of the following equation

$$\frac{W\sqrt{T_g}}{A_m PTI_m} = Mg\left(\frac{k}{R}\right)^{1/2} \left[1 + \frac{k-1}{2}M^2\right]^{-\frac{k+1}{2(k-1)}}$$

3. Flow Velocity, V ~ Ft/sec.

$$V = M \left[\frac{1}{1 + \frac{k-1}{2} M^2} \right]^{1/2} (kRT_g)^{1/2}$$

4. Total K - factor, K_T

$$K_{T} = 4 \frac{L_{n}}{D_{H_{n}}} \left(0.0014 + \frac{0.125}{R_{E} \cdot 32} + K_{G_{n}} \right)$$

5. Pressure Ratio, PS/L'T

$$P_{S/P_{T}} = \left[1 + \frac{k-1}{2} M^{2}\right]^{-k/k-1}$$

6. Section Outlet Total Pressure, PTO ~ lb/in²

$$PTO_{n} = PTI_{n} \left[1-K_{T_{n}} \left(1-{^{P}S/_{P_{T}}} \right) \right]$$

7. Section Inlet Total Pressure, PTI ~ lb/in²

PTI_m is given

$$PTI_{n+1} = PTO_n$$
, $PTI_{n+2} = PTO_{n+1}$, etc.

8. Static Pressure, $P_8 \sim lb/in^2$

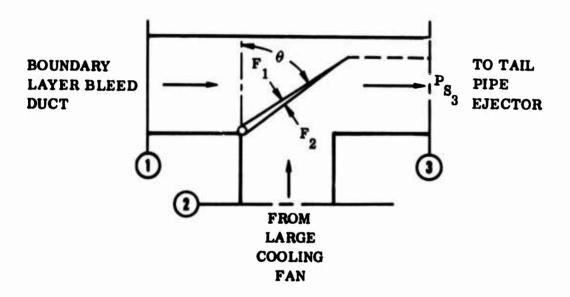
$$P_S = \left(\frac{P_S}{P_T}\right) PTO$$

9. Pressure Drop, $\triangle P \sim lb/in^2$

$$\Delta P_n = PTI_n - PTO_n$$

The results of the computer analysis for the various duct flows at various altitudes and days are presented in Figures 9. 16 through 9. 26. In some instances the effects of changes in pressure, temperature, and density with changes in altitude and type of day can be handled with simple ratios. Thus, some systems or branches may be calculated at standard day, sea level conditions and adjusted to various altitudes by the relationship between the pressure temperature, and/or density. Representative values of these ratios are presented in Table 9. 10.

9.3.3 Boundary Layer Bleed Duct Aft Flapper Position



A description of the aft flapper operation was presented in Section 3.0. Since the flapper is a free hinged door, the position is a function of the forces developed on each side by the two air flows from each duct branch. The system is analyzed as two ducts branching into a single duct by setting the flapper at any fixed position.

The method of approach is as follows:

1. Set the flapper position and treat the flapper as part of the ducts 1 and 2.

- 2. Using the duct characteristics of ducts 1 and 2 at each flapper setting, place into the computer program as described in Section 9. 2.
- 3. From the computer program, obtain the values at a given condition for the following: q_1 , q_2 , P_{S_1} , P_{S_2} , W_1 , W_2 , P_{T_1} and P_{T_2} (see sketch).
- 4. Balance the forces on the flapper at various flow rates

$$F = A_{FLAPPER} P_S + 2 A_{DUCT} q \sqrt{2(1-\cos\theta)}$$

where θ is the angle of flapper with respect to the flow

$$\frac{@ \theta = 0^{\circ}}{\frac{F_{1}}{17.6}} = P_{S_{1}} + 1.435q_{1} \qquad \frac{F_{2}}{17.6} = P_{S_{2}} + .395q_{2}$$

$$@ \theta = 22.5^{\circ}$$

$$\frac{F_{1}}{17.6} = P_{S_{1}} + 2.52q_{1} \qquad \frac{F_{2}}{17.6} = P_{S_{2}} + .395q_{2}$$

$$\underline{@ \theta = 45^{\circ}}$$

$$\frac{F_{1}}{17.6} = P_{S_{1}} + 1.74q_{1} \qquad \frac{F_{2}}{17.6} = P_{S_{2}} + .773q_{2}$$

$$\underline{@ \theta = 67.5^{\circ}}$$

$$\frac{F_{1}}{17.6} = P_{S_{1}} + .886q_{1} \qquad \frac{F_{2}}{17.6} = P_{S_{2}} + 1.125q_{2}$$

- 5. With P_{S_3} , W_3 and Mach number, compare with duct system ΔP to the tailpipe exhaust and the tailpipe ejector performance at the given Mach number.
- 6. When the flapper system flow balances with the tailpipe ejector performance, the flapper is balanced at that Mach number.

Figure 9.27 shows the estimated flapper position at various Mach numbers. Its position is not significant beyond the fact that the division of flow is a unique function of flapper position, which in turn is a function of Mach number and power setting conventional flight mode.

9.3.4 Tailpipe Ejector Analysis

The tailpipe ejector augments cooling airflow in the engine bay, tailpipe, and shroud during turbojet mode operation. The ejector is a simple, conical extension of the shroud past the tailpipe (Figure 3. 6), with the following design characteristics:

$$\frac{D_s}{D_p} = \frac{Shroud Exit Diameter}{Tailpipe Exit Diameter} = 1.10$$

$$\frac{S}{D_p} = \frac{Shroud\ Extension}{Tailpipe\ Exit\ Diameter} = 0.40$$

A full scale ejector with nearly identical design characteristics was experimentally tested and recorded in Reference 13 showing the rela-

tionship between
$$P_p/P_o$$
, P_s/P_o , and $\frac{W_s\sqrt{T_s}}{W_p\sqrt{T_p}}$, see Figures 9.28 and 9.29.

The values of W_p , $\sqrt{T_p}$, and P_{p/P_o} are known for any altitude, day and engine setting, therefore a plot between P_{s/P_o} and W_s can be made at various temperatures. A plot can also be made of P_{s/P_o} vs W_s for ducting system forward of the ejector. When the values of P_{s/P_o} and W_s for the ejector equal the values of P_{s/P_o} and W_s , respectively, for the ducting, then the ejector is in balance. This cross plot of balance flow is presented in Figures 9.30 through 9.42.

9.3.5 Cooling Air Flow Between the Nose Fan and Wing Fan Cavities During Conventional Flight

During CTOL flight mode, the doors are closed at the wing and nose fan, but air gaps exist around the doors. In flight relatively high positive pressures develop at the nose fan doors; and relatively low negative pressures develop at the wing fan (see Figures 9.43 and 9.44).

Cooling air will flow into the nose fan cavity, through the hot gas supply ducts, into the wing fan cavity, and then out the wing fan closures to the outside. See Figures 9.45 and 9.46 for cavity pressure vs flow in or out, and Figure 9.47 for a flow rate vs Δp between the cavities.

9.3.6 Cooling Fan Outlet Total Pressure

The outlet total pressure of the small and large cooling fan is a function of the chamber pressure, fan speed, air density, flow rate and static pressure rise across the fans. The large fan outlet total pressure vs Q and P_i is presented in Figures 9.48 through 9.55. The small fan outlet total pressure vs flow rate and inlet pressure is presented in Figures 9.56 through 9.63.

9.3.7 Cooling Air Weight Flow

The cooling air weight flow from the upper fuselage to the lower fuselage and from the lower fuselage to the outside is a function of the fuselage pressure. The weight flow through each branch as a function of fuselage pressure in the lift fan mode is presented in Figures 9.64 through 9.74, and in the conventional mode is presented in Figures 9.75 through 9.103. A balanced total flow into the center and lower forward fuselage with the balanced total flow out will give a balanced system through each branch such as presented in Figures 9.74, 9.83, 9.102, and 9.103.

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TABLE 9.1
COOLING AIR DUCT DEFINITION - BOUNDARY LAYER
BLEED DUCT TO ENGINE BAY
(See Figure 9.1)

STATION	ARSA	SECTION	SECTION NO.	K(G)	L W.	DH IN.
0	10.8					
1	12.3	STRAISHT - KECTANGULAR	1	0.10	2.44	2.3
2	14.5	Exference (1)	Z	0.02	0.	2,3
3	14.8	STRAIGHT. RESTANSOLER	3	0.	1.66	3.1
4	9.1	Contractor	4	0.03	5.40	3. Z
5	15.7	FXCAUSION	5	0.05	10.13	4.3
6	19.2	CORVING- RECTANGULAR	6	0.14	7.13	5.7
7	19.2	SPLITTER	7	0.32	0.	5.7
8	19.3	CURVINZ RECONSCIENT	8	0.08	2.20	3.9
9	18.6	PERTANGUER	9	0.	4.20	3.9
10	18.Z	CORVING	10	0.04	4.00	4.0
//	20.4	CUESTING	//	0.09	4.30	4.4
12	20.4	THEILUT.	12	0.01	8.00	4.9
/3	20.4	BELLOWS	13	0.04	3.15	4.9
14	20.9	STRAIGHT	14	0.	16.10	4.9
15	399.0	EXPONSIO	15	0,90	0.	4.9

TABLE 9.2 COOLING AIR DUCT DEFINITION - LARGE COOLING FAN TO BOUNDARY LAYER BLEED DUCT (See Figure 9.2)

STATION)	APEA	SENTION	SECTIONS	K(G)	L	D ₄
0	8.15					
,	8,15	CURVING - RECTANGULAR	,	0.91	3.9	2.74
2	8.15	CURVING RECTANGULAR	Z	0.01	0.4	2.74
3	8.15	STANDAT -	3	0.	2.3	2.74
4	8,15	CURVING - RECTANGULAR	4	0.09	12.3	2.79
5	7.23	EXPANSION - RECTANGULAR	5	0.01	4.2	2.88
6	8,92	CURVING - RECTANGULAR	6	0.02	3.6	2.96
7	8.92	CURVING - RECTANGULAR	7	0,13	Z.5	2.96
8	8.92	STRAIGHT RECTANGULAR	8	0.	4.0	2.96
9	8,92	RECTINGULAR 90°-BEND	9	1.87	5.2	3.36

TABLE 9.3
COOLING AIR DUCT DEFINITION - ENGINE BAY TO TAIL PIPE EJECTOR
(See Figure 9.3)

NO.	AREA IN2	SECTIONS SHAPE	SECTION NO.	K(G)	L N.	DH IN.
15	399.0					
16	284.5	ANNULUS	16	0.	27.0	9.1
17	379.0	ANNULUS	17	0.	26.9	8.9
18	394,0	Apmot VC	18	0.	3,0	10.2
19	92.6	CONTRACTION	19	0.//	0.	3.0
20	92.6	CURVING- ANDULUS	20	0.05	13.5	3.2
21	84.8	CONTRACTION	21	0.02	0.	3.2
22	84.8	ANNULUS	22	0.	85.6	3.2
23	84.8	CURVING ANDLUS	23	0,06	17.0	3.2
24	92.6	EXPONSION -	24	0.	5.6	3.4
25	92.6	EXPANSION	25	1.0	0.	3.4

TABLE 9.4
COOLING AIR DUCT DEFINITION - SMALL COOLING FAN TO
ELECTRONIC COMPARTMENT
(See Figure 9.4)

	1		1			
Chritate Ou,	DREA	SECTION SHAPE	SECTION) NO	K(G)	بدر	DH IN
σ	11.2	• •		-10 FERREL WAR NO		
1	28.4	CURVING FERTANGULAR	/	0.16	5.1	5.8
2	28,4	HYDRAULIZ	2	18.6	0.	5.8
3	16.0	DIFFUSE &	3	0.04	5.1	4.8
4	16.0	STRAIGHT. CIRCULAR	4	0.	4.5	4.5
5	16.0	CURVING - CIRCULAR	5	0.31	21.1	4.5
6	16.0	STERIOUT - GROVER	6	0.	1.0	9.5
7	438.0	EXPANSION	7	0.95	0.	4.5
8	438.0	Straight - Feotenburg	8	0.	14.5	23.9
9	203.0	Courters (1)	9	0.34	0.	23.9
10	660.0	EXPAILSION	10	0.48	0.	23.9
11	660.0	STRAIGHT . RECTANGULAR	11	0.	18.5	32.5
12	28.3	CONTERNO VI	12	2.88	0.	32.5

TABLE 9.5
COOLING AIR DUCT DEFINITION - L.H. LARGE COOLING FAN
TO CENTER FUSELAGE
(See Figure 9.5)

SMITHAL) LO	AFEA III	35000000 54975	200 27 Jr. 10 12 2	K%)	L IN.	DH IN.
٥	9.7					
,	9.7	STRAIGHT-	,	0.	1.75	3.02
2	9.7	CURYING . PECTROLOGIC	Z	0.09	1.70	3.02
3	9.7	STRAG HT- RECTAINGUING	3	0.	.85	3.02
4	9.7	ANGLE	4	0.08	0.	3.02
5	15.0	EXTENDING RECTANGLE	5	0.12	12.0	241
6	15.0	EXPAUSION	6	1.0	0.	9.41

TABLE 9.6 COOLING AIR DUCT DEFINITION - R.H. LARGE COOLING FAN TO CENTER FUSELAGE (See Figure 9.6)

STATION NO	AREA IN ^Z	SECTION S NAPE	Sec tion No	k(G)	L	DH IN
0	9.7					
1	9.7	Straight- Rectanguar	,	0.	1.75	3.02
Z	9.7	Curving - Restangular	Z	0.09	1.70	3,02
3	9.7	STRAIGHT - RECTANGULAR	3	0.	.85	3.02
4	9.7	ANGLE	4	0.06	0.	3,02
5	14.6	EXPANSION	5	0.11	9.8	8,01
6	12.Z	STRAIGHT- REETANGULAS	6	0.	2.0	12.20
7	12.2	ANGLE	7	0.08	0.	12.20
&	15.9	EXPANSION	8	0.05	11.4	8.84
9	15.9	STRPIGHT - ELLIPSE	9	0.	22.8	4.25
10	15.9	ANGLE -	10	0.02	0.	4.25
1	15,9	STRAIGHT - ELLIPS E	//	0.	16.7	4,25
12	15.9	EXPANSION	12	1.0	Ö.	4.25

TABLE 9.7
COOLING AIR DUCT DEFINITION - ELECTRONIC COMPARTMENT
TO PITCH FAN AIR EJECTOR
(See Figure 9.7)

4				, ,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	,	T .
STATI CA	PREA	SECTION	SECTION	K(G)		\mathcal{D}_{*}
No.	INZ	SHAPE	Ν̈́O		1 N	. / <i>M</i> .
0	253.0	=				
1_	55.0	CONTRACTIONS	/	0.46	0.	
2	253.0	EXFRUSION	2	0.63	0.	
3	253.0	STRAIGHT	3	0.	9.3	10.5
4	96.0	CONTRACTO	4.	0.39	0,	
5	253.0	EXPANSION	5	0.39	0.	
6	253.0	57 Karang	6.	0,	11.0	10.5
7	96D	CONTRACTION	7	0.39	<i>O</i> .	
8	253,0	EYPHIDSION.	٤	0,39	0.	-
9	2.53,0	STRAIGHT	9	0,	22.0	10.5
10	76.0	contensity!	10	0.39	0,	
//	253.0	EXPRISION	//	0.34	0.	_
12	253.0	ミファイノイリア	12	0.	9.5	10.5
13	97.0	CONTRACTICAL	13	0.38	0.	
14.	414.0	EXPRISION	14	0.59	0,	
15	414.0	Della Francisco	15	0.37	3Z.1	13.3
16	7.0	90°-BSND Recornson	16	0.01	0.	_
17	4.7	STENIONT- STEENING	17	0.	2.0	1.1
18	4.3	POT ESTIVE STREAMS WE	15	8.40	2.6	1.1
14	4.3	EYPANS".	19	1.0	0,	

TABLE 9.8 COOLING AIR DUCT DEFINITION - CENTER FUSELAGE TO FLAP ACTUATOR COMPARTMENT (See Figure 9.8)

STATION NO	AREA IN ²	SECTION) SHAPE	SECTION NG	K(G)	L IN.	DH IN.
0	19.0					
,	14.0	CONTRACTION	B	0,48	0.	_
Z	480.0	EXPANSION	2	1.0	0.	-
3	48.0	STRAIBHT- RECTANGIVAL	3	0.	10.5	16.1
4	9.0	CONTRACTICN	4	0.48	0.	
5	480.0	EXPANSION	5	1.0	0.	_
6	48.0	STRAIGHT - RECTANGULAR	6	a	11.0	16.1
7	16.0	CONTRACTION	7	0.48	0.	Constitution of the Consti
8	16.0	מוניציתמן דא	8	1.0	0.	_

TABLE 9.9
COOLING AIR DUCT DEFINITION - CENTER FUSELAGE
TO WING FAN AIR EJECTORS
(See Figure 9.9)

(See Figure 3.3)						
STATION NO	APSA INZ	SECTION SHAPE	SECTION NO	K(G)	<u> </u>	Dw IN.
0	86.6					
,	86.6	CHITERSTIND	,	0.43	0.	_
z	234.0	EXPONSION	2	0.40	0.	_
3	159.6	STEPLENT	3	0.01	4.4	16.6
4	85,0	CONTRACTION	4	0.30	0.	-
5	157.0	EYPAUSION	5	021	0.	_
6	123.0	STATIST	6	0.01	3.2	10.6
7	71.0	C 2 1 1 7 K 7 2 2 1 1 1 1 1	9	0,27	0.	_
8	123.0	TATIONES	8	0.17	0.	_
9	91.0	SWALLET	9	0.01	3.4	9.4
10	50.0	Contraction	10	0.29	0.	_
11	91.0	EXPADSIBAL	11	0.20	0.	-
12	75.0	STRAIGHT	12	0.01	4.3	7.8
13	44.0	CONTRACTION	/3	0.27	0.	_
14	75.0	Expression	1.1	0.16	0.	_
15	46.0	STATISHT	15	0.01	7.4	6.Z
16	26.0	CONTINGTON	16	0.28	0.	
17	46.0	Expression	17	0.18	0.	
18	35.0	STRAIGHT	/8	0.01	7.3	5.3
19	18.0	CONTRACTION	19	0.32	0.	_
20	35.0	Exemples	20	0.23	0.	_
21	24.0	STEARSHT	21	0.01	4.0	4.5
22	10.1	COUTRACTION	22	0.37	0.	-
23	5.0	STRAIGHT	23	0.32	0.	_
24	3.2	FLOW BEND	24	1.43	5.0	1.8
25	8.0	STRAIGHT	25	0.63	6.0	2.2
26	4.Z	90 BEND	26	0.60	6.0	7.5
27	4.2	EXPANSION	27	1.00	0.	-

TABLE 9. 10
ARDC STANDARD DAY AND ANA BULLETIN 421 HOT DAY ALTITUDE CONDITIONS REFERENCED TO ARDC STANDARD DAY SEA LEVEL

ALTITUDE -FEET	PREF	VTRE#	P PREF T	8 8REF
STANDARD DAY SEA LEVEL 5,000 10,000 20,000 30,000 40,000	1.0 .8321 .6878 .4579 .2975	1.0 1.0176 1.0362 1.0766 1.1222 1.1532	1.0 .8467 .7127 .4571 .3338 .2143	1.0 .8612 .7379 .5321 .3736 .2461
HOT DAY SEA LEVEL 2,500 5,000 10,000 20,000 30,000 40,000	1.0 .9/93 .8439 .7081 .4890 .3279 .2125	.9600 .9680 .9763 .9933 1.0302 1.0715 1.1216	. 9600 . 8819 . 8239 . 7033 . 5038 . 3513 . 2383	.9229 .8628 .8057 .7002 .5195 .3774 .2662

REFERENCE : STANDARD DAY SEA LEVEL

PRESSURE $P = 2/16.2 \#/FT^2$ TEMPERATURE T = 5/8.69 %SPECIFIC WEIGHT $Y = .07647 \#/FT^3$

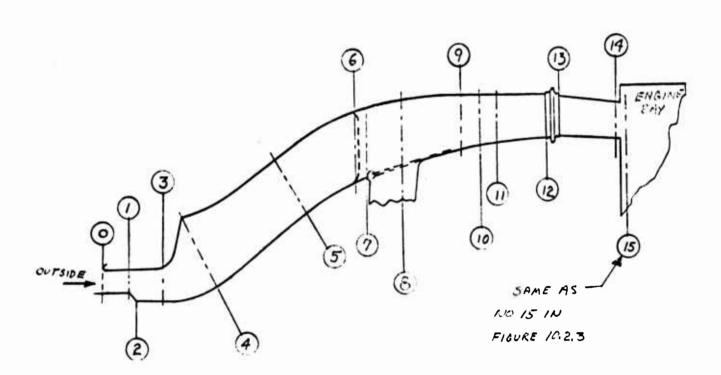


Figure 9.1 Cooling Air Duct Definition - Boundary Layer Bleed Duct to Engine Bay

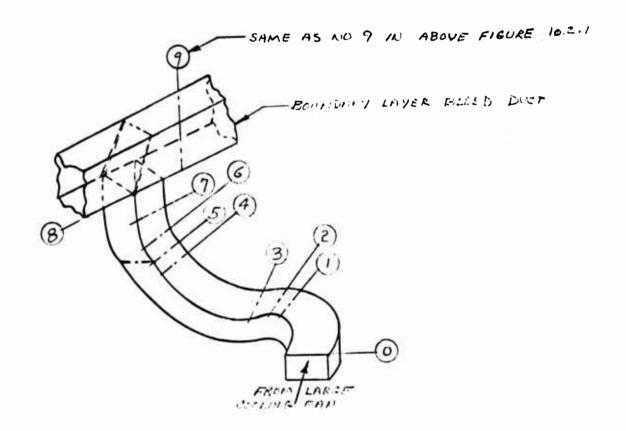


Figure 9.2 Cooling Air Duct Definition - Large Cooling
Fan to Boundary Layer Bleed Duct

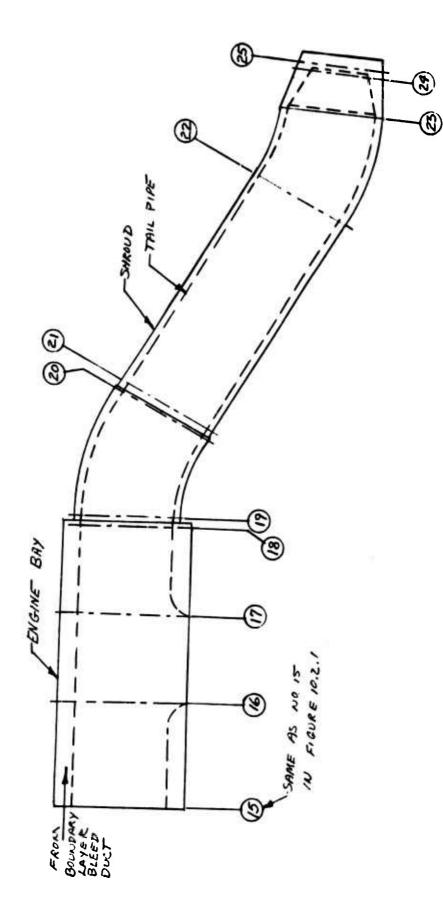


Figure 9.3 Cooling Air Duct Definition - Engine Bay to Tailpipe Ejector

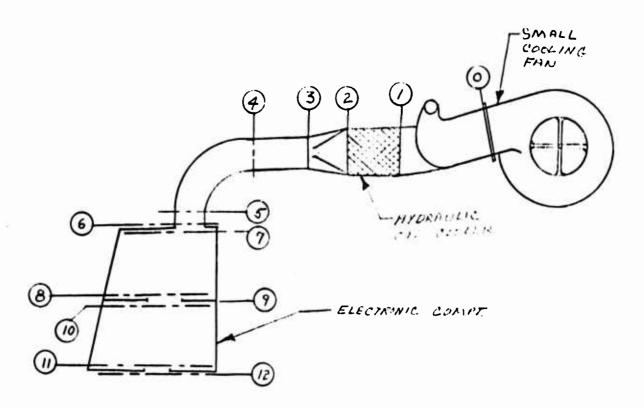


Figure 9.4 Cooling Air Duct Definition - Small Cooling
Fan to Electronic Compartment

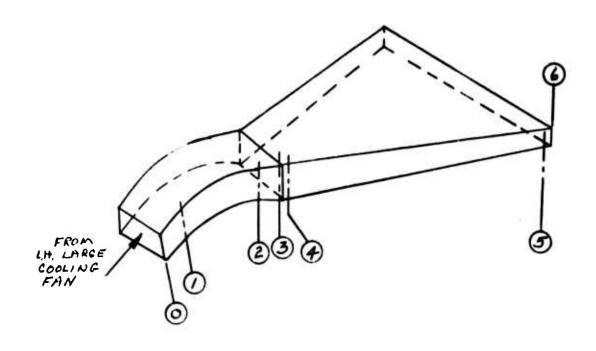


Figure 9.5 Cooling Air Duct Definition - L. H. Large Cooling Fan to Center Fuselage

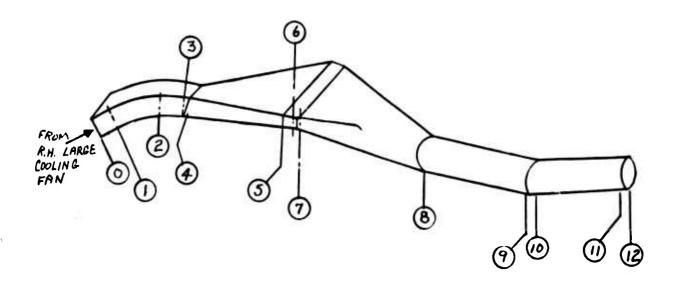


Figure 9.6 Cooling Air Duct Definition - R.H. Large Cooling Fan to Center Fuselage

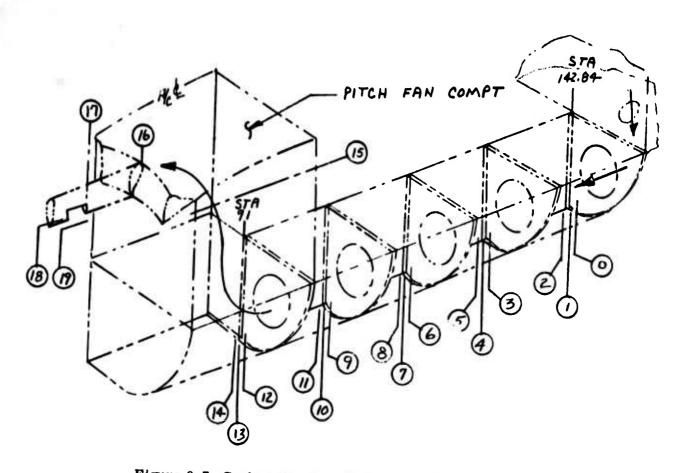


Figure 9.7 Cooling Air Duct Definition - Electronic Compartment to Nose Fan Air Ejectors

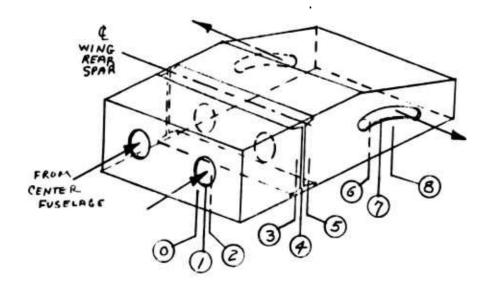


Figure 9.8 Cooling Air Duct Definition - Center Fuselage to Flap Actuator Compartment

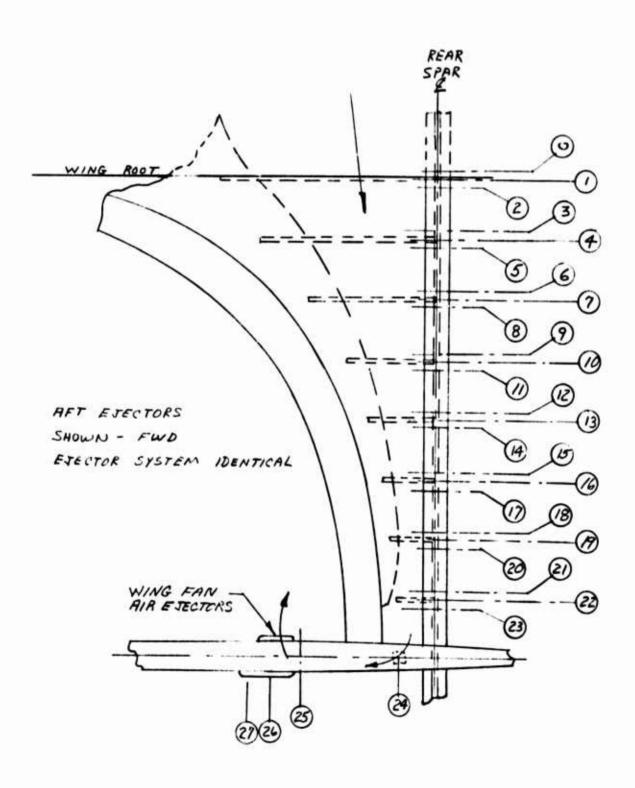


Figure 9.9 Cooling Air Duct Definition - Center Fuselage to Wing Fan Air Ejectors

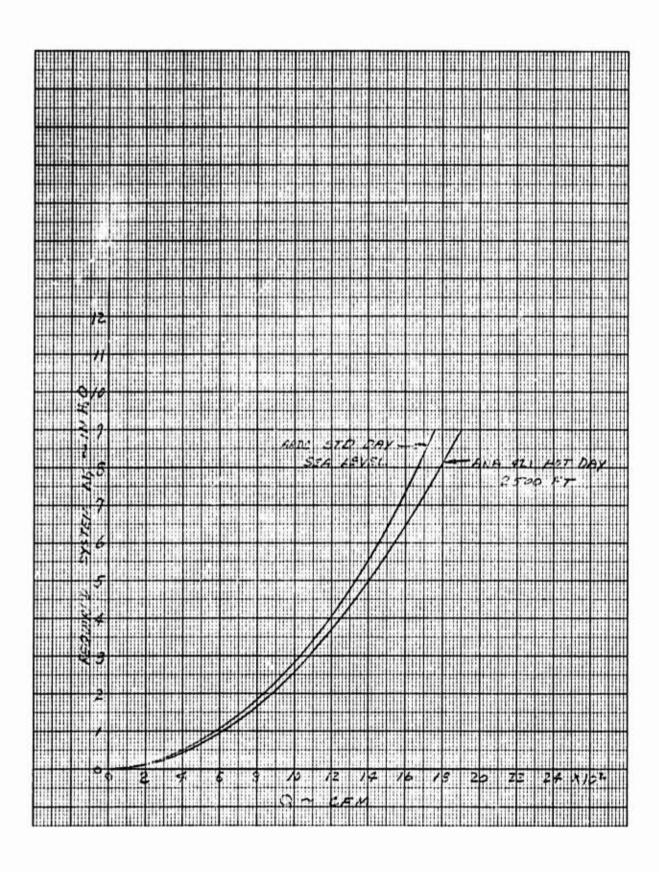


Figure 9. 10 Duct Pressure Loss - Cockpit to Cooling Fan Compartment Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

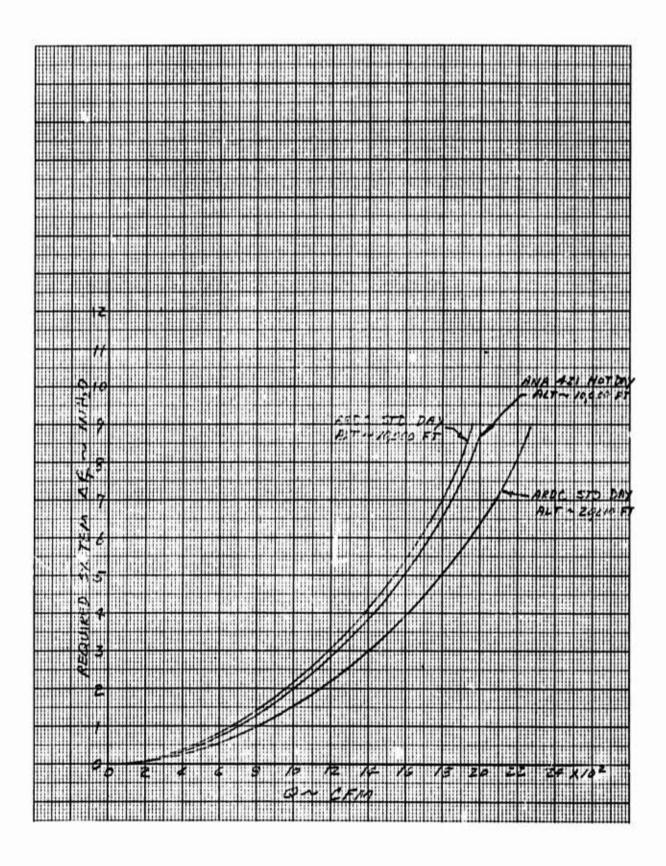


Figure 9. 11 Duct Pressure Loss - Cockpit to Cooling Fan
Compartment Vs Cooling Air Flow - Standard
Day 10,000 and 20,000 Ft., and Hot Day 10,000 Ft.

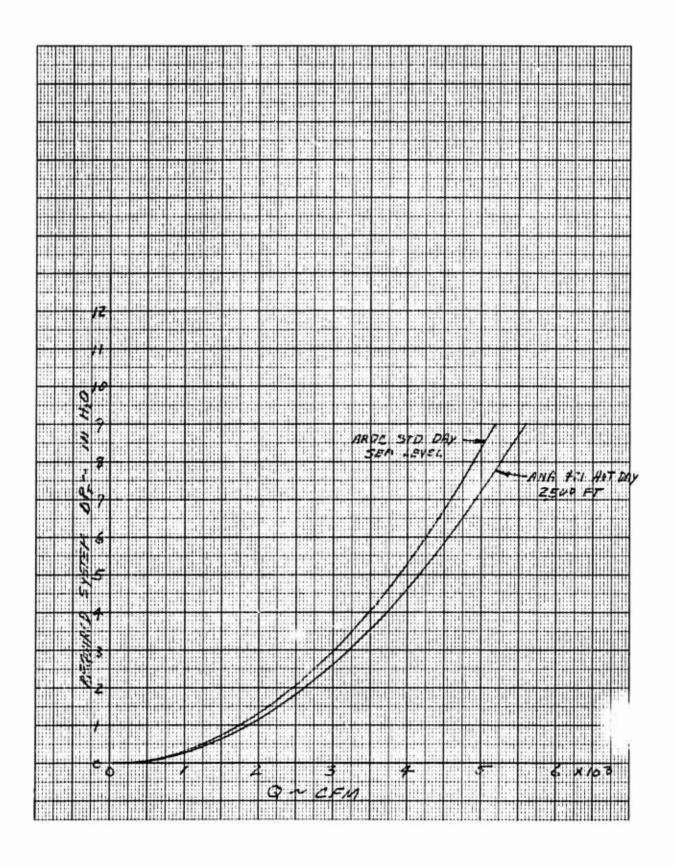


Figure 9.12 Duct Pressure Loss - Fuselage Ports to Cooling Fan Compartment Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

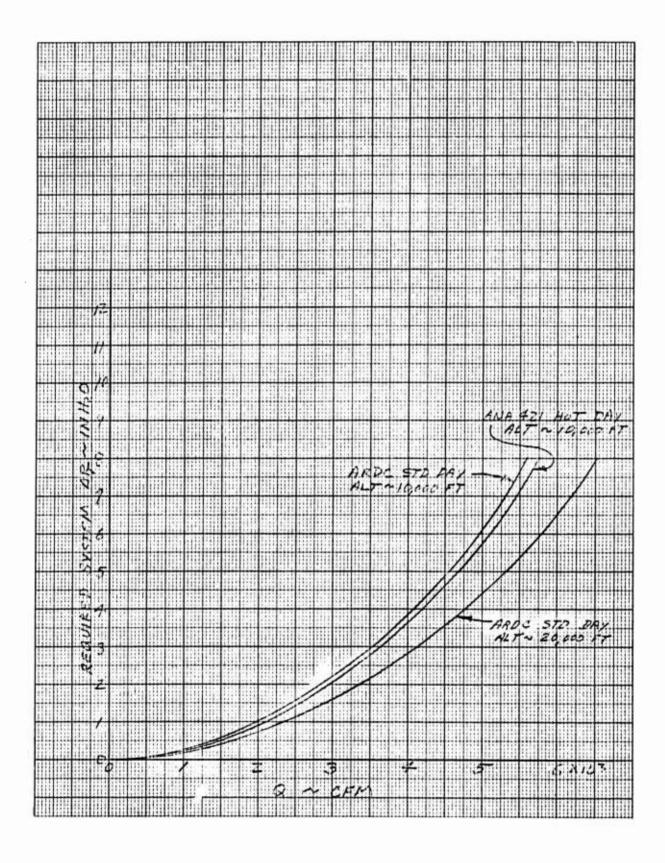


Figure 9. 13 Duct Pressure Loss - Fuselage Ports to Cooling Fan Compartment Vs Cooling Air Flow - Standard Day 10,000 and 20,000 Ft, and Hot Day 10,000 Ft.

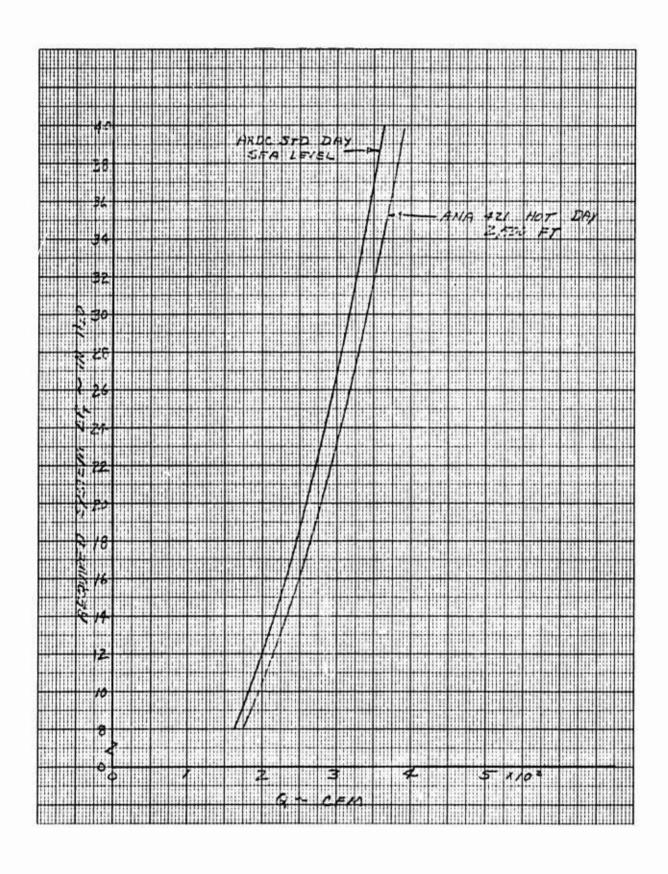


Figure 9.14 Duct Pressure Loss - Small Cooling Fan to Generator Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

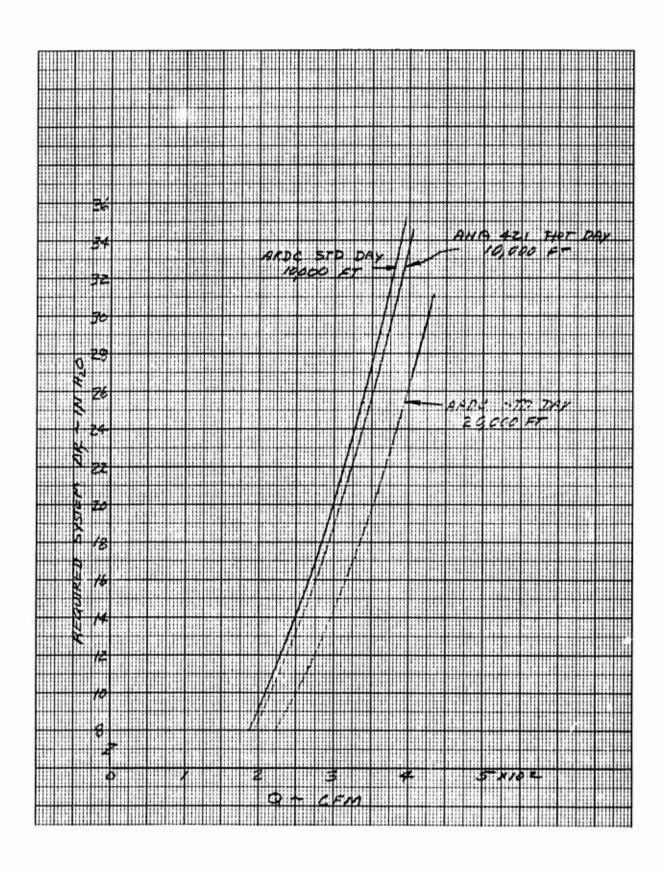


Figure 9.15 Duct Pressure Loss - Small Cooling Fan to Generator Vs Cooling Air Flow - Standard Day 10,000 and 20,000 Ft., and Hot Day 10,000 Ft.

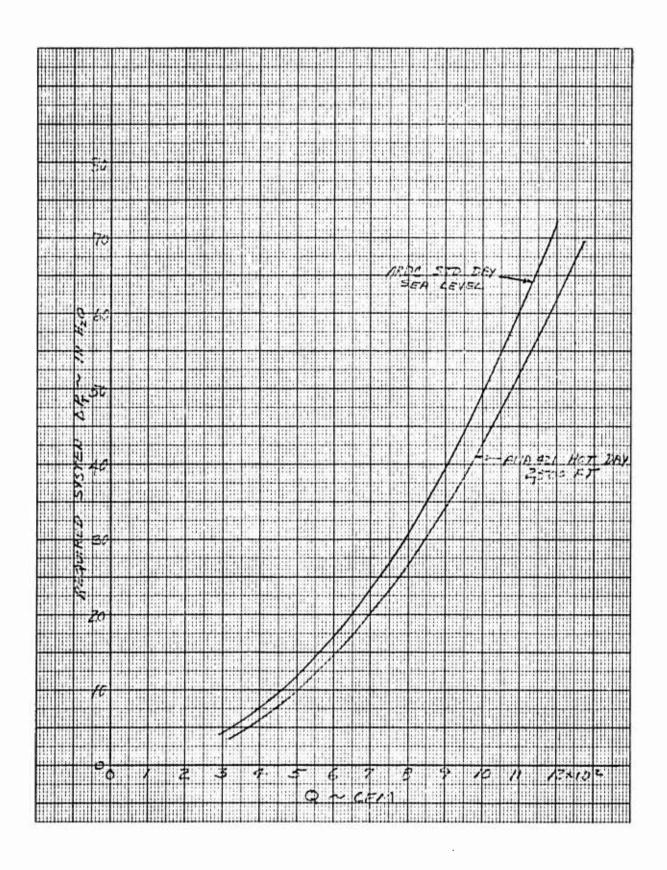


Figure 9. 16 Duct Pressure Loss - Small Cooling Fan to Electronic Compartment Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

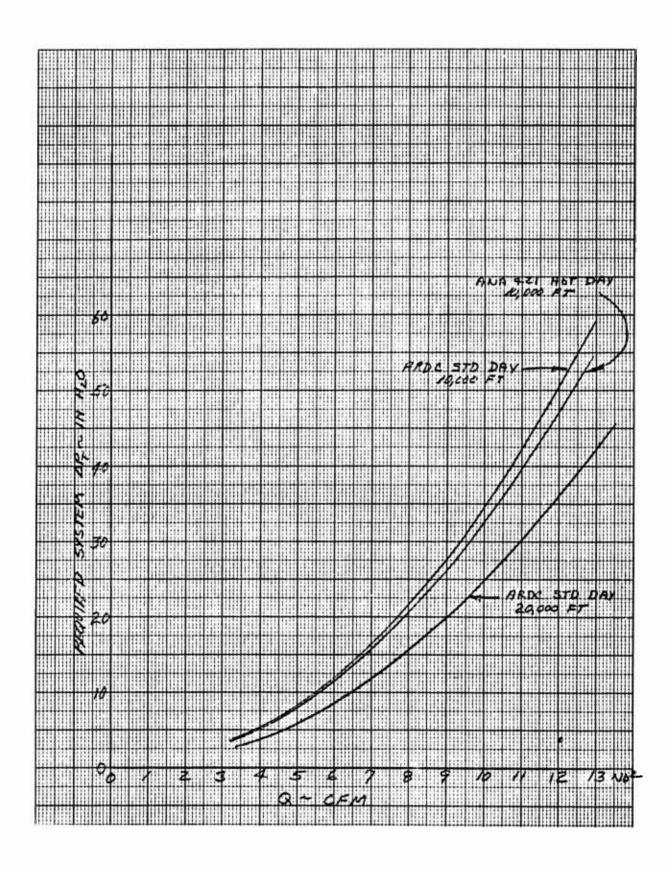


Figure 9.17 Duct Pressure Loss - Small Cooling Fan to Electronic Compartment Vs Cooling Air Flow - Standard Day 10,000 and 20,000 Ft., and Hot Day 10,000 Ft.

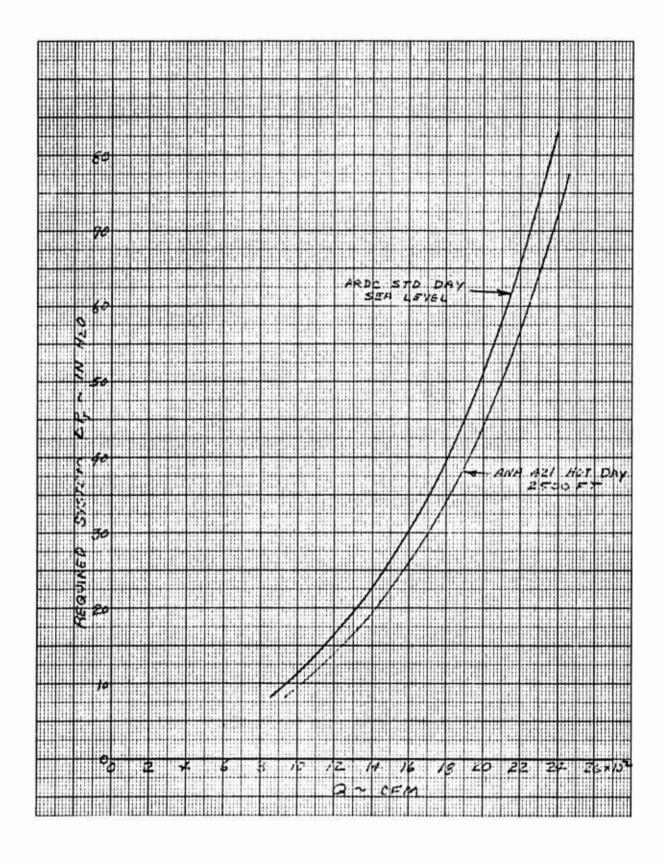


Figure 9. 18 Duct Pressure Loss - L. H. Large Cooling Fan to Center Fuselage Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

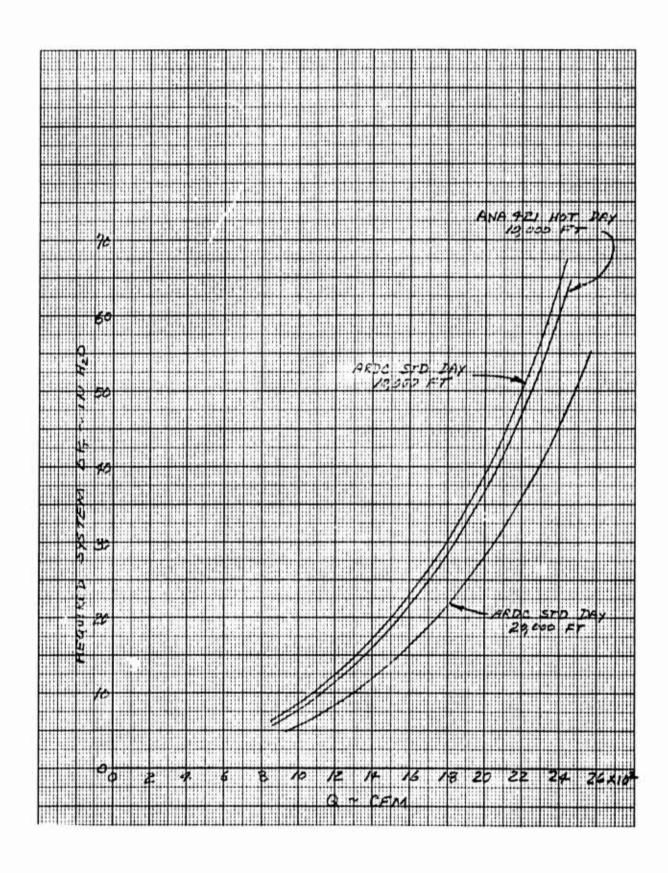


Figure 9. 19 Duct Pressure Loss - L. H. Large Cooling Fan to Center Fuselage Vs Cooling Air Flow - Standard Day 10,000 and 20,000 Ft., and Hot Day 10,000 Ft.

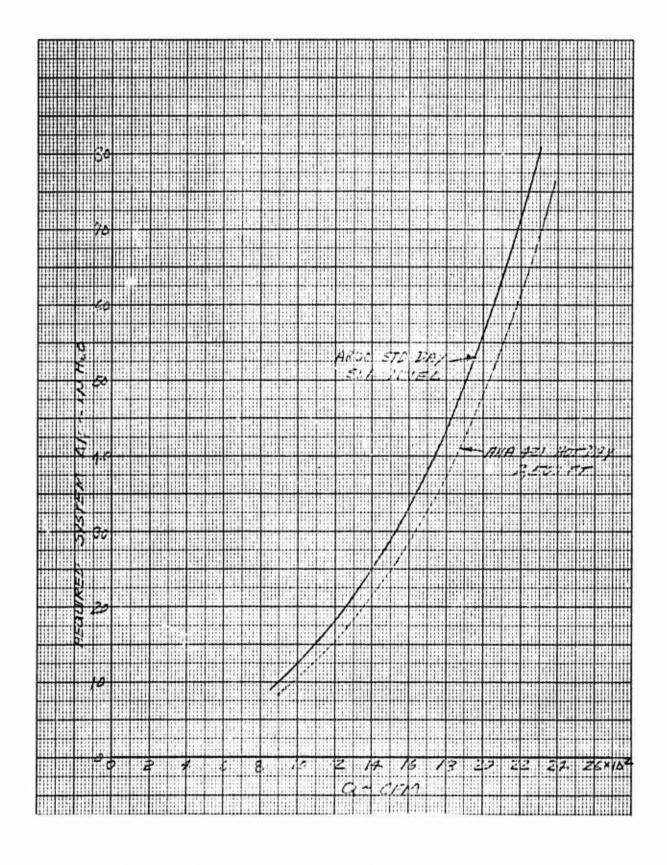


Figure 9. 20 Duct Pressure Loss - R.H. Large Cooling Fan to Center Fuselage Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

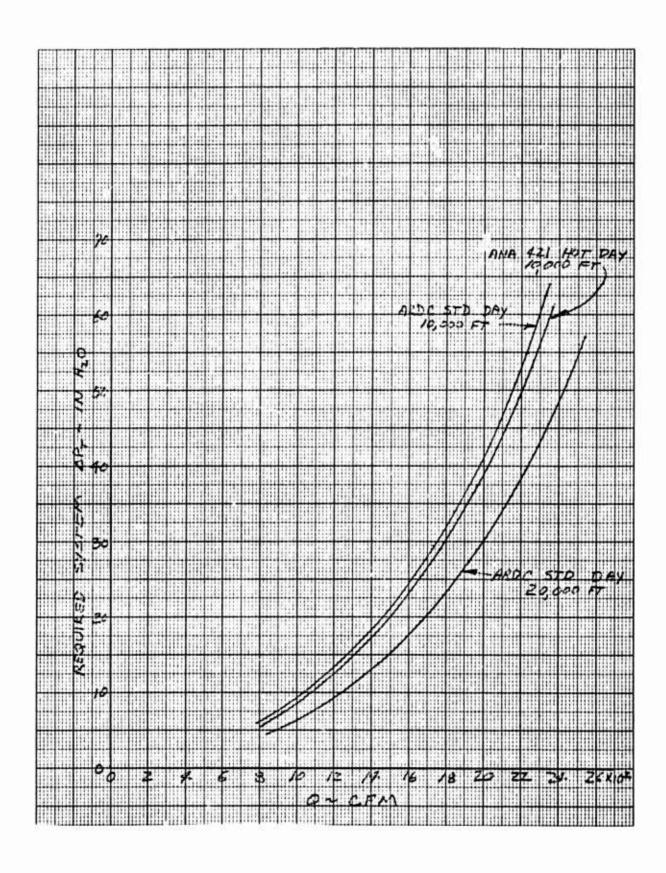


Figure 9. 21 Duct Pressure Loss - R.H. Large Cooling Fan to Center Fuselage Vs Cooling Air Flow - Standard Day 10,000 and 20,000 Ft., and Hot Day 10,000 Ft.

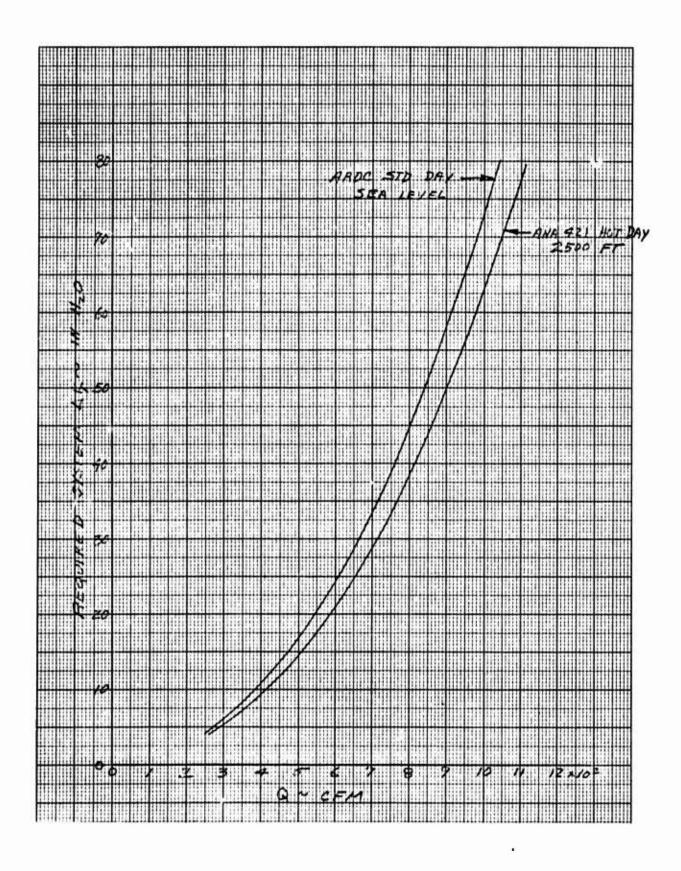


Figure 9. 22 Duct Pressure Loss - Large Cooling Fan to Tailpipe Ejector Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

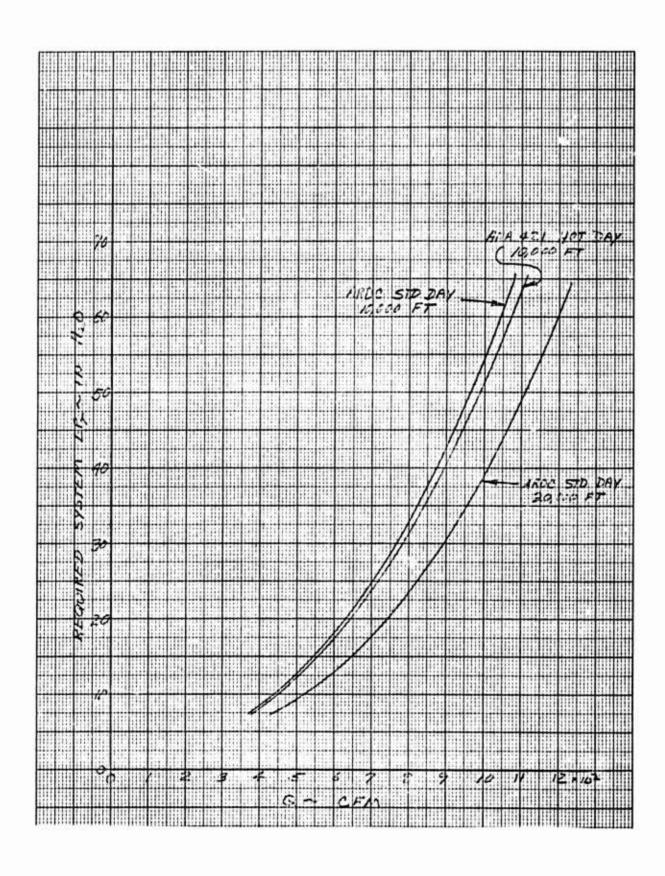


Figure 9.23 Duct Pressure Loss - Large Cooling Fan to Tailpipe Ejector Vs Cooling Air Flow - Standard Day 10,000 and 20,000 Ft., and Hot Day 10,000 Ft.

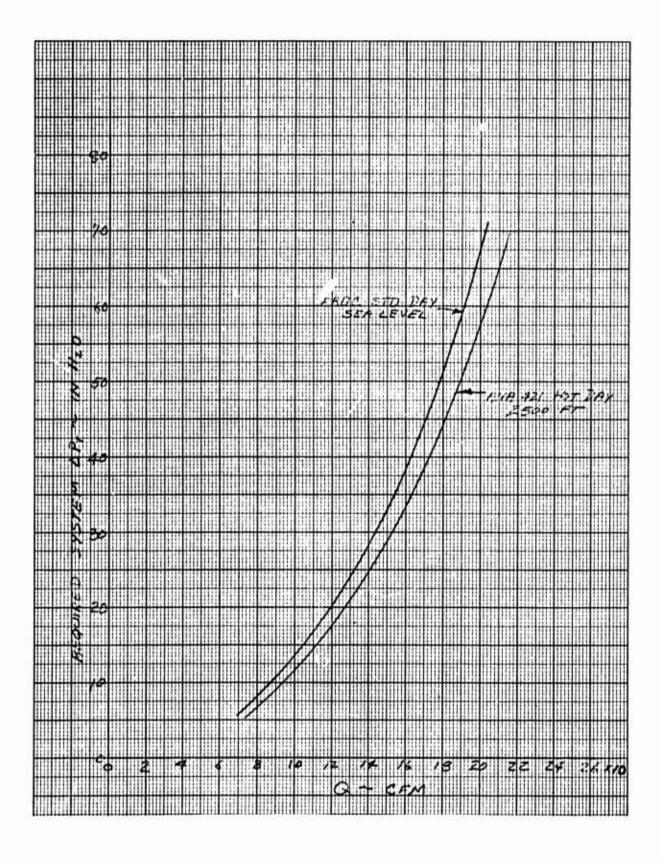


Figure 9.24 Duct Pressure Loss - Electronic Compartment to Nose Fan Ejector Vs Cooling Air Flow -Standard Day Sea Level, and Hot Day 2,500 Ft.

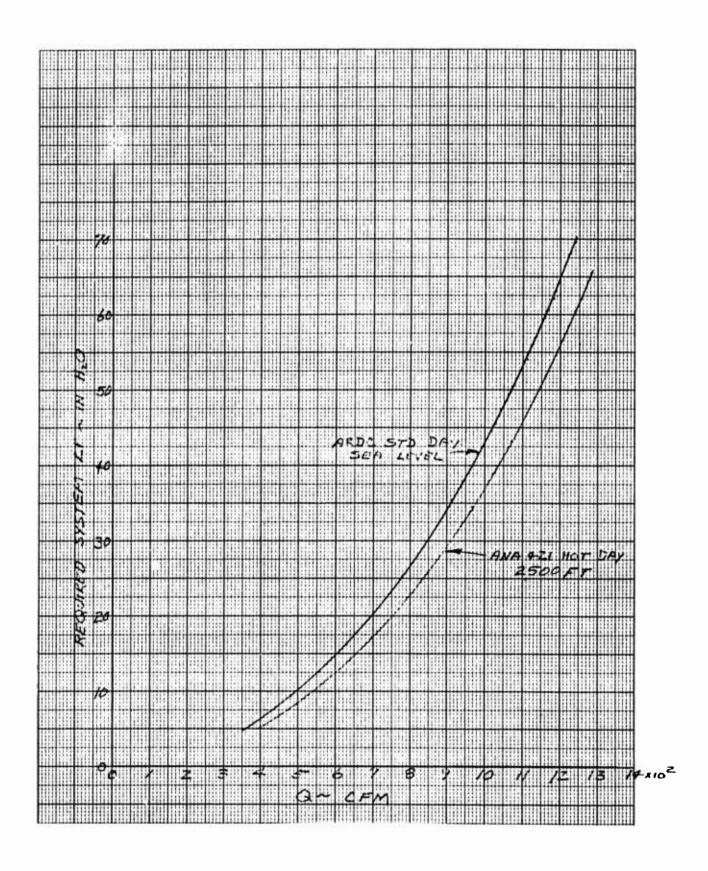


Figure 9.25 Duct Pressure Loss - Center Fuselage to Flap Actuator Compartment Vs Cooling Air Flow -Standard Day Sea Level, and Hot Day 2,500 Ft.

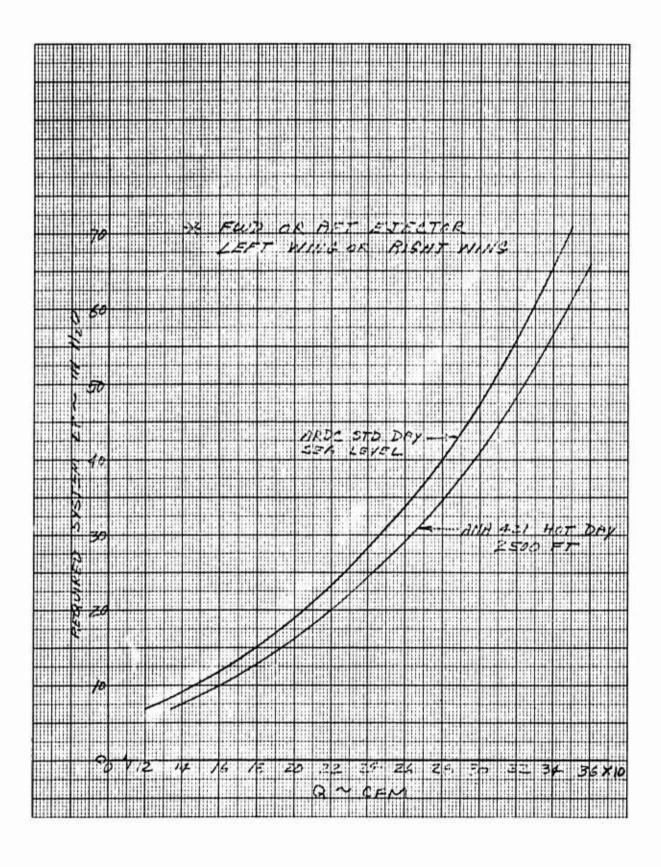


Figure 9.26 Duct Pressure Loss - Center Fuselage to Wing Fan Ejector Vs Cooling Air Flow - Standard Day Sea Level, and Hot Day 2,500 Ft.

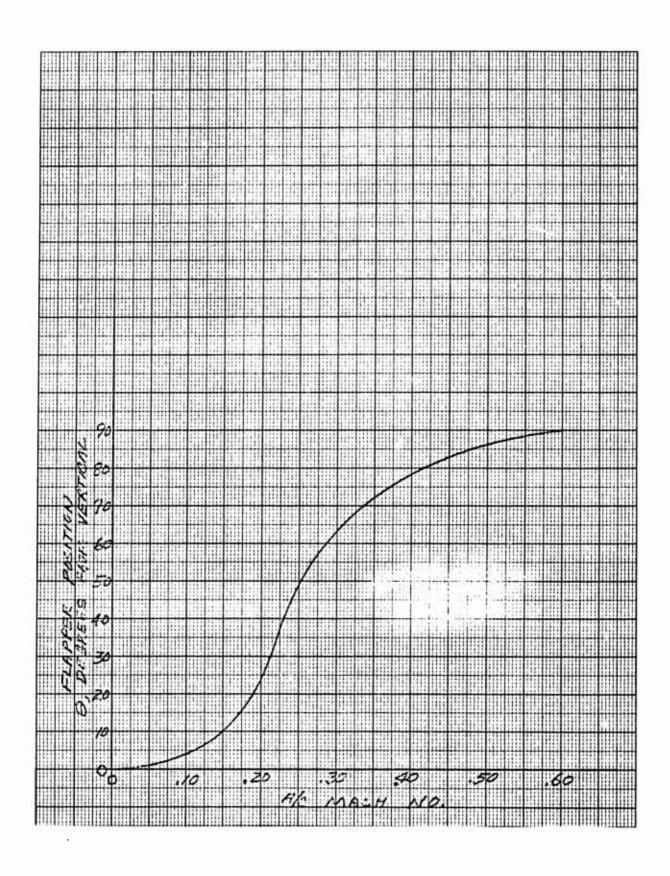


Figure 9, 27 Boundary Layer Bleed Duct Aft Flapper Position Vs Aircraft Mach No. - Standard Day, Sea Level, 100% RPM

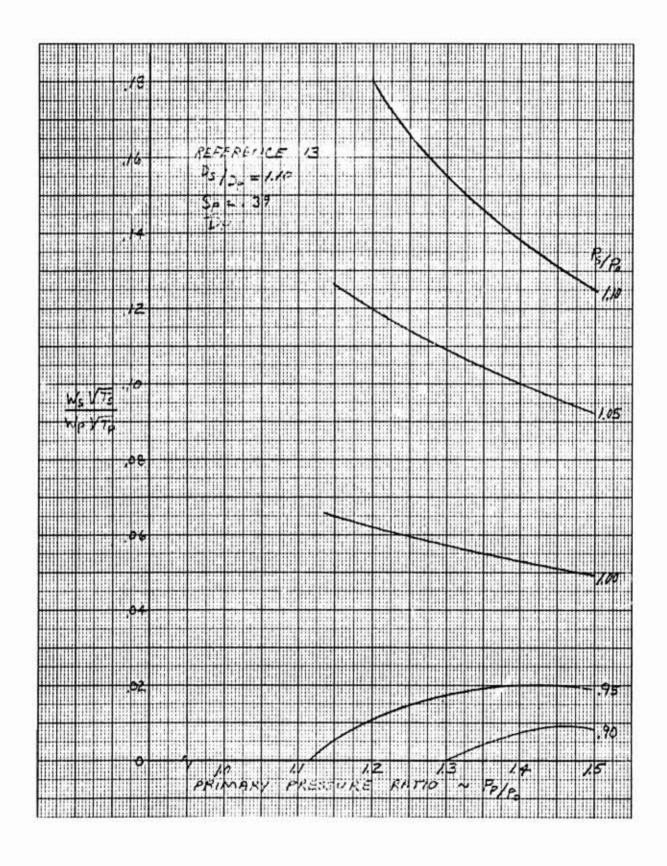


Figure 9.28 Tailpipe Ejector Weight Flow Ratio Vs Primary and Secondary Pressure Ratio, $P_D/P_0 = 1.1$ to 1.5

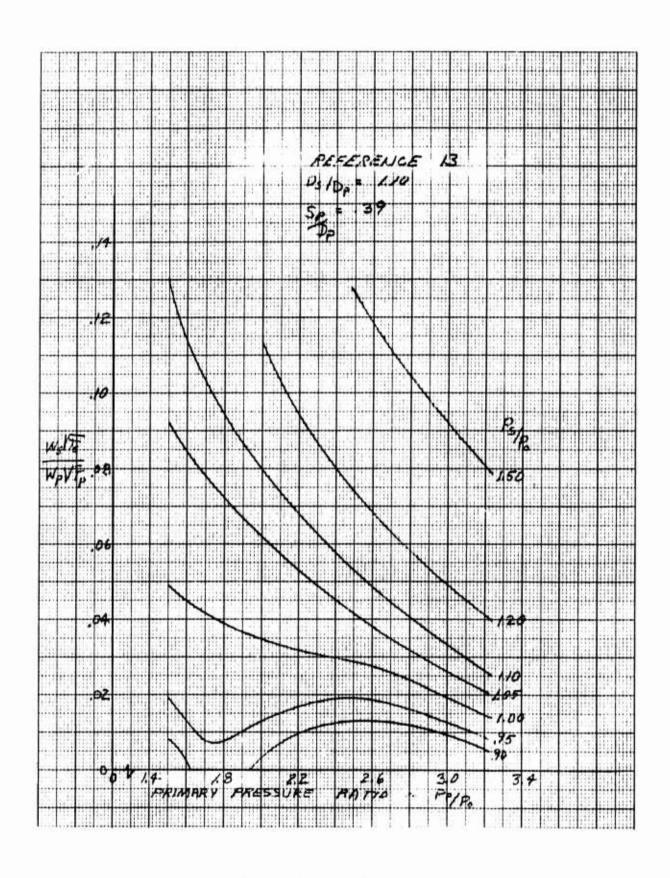


Figure 9. 29 Tailpipe Ejector Weight Flow Ratio Vs Primary and Secondary Pressure Ratio, P_p/P_0 = 1.5 to 3.2

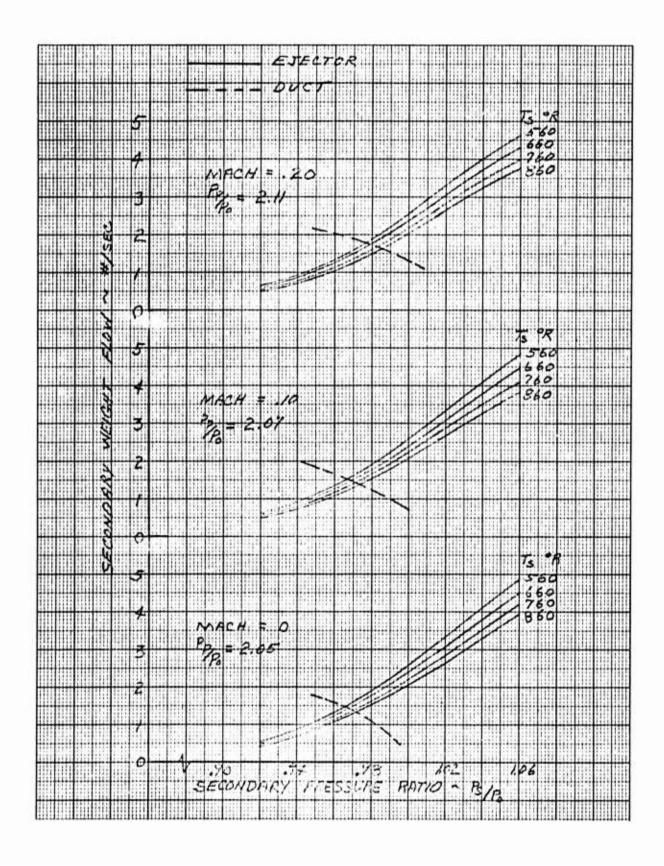


Figure 9.30 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 100% RPM and Mach No. = 0, 0.1 and 0.2

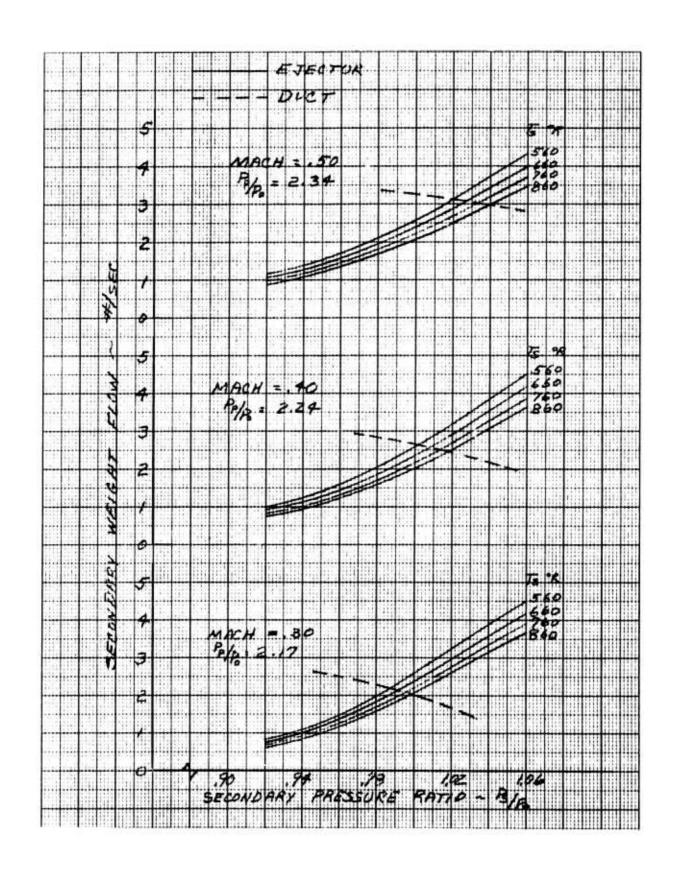


Figure 9.31 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 100% RPM and Mach No. = 0.3, 0.4 and 0.5

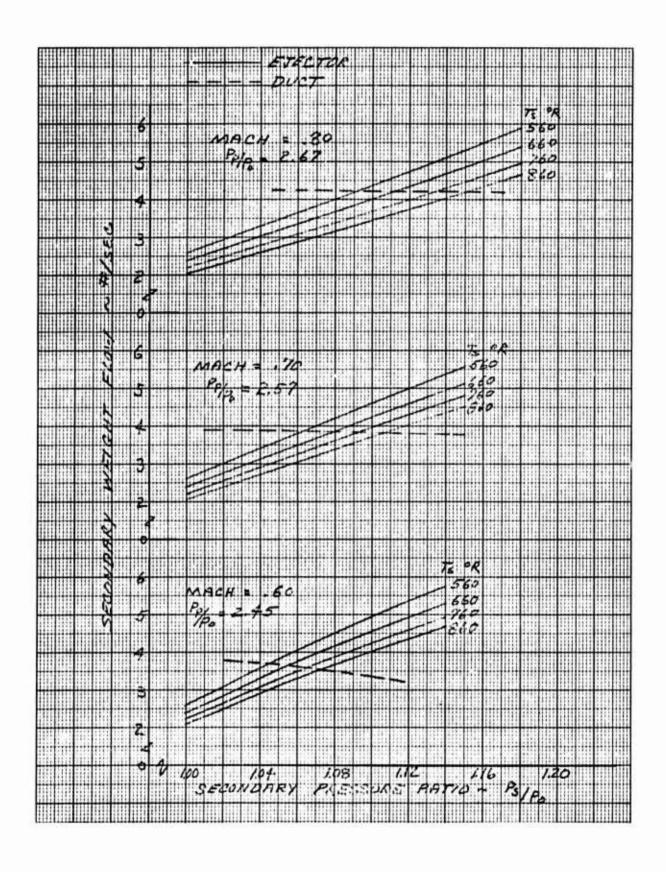


Figure 9.32 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 100% RPM and Mach No. = 0.6, 0.7 and 0.8

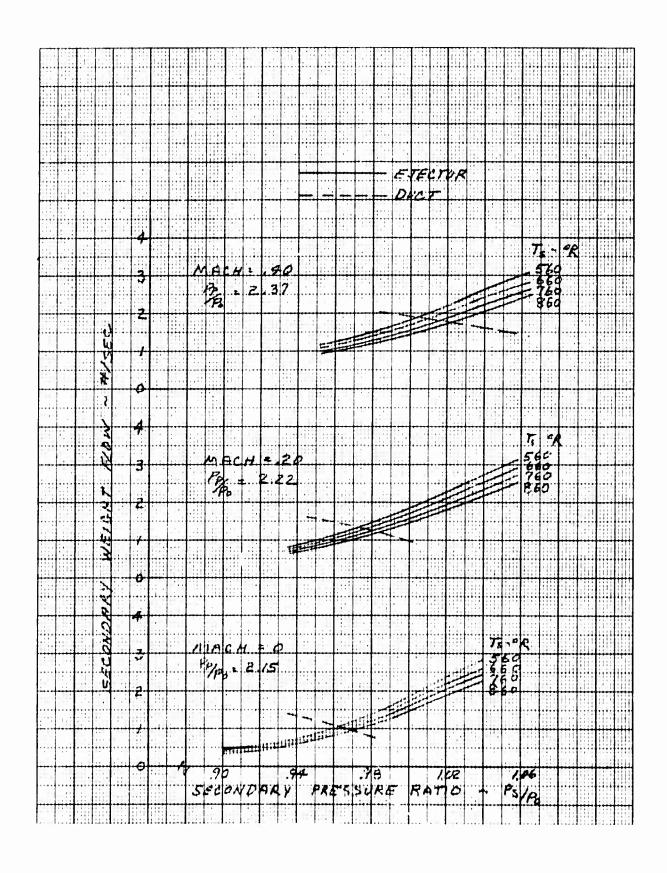


Figure 9.33 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day 10,000 Ft., 100% RPM and Mach No. = 0, 0.2 and 0.4

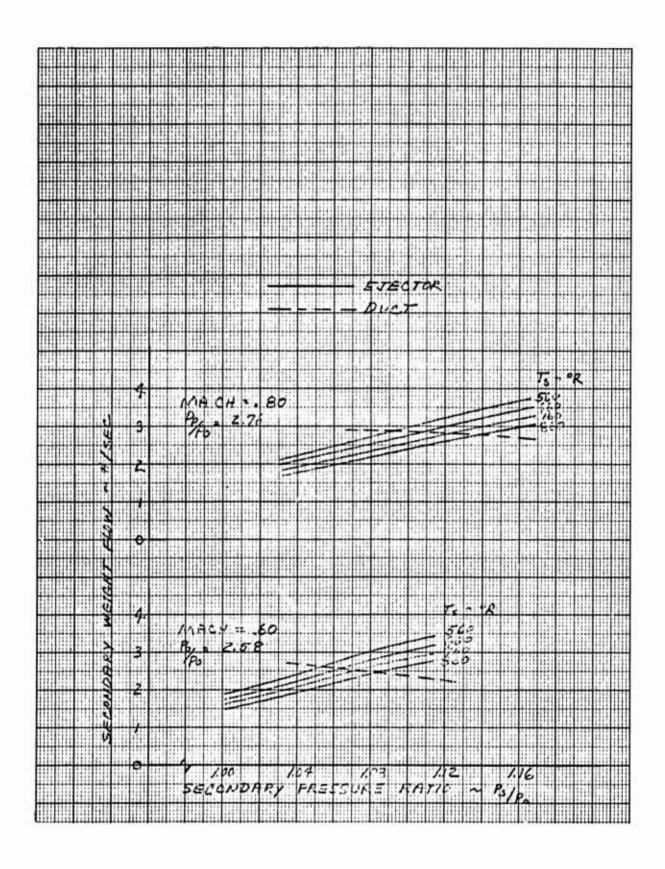


Figure 9.34 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day 10,000 Ft., 100% RPM and Mach No. = 0.6 and 0.8

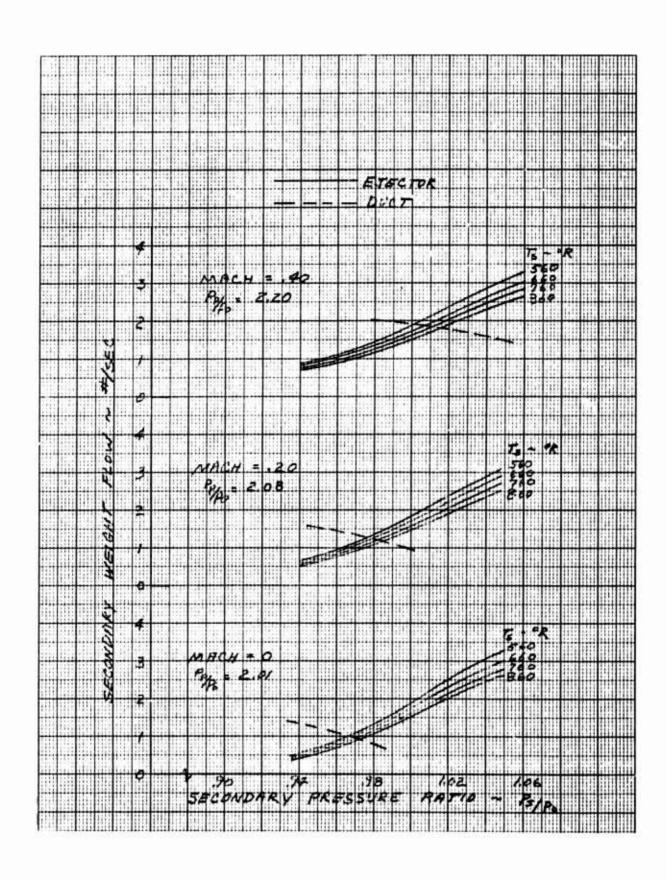


Figure 9.35 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Hot Day, 10,000 Ft., 100% RPM and Mach No. = 0, 0.2 and 0.4

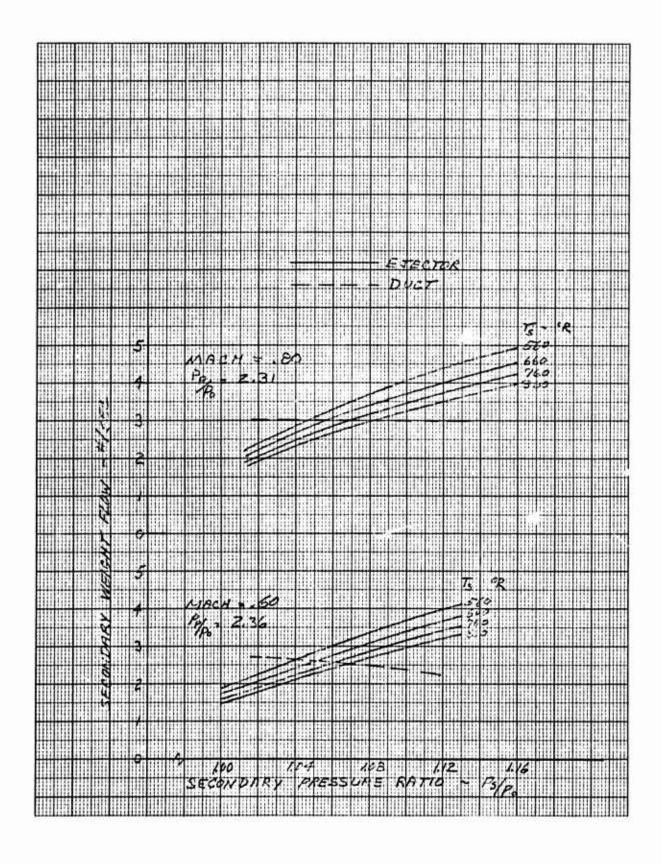


Figure 9.36 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Hot Day 10,000 Ft., 100% RPM and Mach No. = 0.6 and 0.8

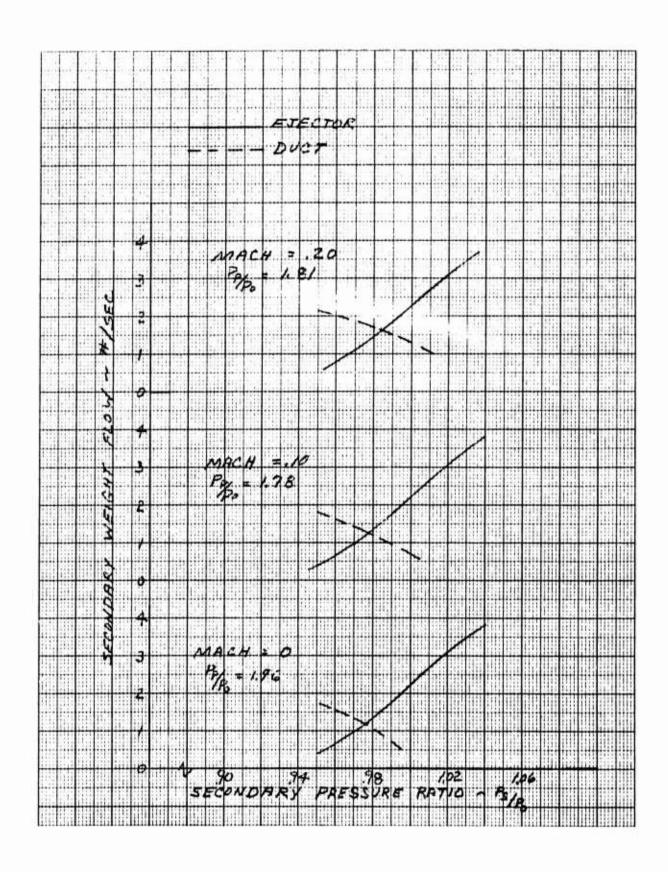


Figure 9.37 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 95% RPM and Mach No. = 0, 0.1 and 0.2

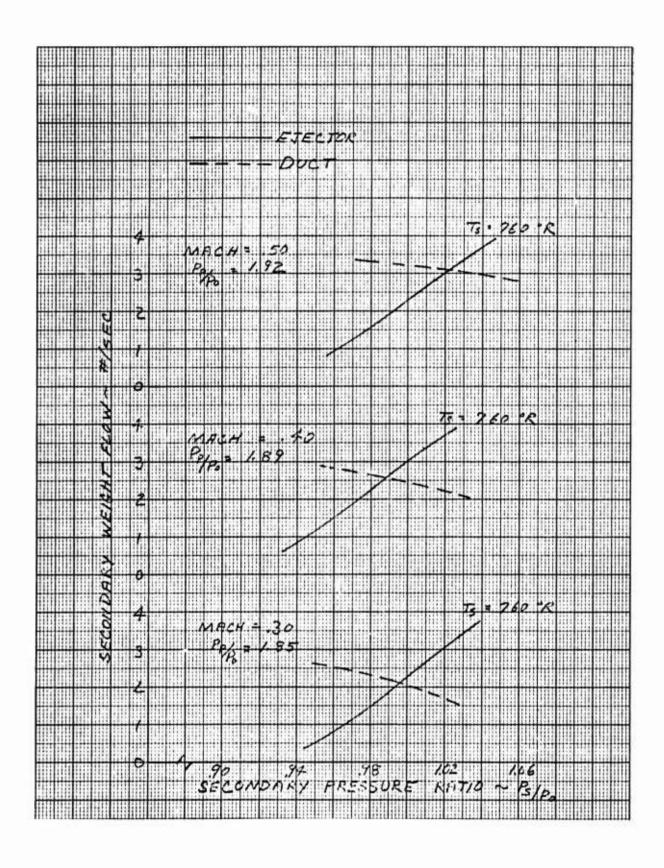


Figure 9.38 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 95% RPM and Mach No. = 0.3, 0.4 and 0.5

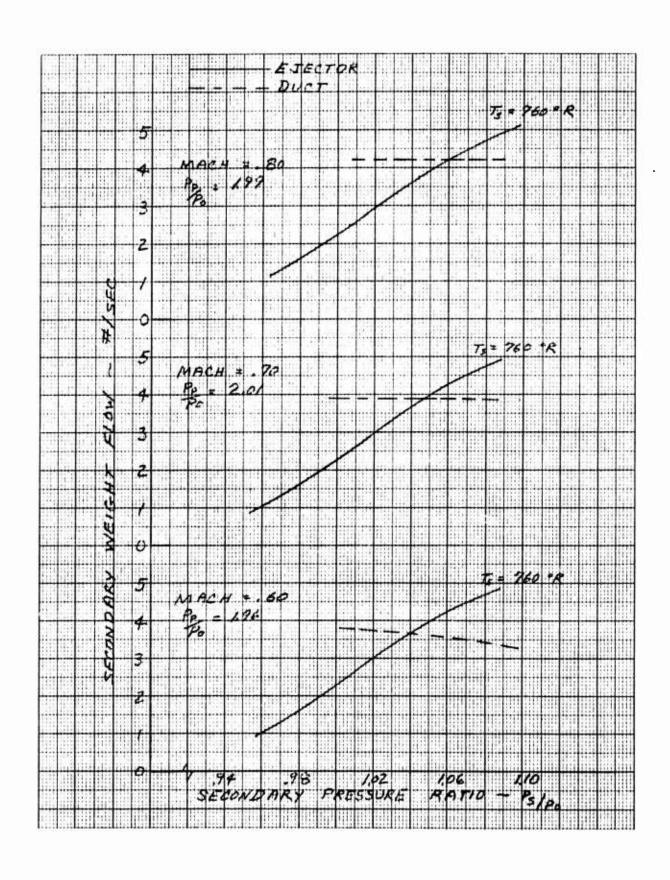


Figure 9.39 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 95% RPM and Mach No. = 0.6, 0.7 and 0.8

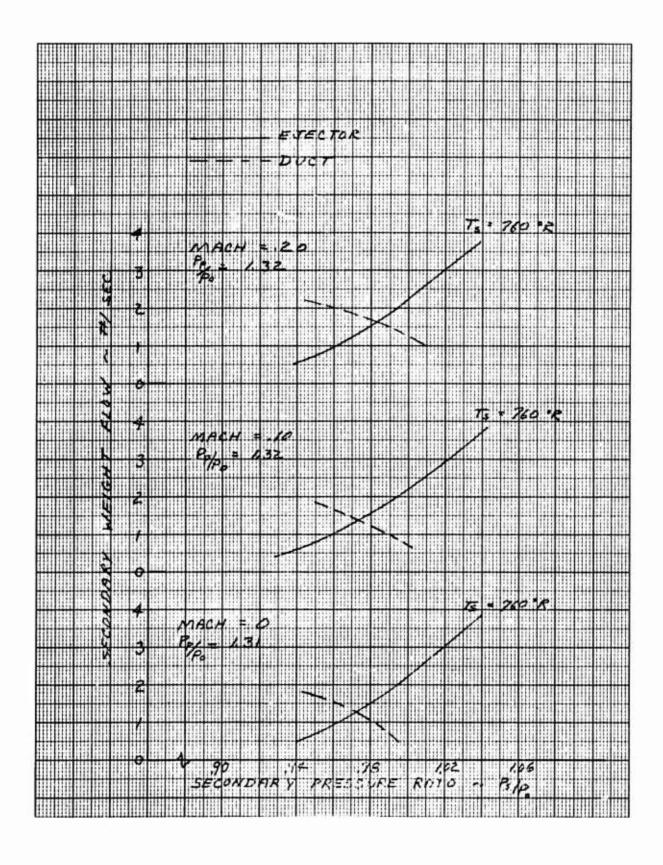


Figure 9.40 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 85% RPM and Mach No. = 0, 0.1 and 0.2

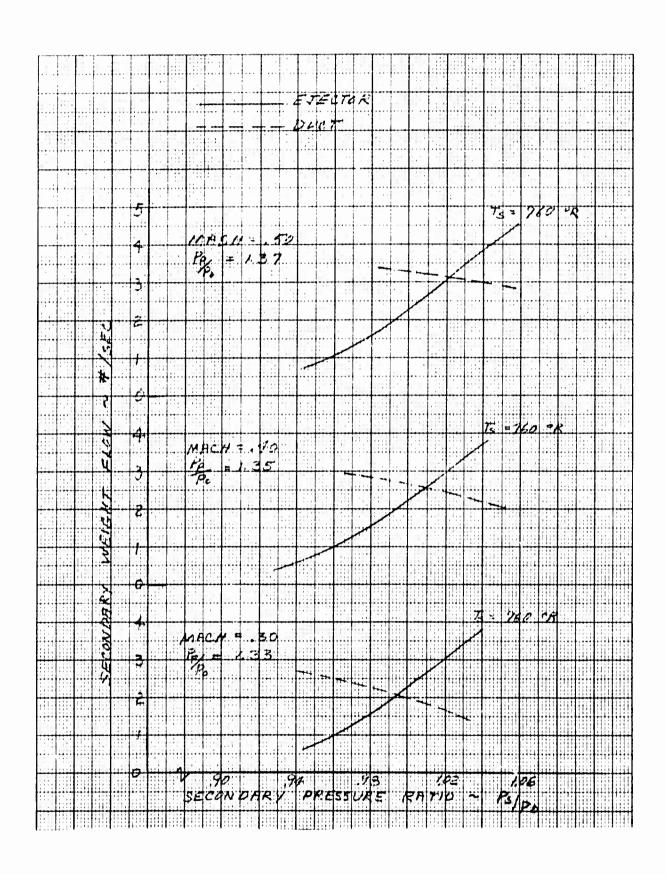


Figure 9.41 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 85% RPM and Mach No. = 0.3, 0.4 and 0.5

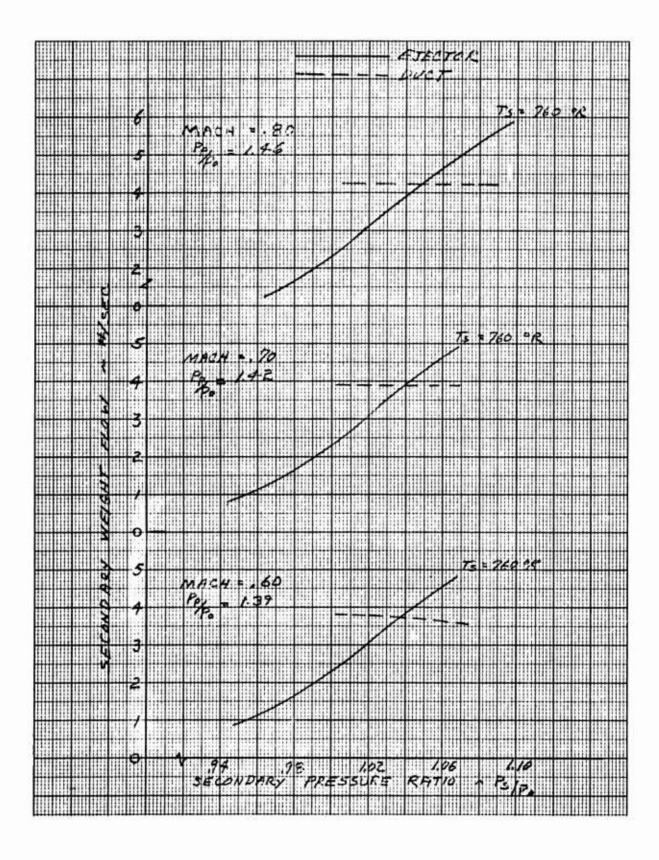


Figure 9.42 Tailpipe Ejector Secondary Weight Flow Vs Secondary Pressure Ratio - Standard Day Sea Level, 85% RPM and Mach No. = 0.6, 0.7 and 0.8

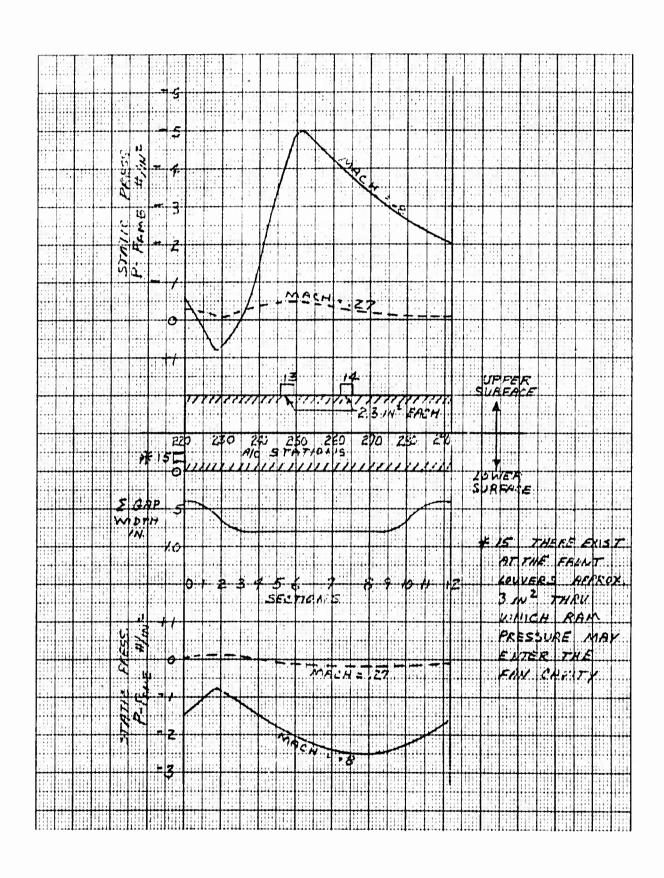


Figure 9.43 Wing Fan Region Chordwise Pressure Distribution and Leakage Area

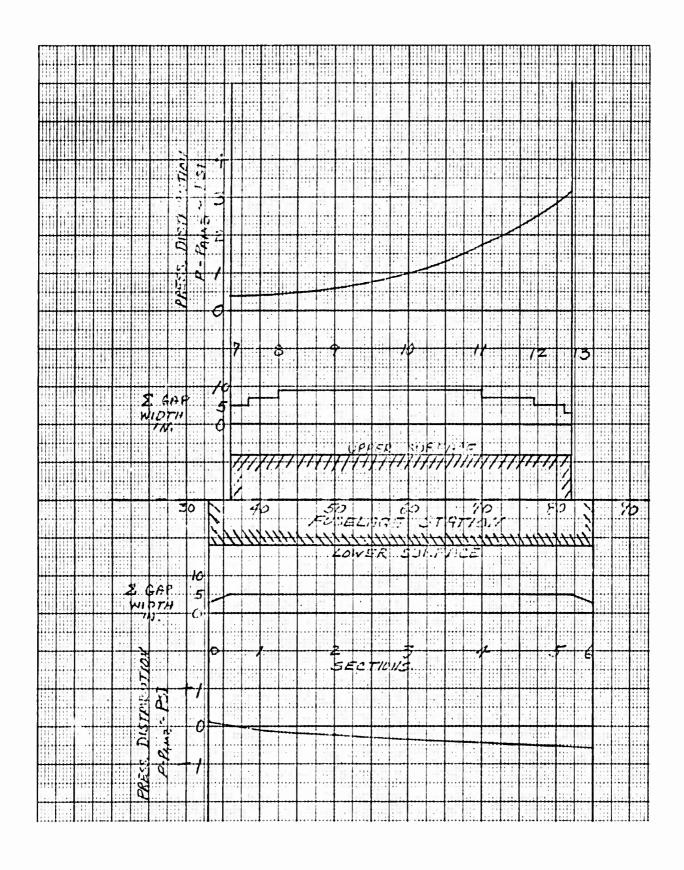


Figure 9.44 Nose Fan Doors - Pressure Distribution And Leakage Area - Mach No. = 0.8

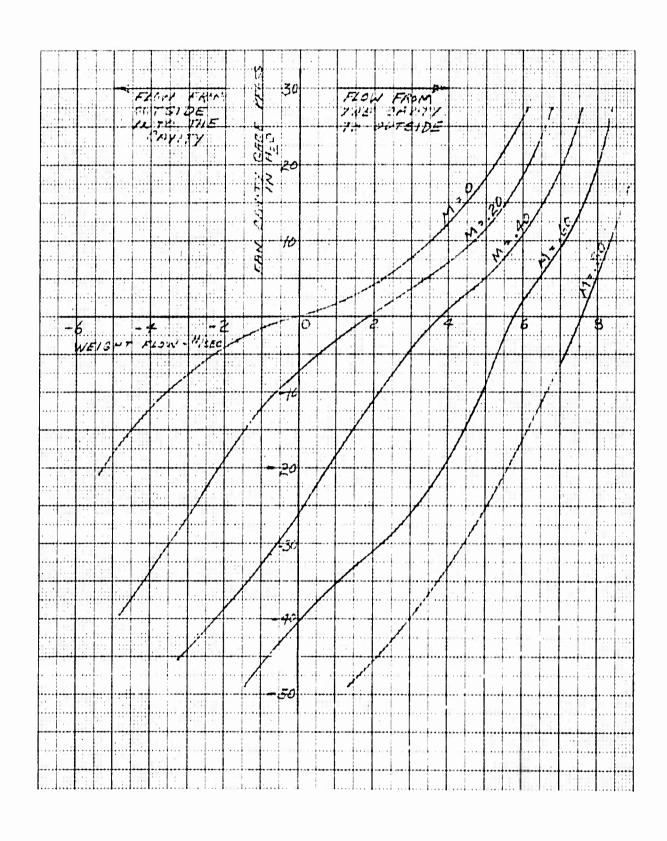


Figure 9.45 Wing Fan Cavity Pressure Vs Air Flow Between Fan Cavity And Outside and Mach No. - Standard Day, Sea Level

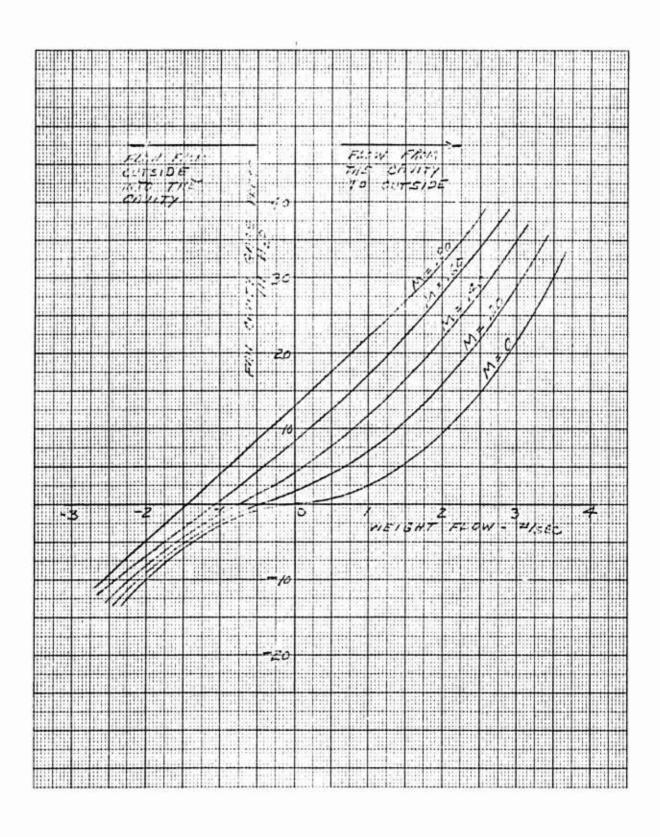


Figure 9.46 Nose Fan Cavity Pressure Vs Air Flow Between Fan Cavity And Outside, and Mach No. - Standard Day, Sea Level

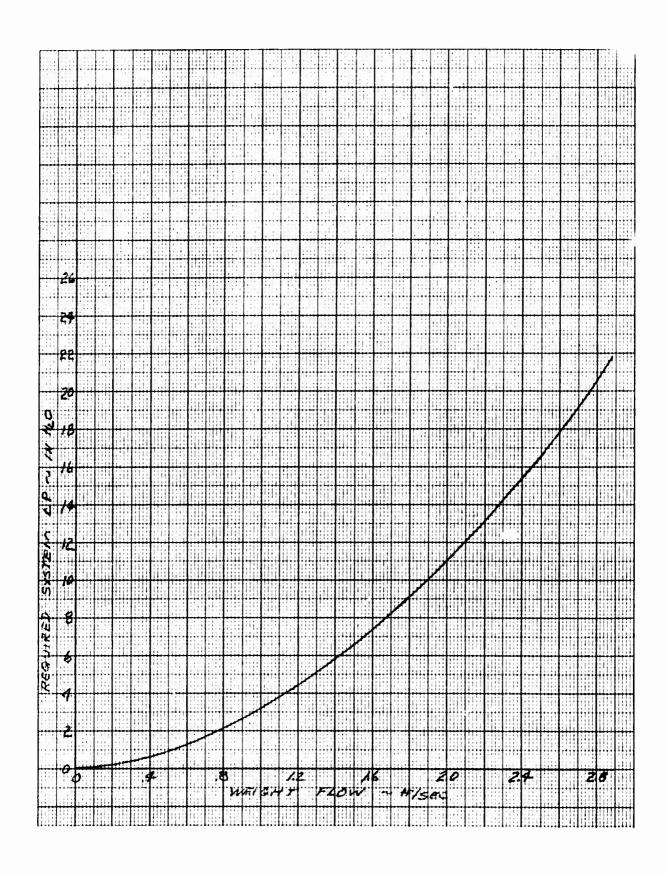


Figure 9.47 Pressure Loss Vs Weight Flow in the Lift Fan Supply Ducts from the Nose Fan Cavity to the Wing Fan Cavity - Standard Day Sea Level

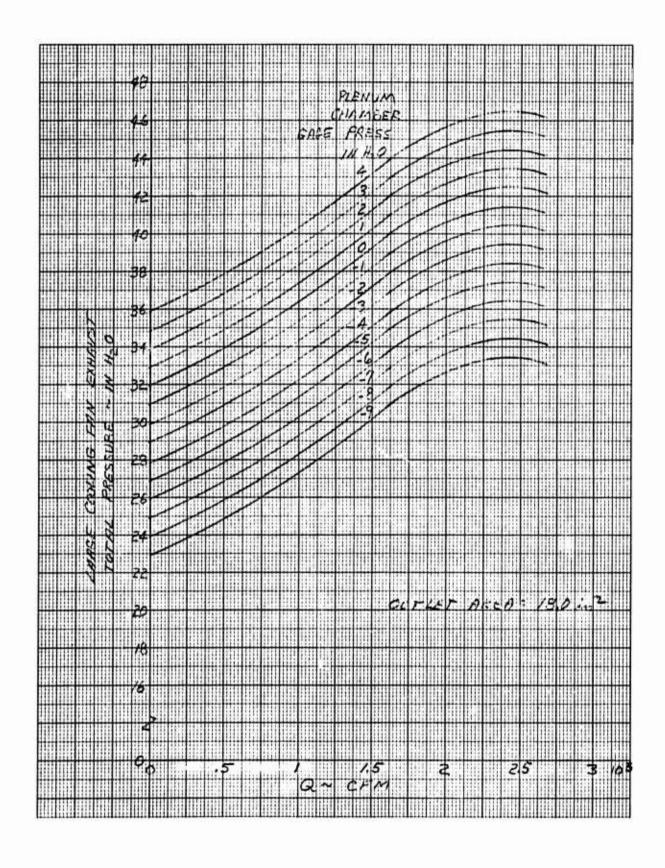


Figure 9.48 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sea Level, 100% RPM

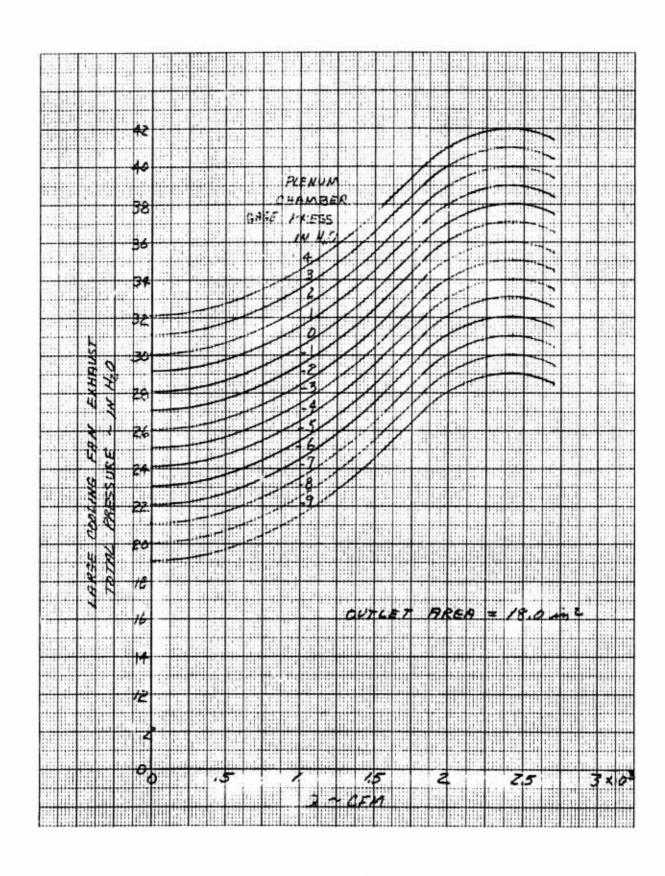


Figure 9.49 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure - Hot Day, 2,500 Ft., 100% RPM

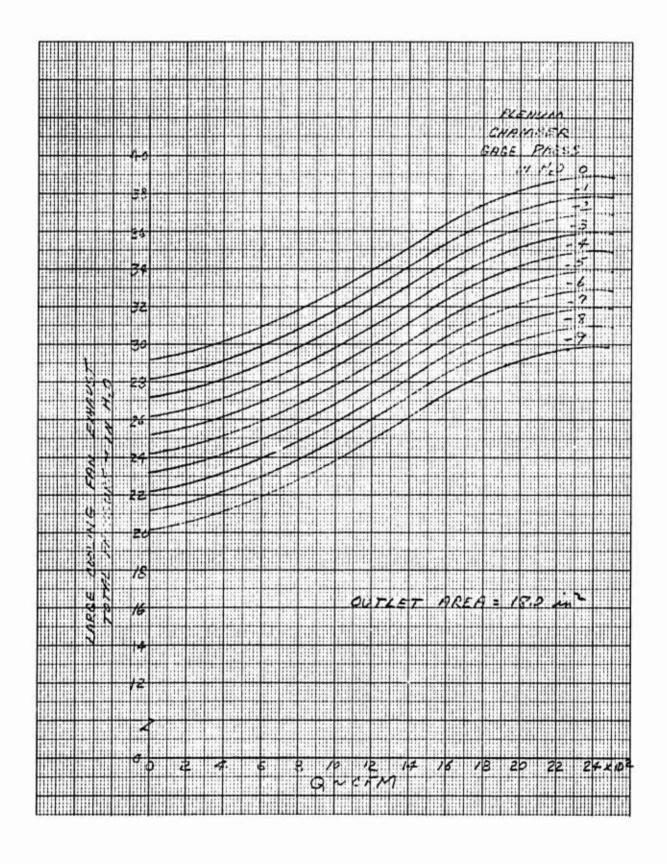


Figure 9.50 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sea Level, 95% RPM

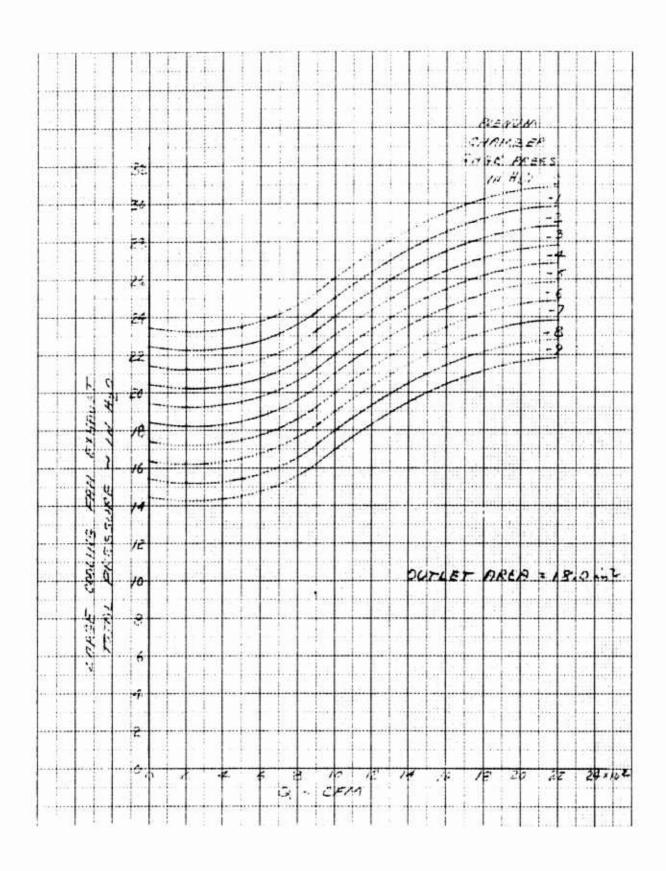


Figure 9.51 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sca Level, 85% RPM

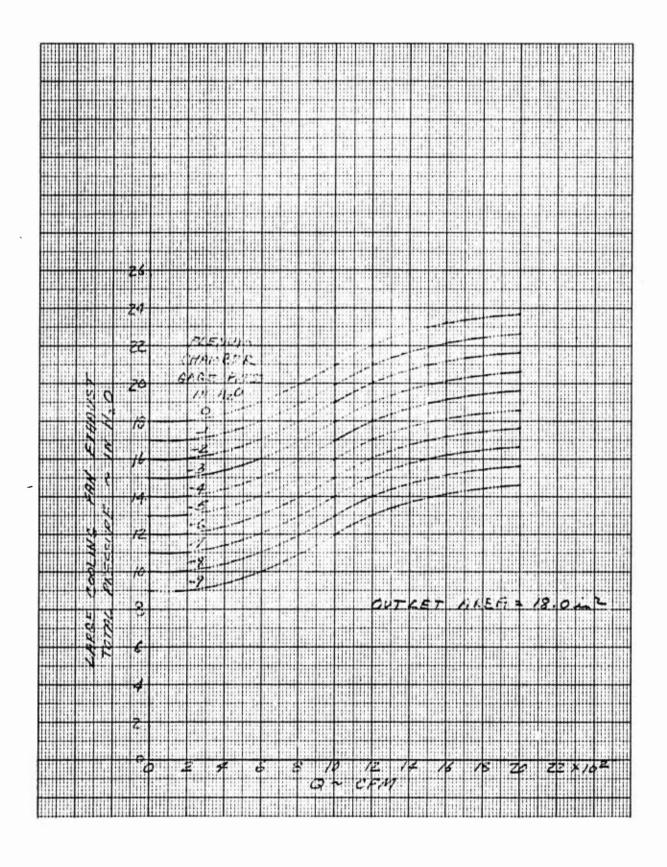


Figure 9.52 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sea Level, 75% RPM

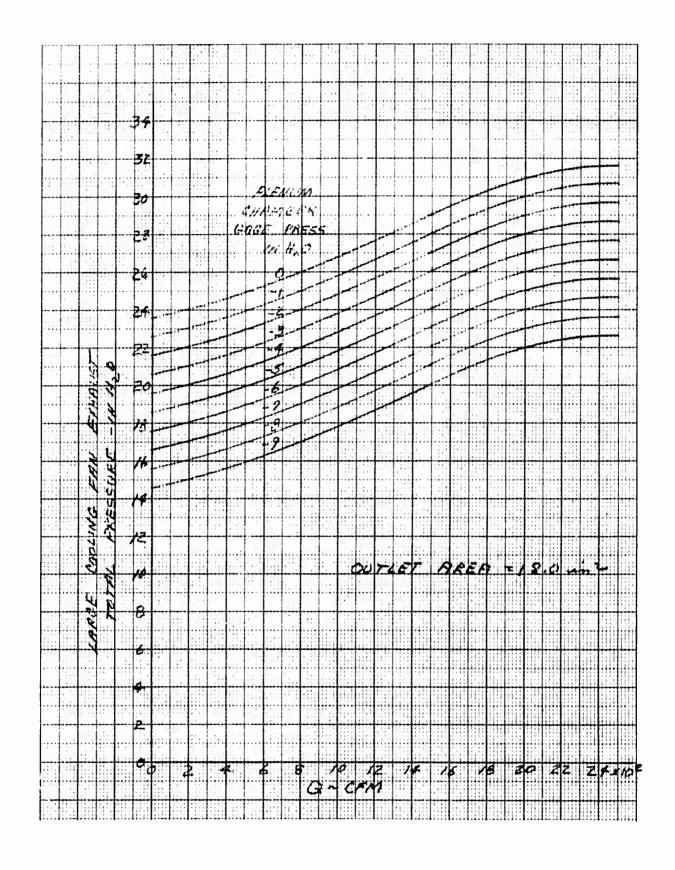


Figure 9.53 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, 10,000 Ft., 100% RPM

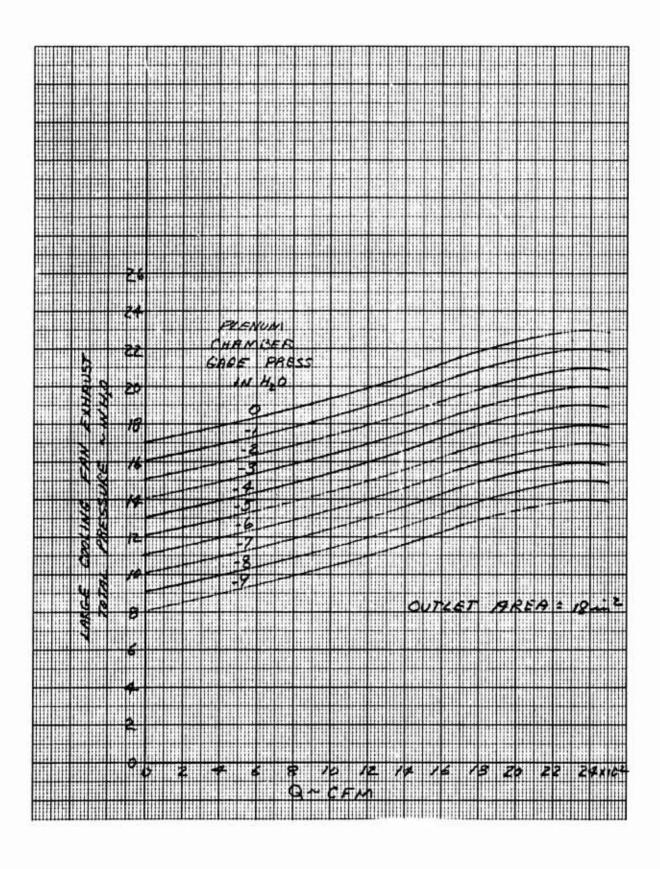


Figure 9.54 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, 20,000 Ft., 100% RPM

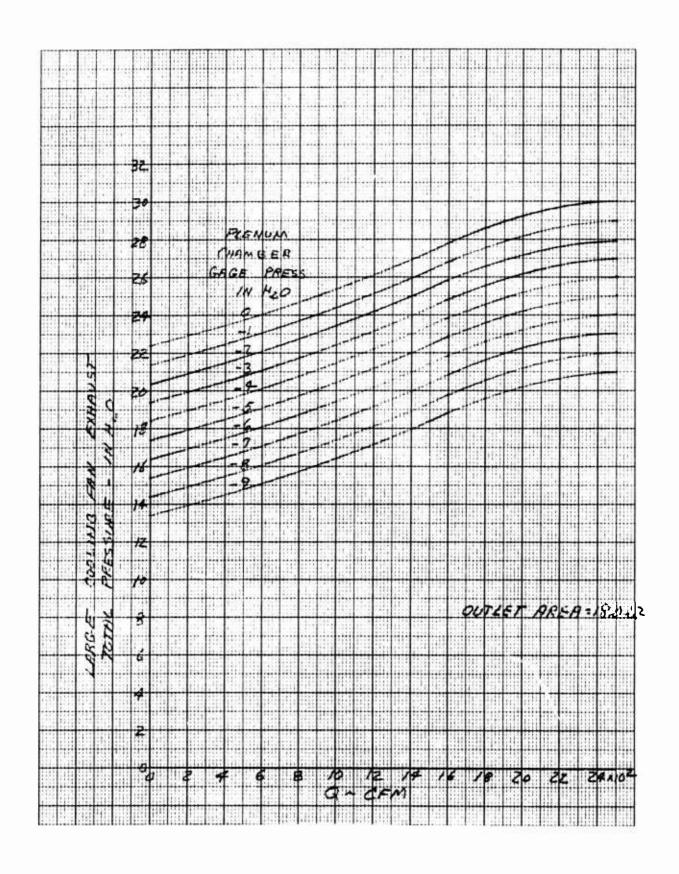


Figure 9.55 Large Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Hot Day, 10,000 Ft., 100% RPM

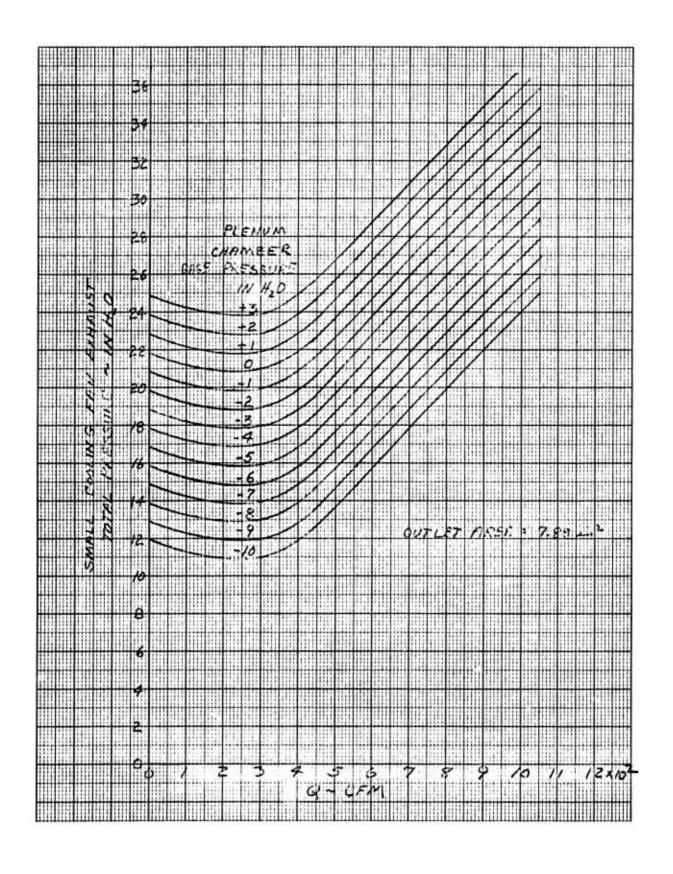


Figure 9.56 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sea Level, 100% RPM

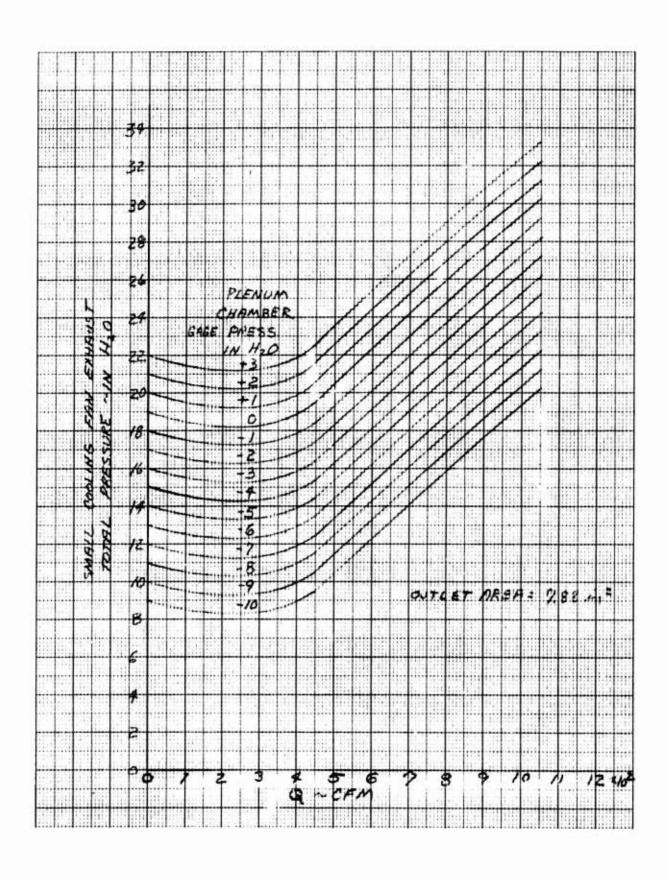


Figure 9.57 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Hot Day, 2,500 Ft., 100% RPM

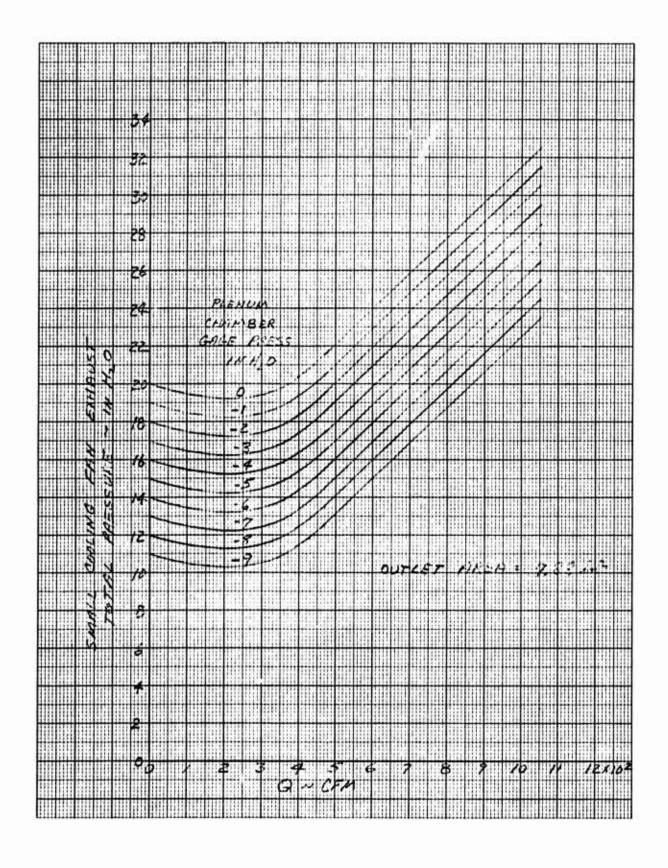


Figure 9.58 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sea Level, 95% RPM

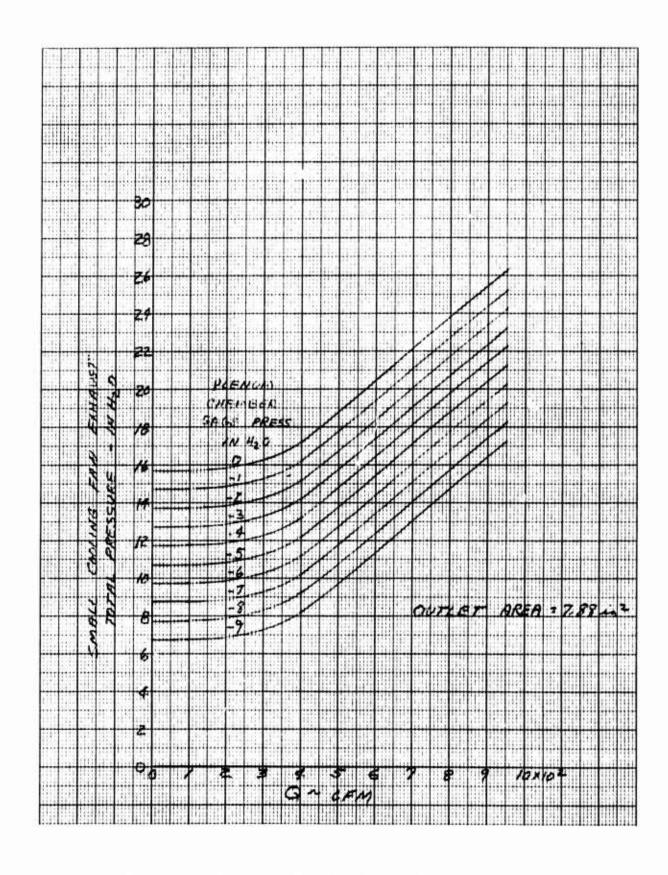


Figure 9.59 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sca Level, 85% RPM

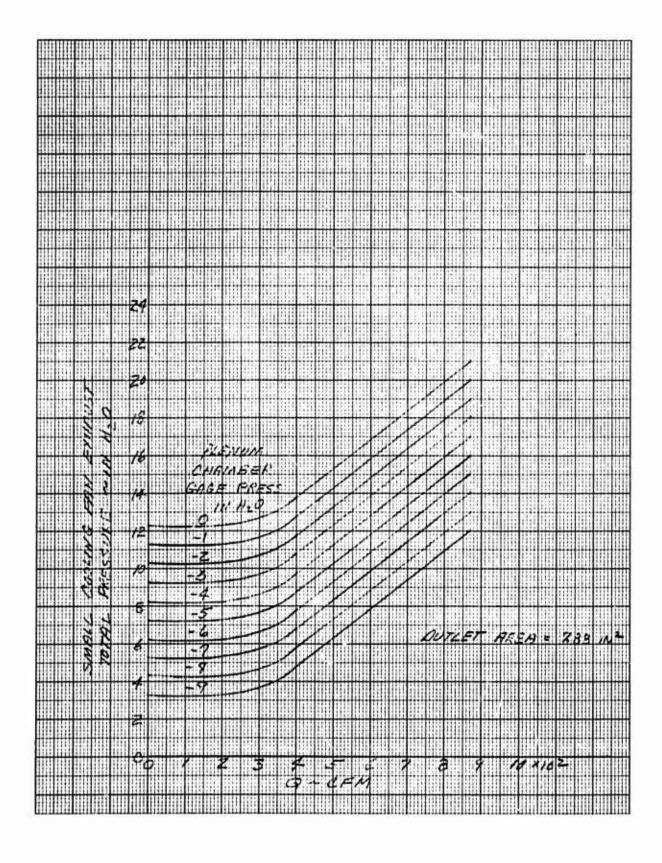


Figure 9.60 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, Sea Level, 75% RPM

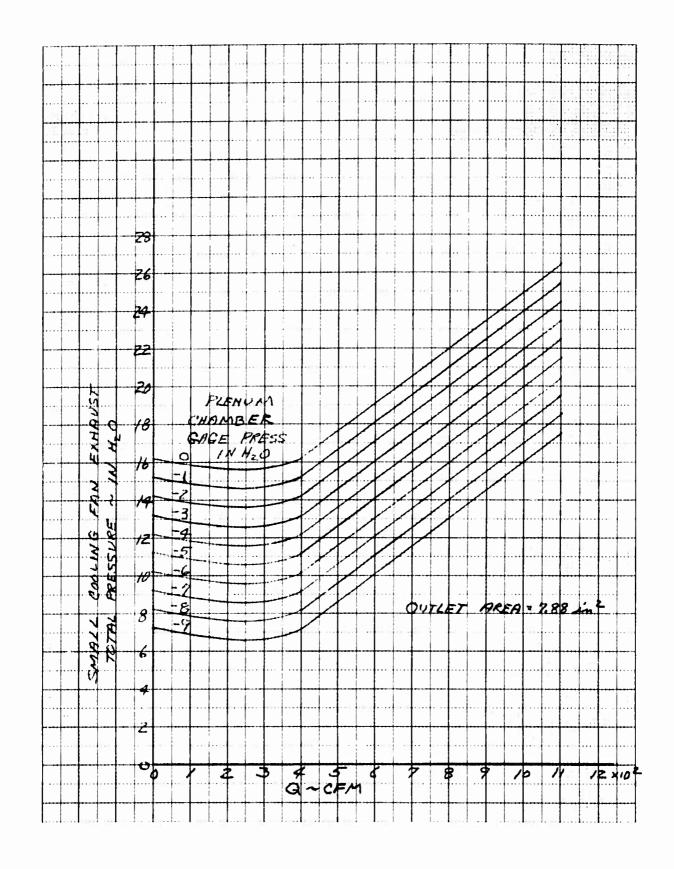


Figure 9.61 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, 10,000 Ft., 100% RPM

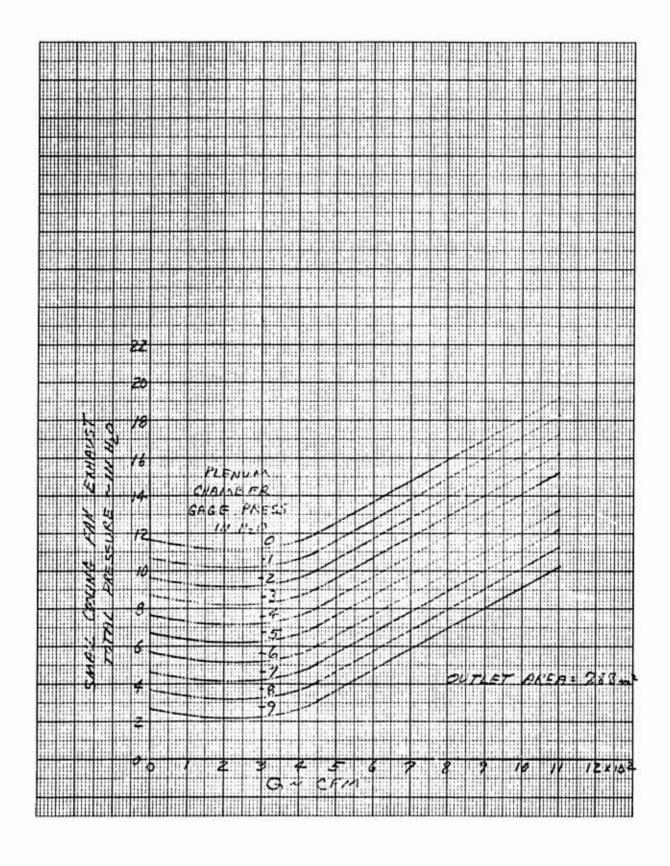


Figure 9.62 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Standard Day, 20,000 Ft., 100% RPM

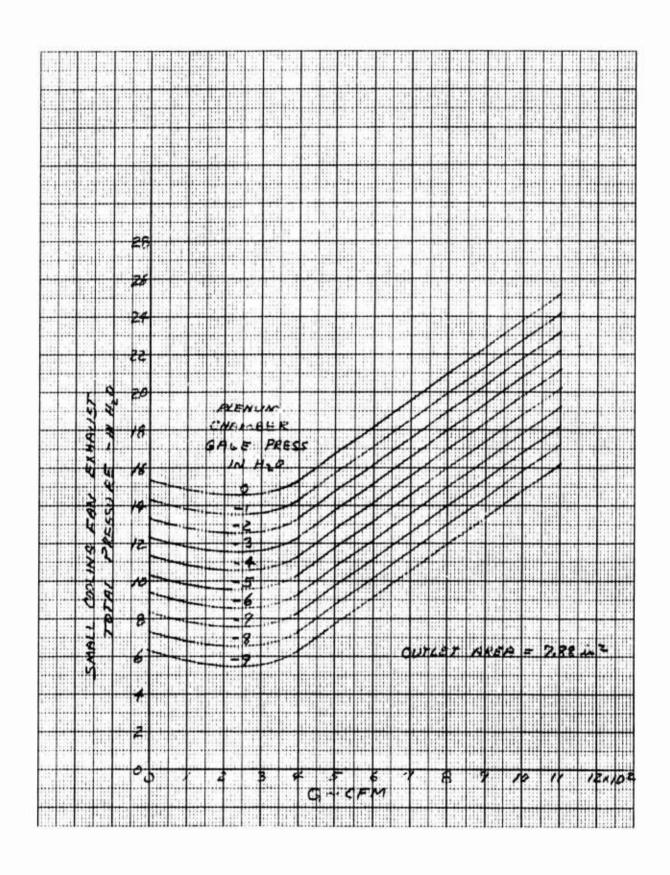


Figure 9.63 Small Cooling Fan Exhaust Total Pressure Vs Flow Rate and Plenum Chamber Pressure -Hot Day, 10,000 Ft., 100% RPM

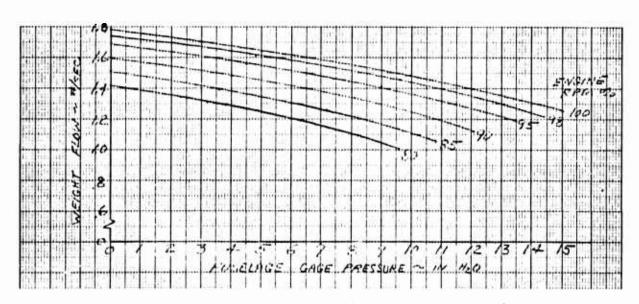


Figure 9.64 Cooling Air Weight Flow - Cockpit to Cooling
Fan Compartment Vs Fuselage Pressure and %
RPM - Fan Mode, Standard Day, Sea Level

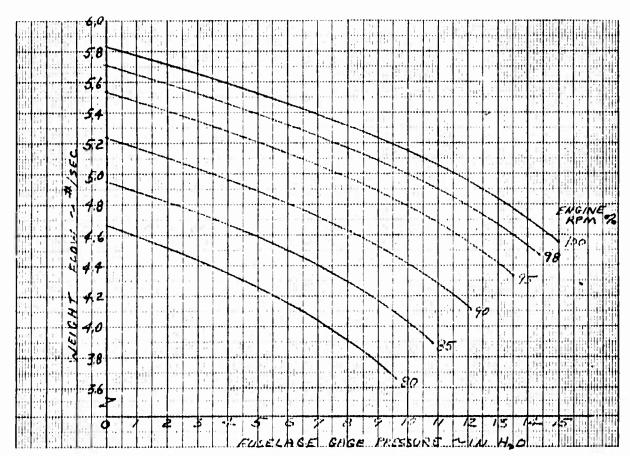


Figure 9.65 Cooling Air Weight Flow - Fuselage Ports to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

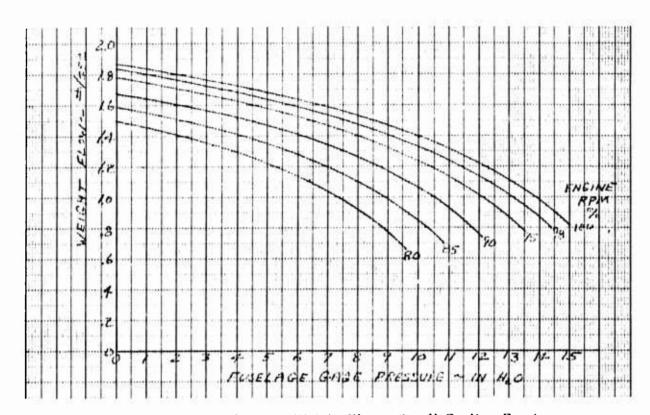


Figure 9.66 Cooling Air Weight Flow - Small Cooling Fan to Electronic Compartment Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

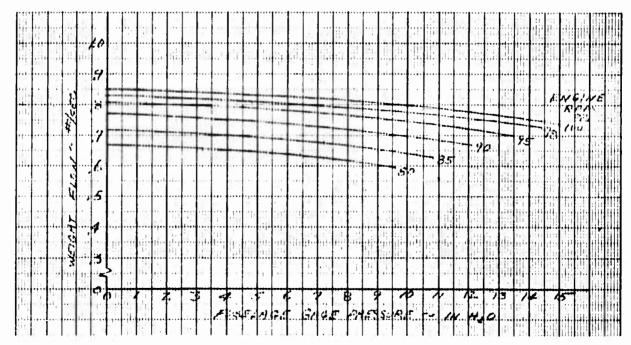


Figure 9.67 Cooling Air Weight Flow - Small Cooling Fan to Generator Vs Fuselage Pressure and % RPM -Fan Mode, Standard Day, Sea Level

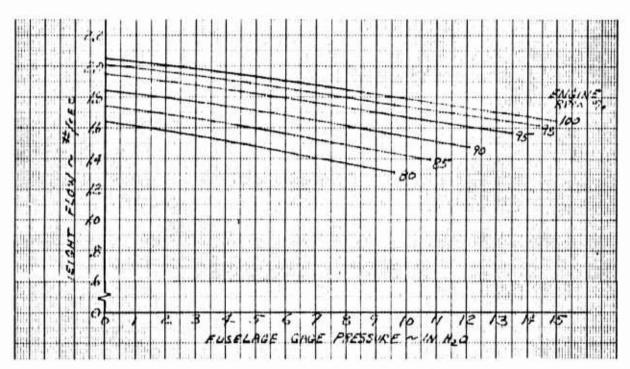


Figure 9.68 Cooling Air Weight Flow - L. H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

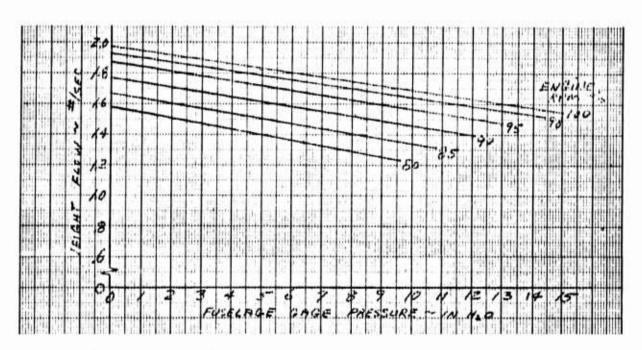


Figure 9.69 Cooling Air Weight Flow - R.H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

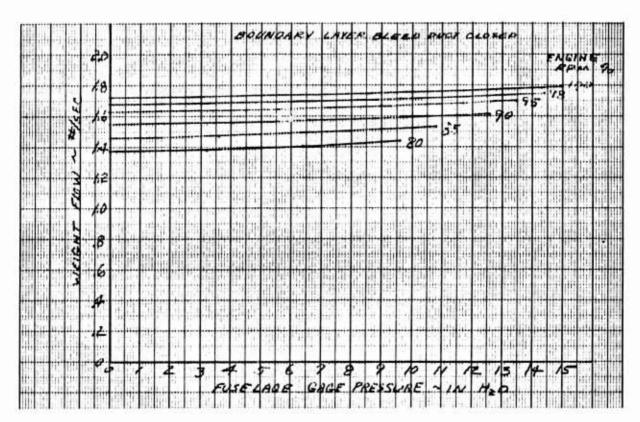


Figure 9.70 Cooling Air Weight Flow - Large Cooling Fan to Tailpipe Ejector Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

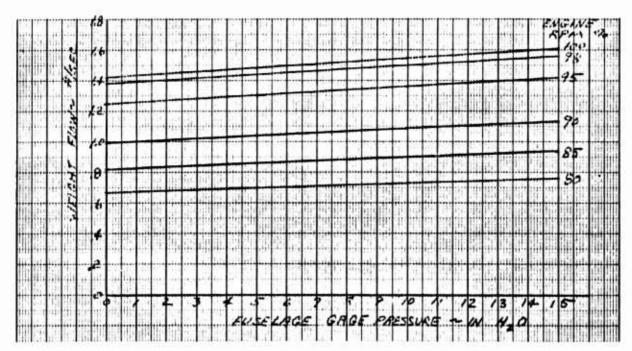


Figure 9.71 Cooling Air Weight Flow - Center Fuselage to Wing Fan Cavity Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

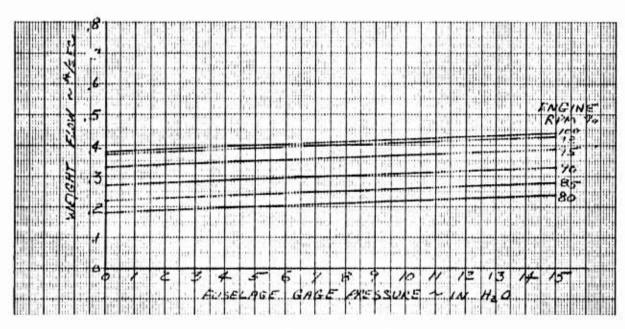


Figure 9.72 Cooling Air Weight Flow - Forward Fuselage to Nose Fan Cavity Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

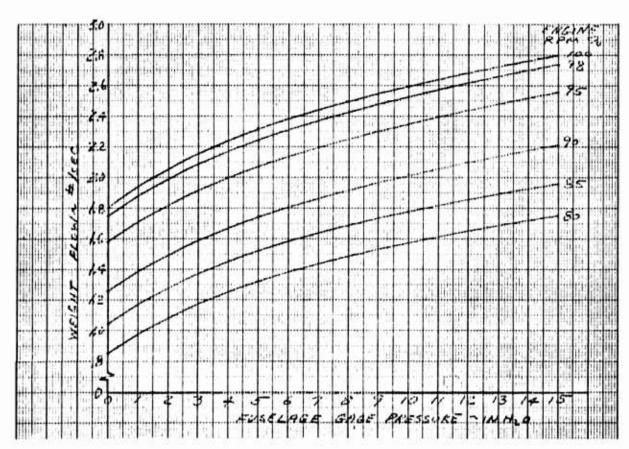


Figure 9.73 Cooling Air Weight Flow - Wing and Nose Fan Ejectors and Flap Actuator Slot to Outside Vs Fuselage Pressure and % RPM - Fan Mode, Standard Day, Sea Level

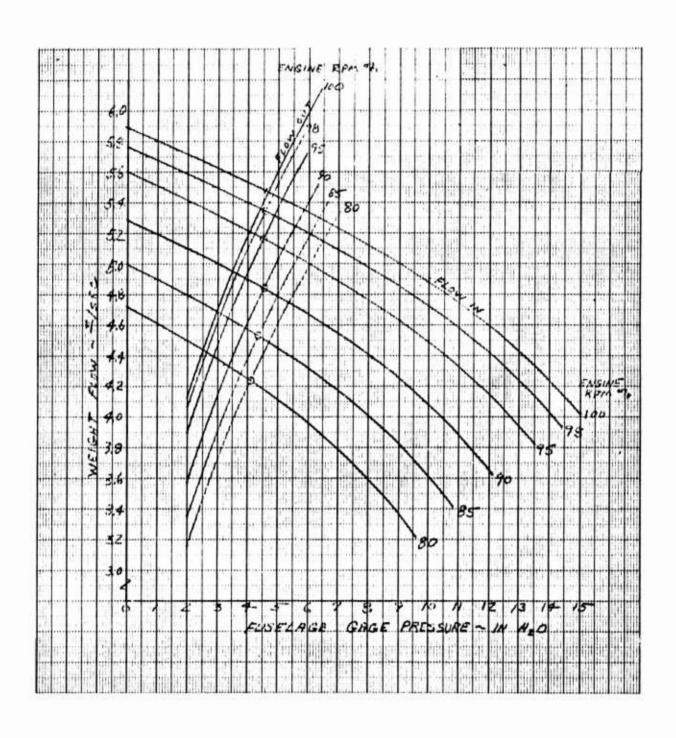


Figure 9.74 Cooling Air Weight Flow - Balance of Flow Thru
The Lower Fuselage Vs Fuselage Pressure and
% RPM - Fan Mode, Standard Day, Sea Level

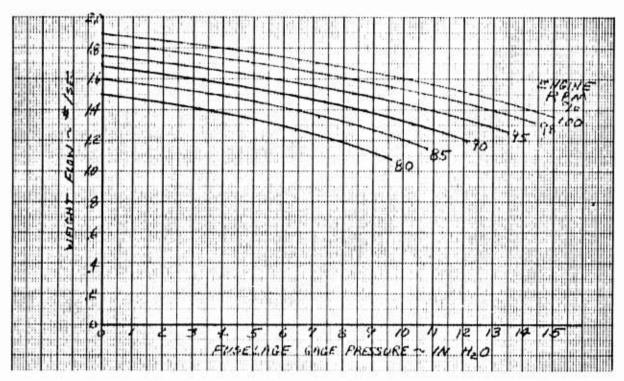


Figure 9.75 Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day, Sea Level

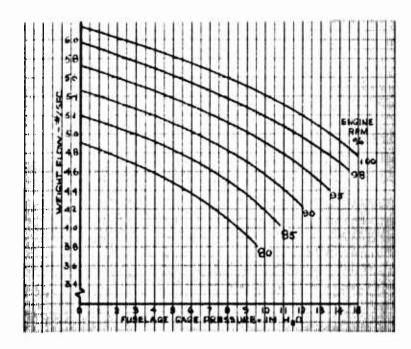


Figure 9.76 Cooling Air Weight Flow - Fuselage Ports to
Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Mode, Standard
Day, Sea Level

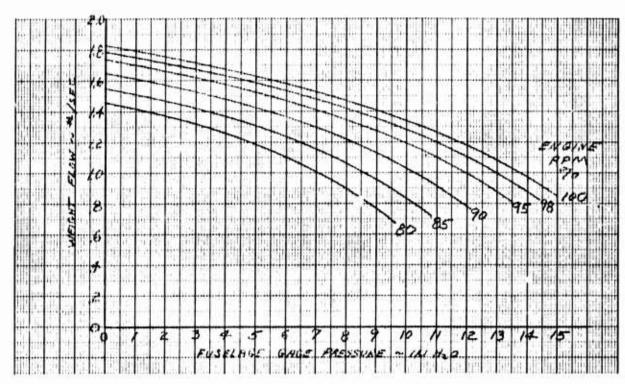


Figure 9.77 Cooling Air Weight Flow - Small Cooling Fan to Electronic Compartment Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day, Sea Level

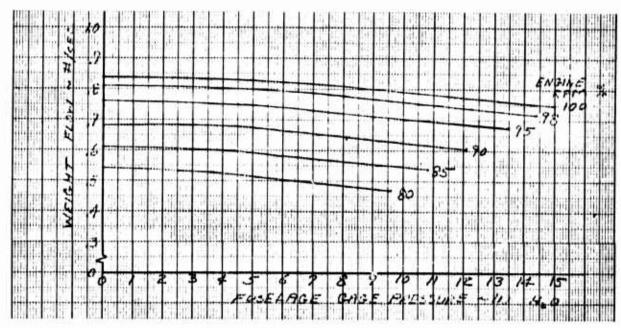


Figure 9.78 Cooling Air Weight Flow - Small Cooling Fan to Generator Vs Fuselage Pressure and % RPM -Conventional Mode, Standard Day, Sea Level

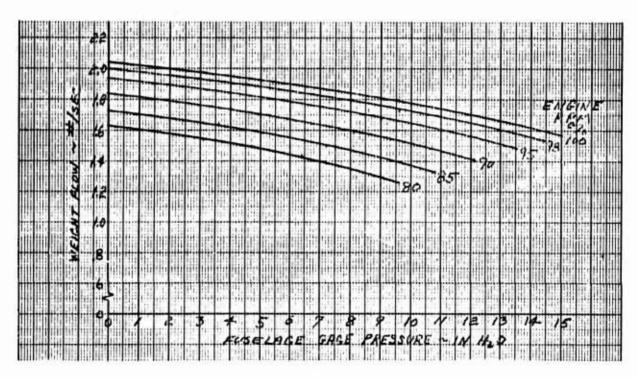


Figure 9.79 Cooling Air Weight Flow - L.H. Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM -Conventional Mode, Standard Day, Sea Level

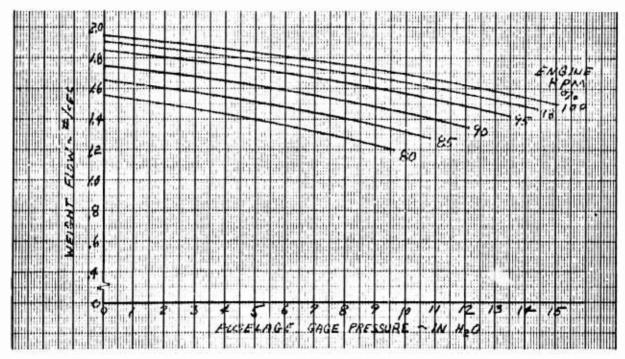


Figure 9.80 Cooling Air Weight Flow - R.H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day, Sea Level

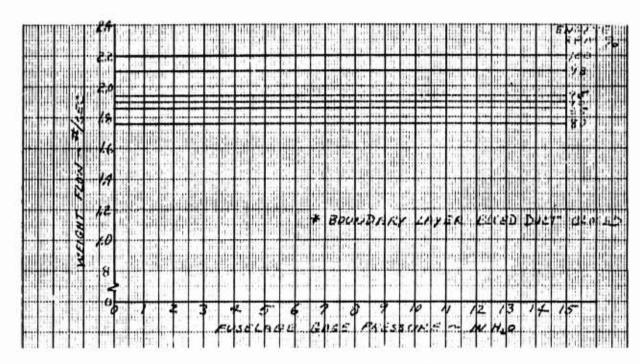


Figure 9.81 Cooling Air Weight Flow - Large Cooling Fan to Tailpipe Ejector Vs Fuselage Pressure and % RPM -Conventional Mode, Standard Day, Sea Level

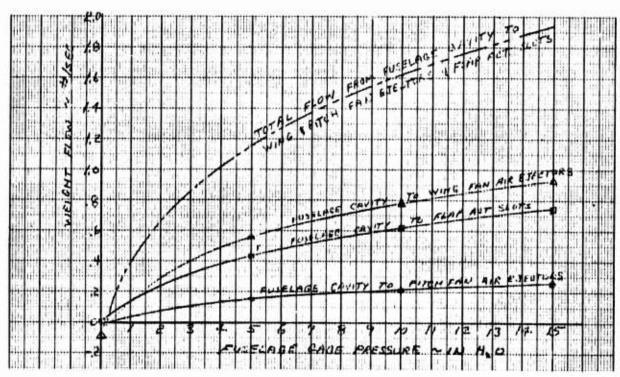


Figure 9.82 Cooling Air Weight Flow - Wing and Nose Fan Ejectors and Flap Actuator Slot to Outside Vs Fuselage Pressure and RPM - Conventional Mode, Standard Day, Sea Level

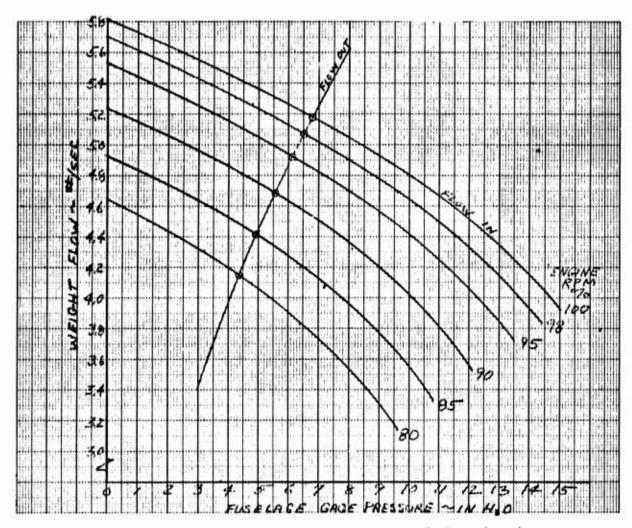


Figure 9.83 Cooling Air Weight Flow Balance of Flow Thru the Lower Fuselage Vs Fuselage Pressure and % RPM - Conventional Mode, Standard Day, Sea Level

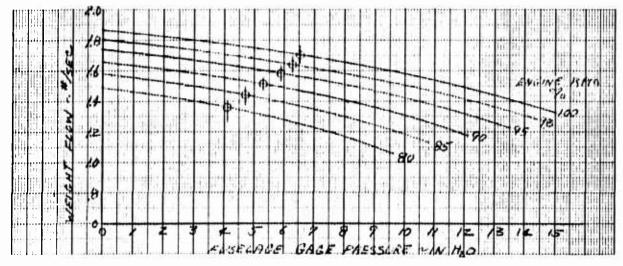


Figure 9.84 Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.2

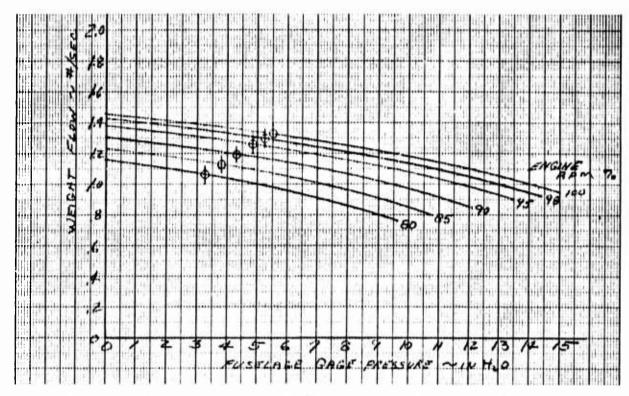


Figure 9.85 Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.4

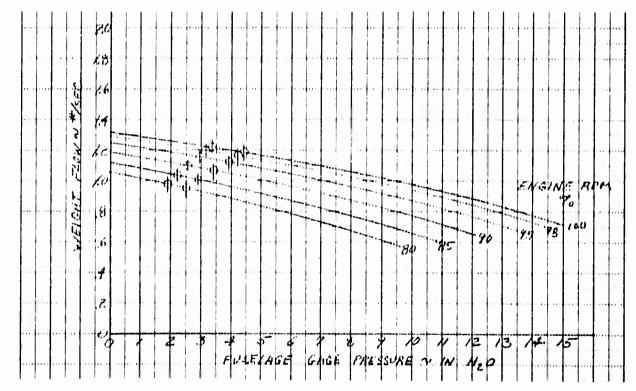


Figure 9.86 Cooling Air Weight Flow - Cockpit to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.6 and 0.8

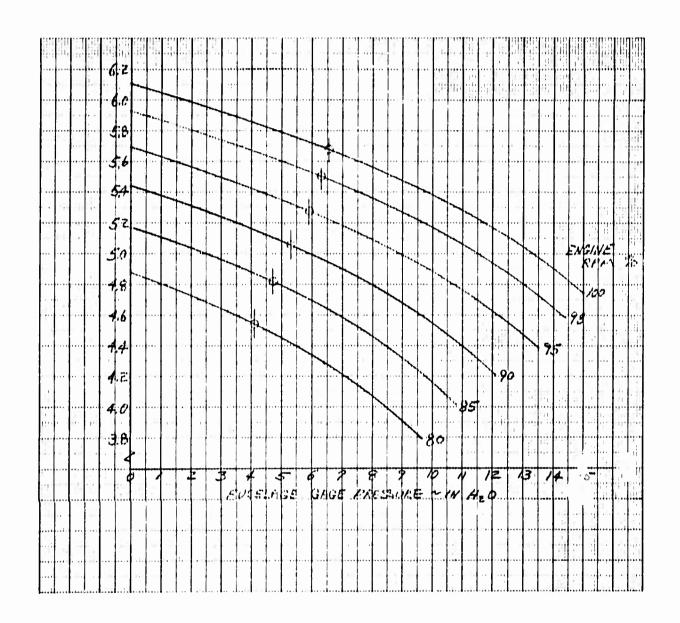


Figure 9.87 Cooling Air Weight Flow - Fuselage Ports to Cooling Fan Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.2

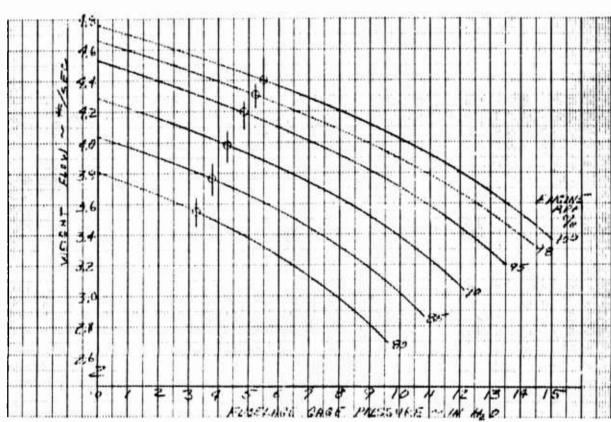


Figure 9.88 Cooling Air Weight Flow - Fuselage Ports to Cooling Fan
Compartment Vs Fuselage Pressure and % RPM - Conventional
Flight Mode, Standard Day, Sea Level, Mach No. = 0.4

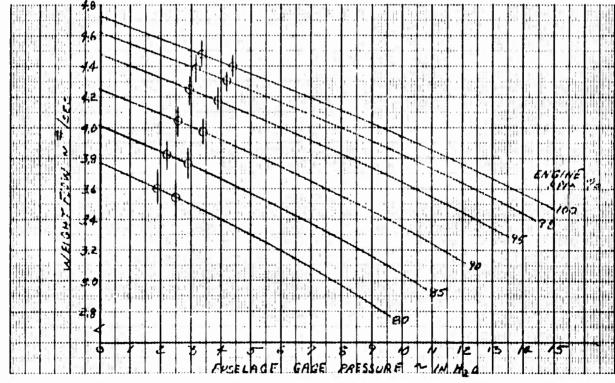


Figure 9.89 Cooling Air Weight Flow - Fuselage Ports to Cooling Fan Compartment Vs Fuselage Pressure % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.6 and 0.8

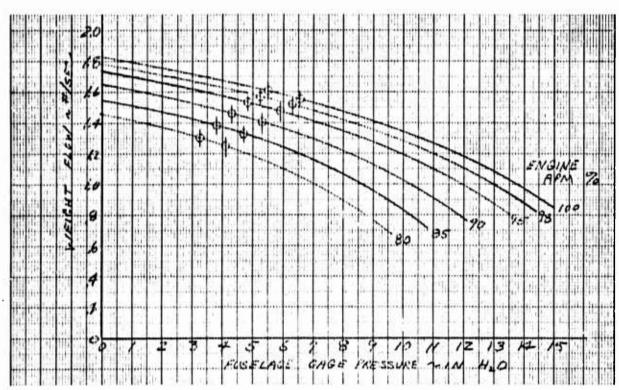


Figure 9.90 Cooling Air Weight Flow - Small Cooling Fan to Electronic Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.2 and 0.4

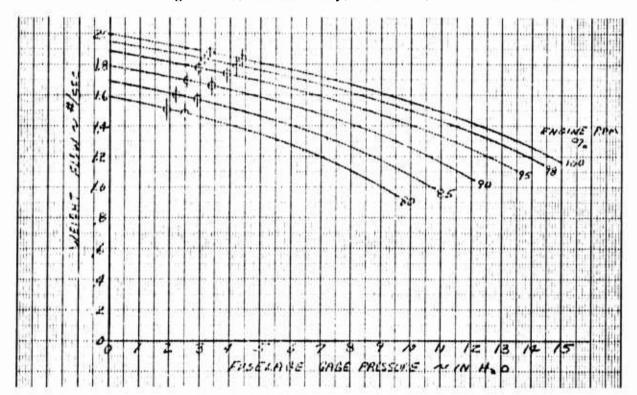


Figure 9.91 Cooling Air Weight Flow - Small Cooling Fan to Electronic Compartment Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.6 and 0.8

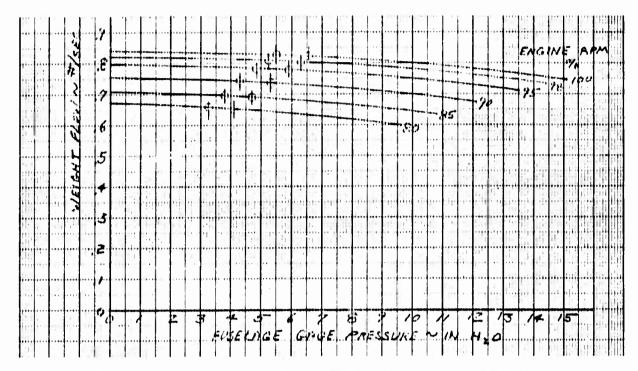


Figure 9.92 Cooling Air Weight Flow - Small Cooling Fan to Generators Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.2 and 0.4

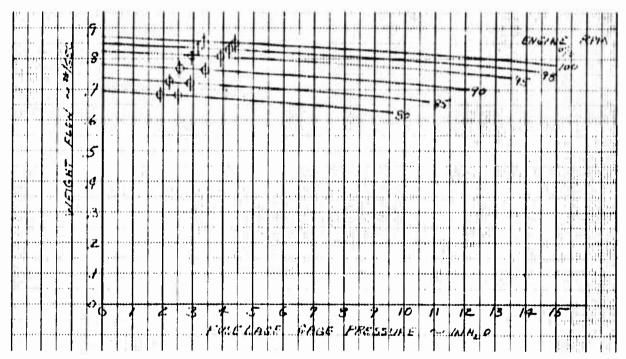


Figure 9.93 Cooling Air Weight Flow - Small Cooling Fan to Generators Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.6 and 0.8

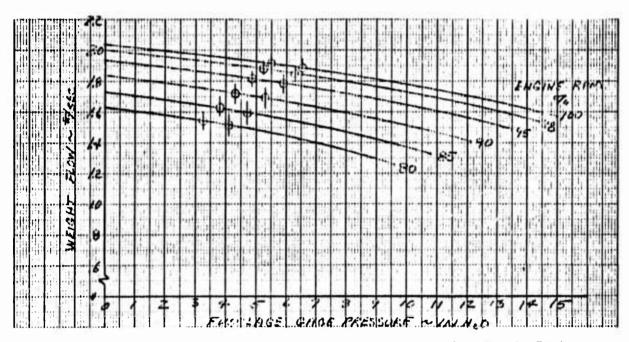


Figure 9.94 Cooling Air Weight Flow - L.H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Convention 1 Flight Mode, Standard Day, Sea Level, Mach No. = 0.2 and 0.4

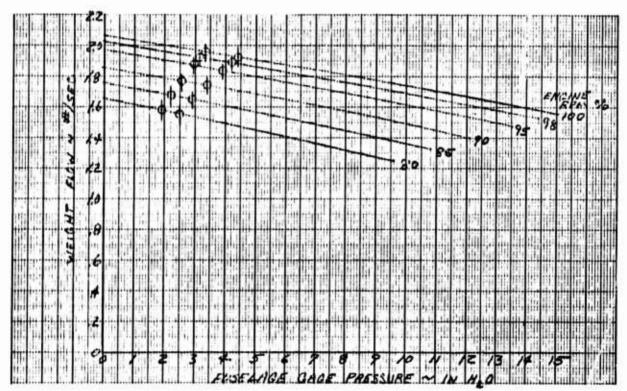


Figure 9.95 Cooling Air Weight Flow - L.H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.6 and 0.8

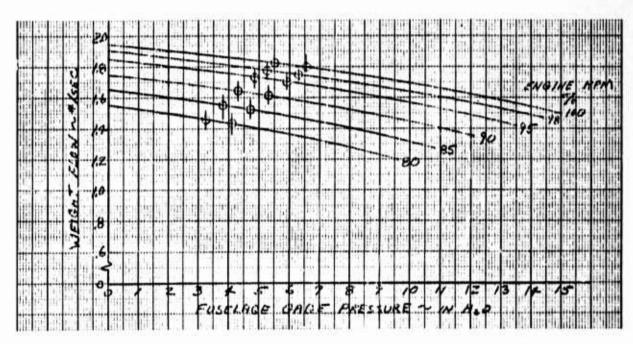


Figure 9.96 Cooling Air Weight Flow - R.H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.2 and 0.4

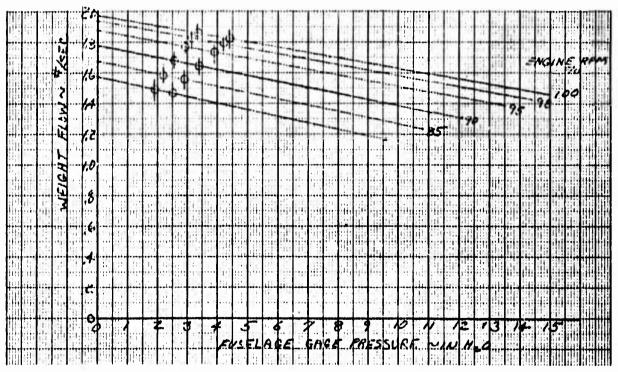


Figure 9.97 Cooling Air Weight Flow - R.H. Large Cooling Fan to Center Fuselage Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.6 and 0.8

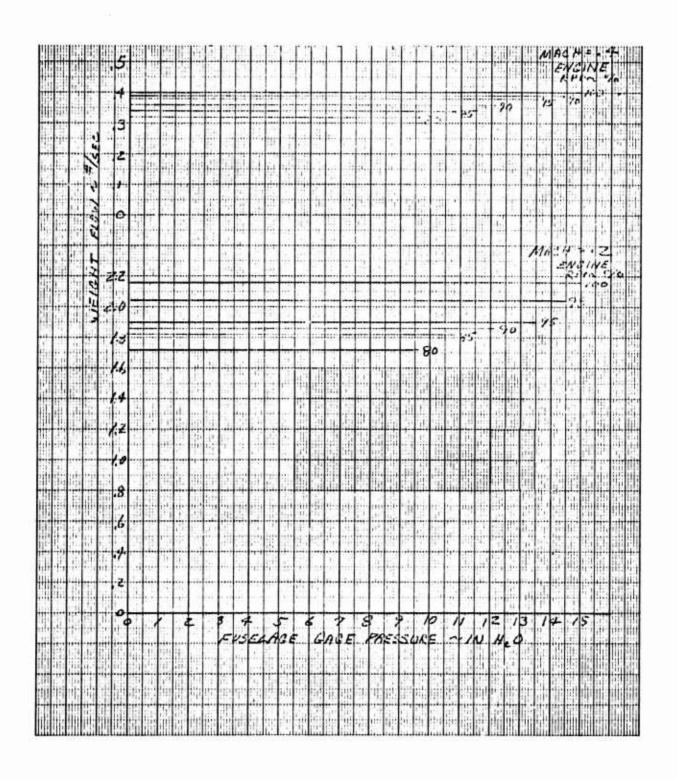


Figure 9.98 Cooling Air Weight Flow - Large Cooling Fans to Engine Bay Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.2 and 0.4

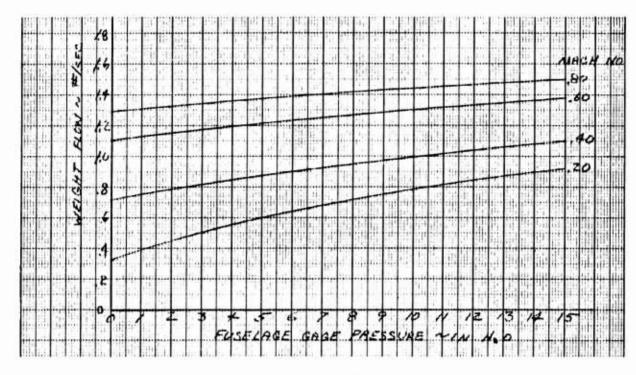


Figure 9.99 Cooling Air Weight Flow - Center Fuselage to Wing Fan Air Ejectors Vs Fuselage Pressure and Mach No. - Conventional Flight Mode, Standard Day, Sea Level

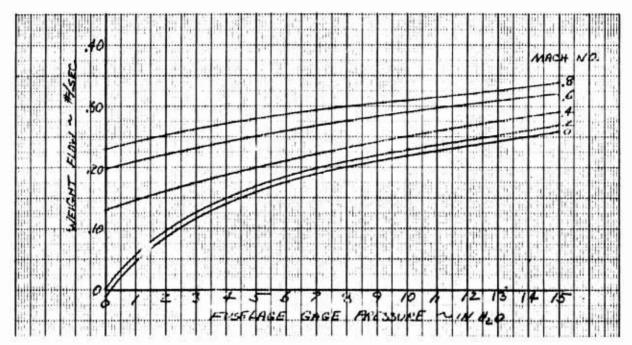


Figure 9. 100 Cooling Air Weight Flow - Center Fuselage to Nose Fan Air Ejectors Vs Fuselage Pressure and Mach No. - Conventional Flight Mode, Standard Day, Sea Level

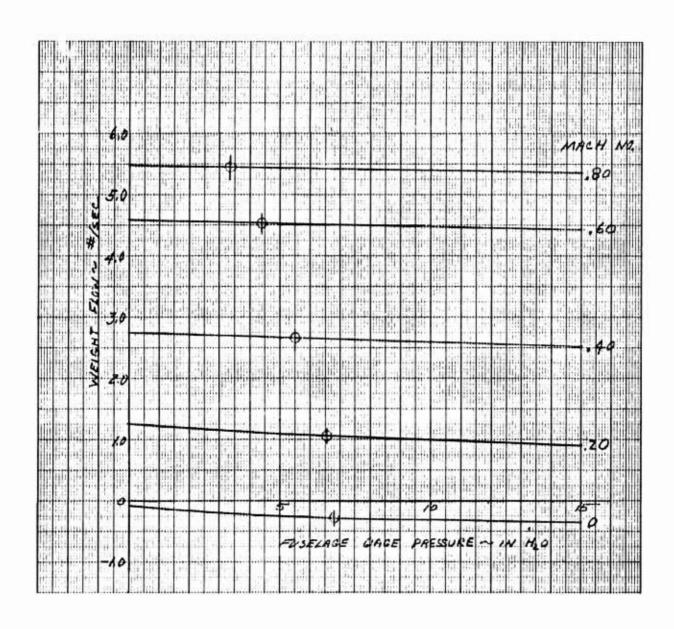


Figure 9. 101 Cooling Air Weight Flow - Outside to Nose Fan
Cavity Vs Fuselage Pressure and Mach No. Conventional Flight Mode, Standard Day, Sea Level

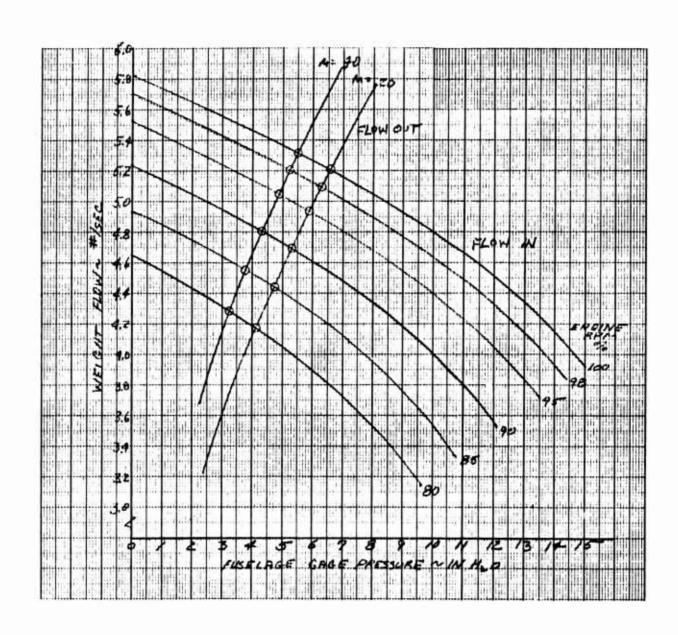


Figure 9.102 Cooling Air Weight Flow - Balance of Flow Into and Out of the Lower Fuselage Vs Fuselage Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.2 and 0.4

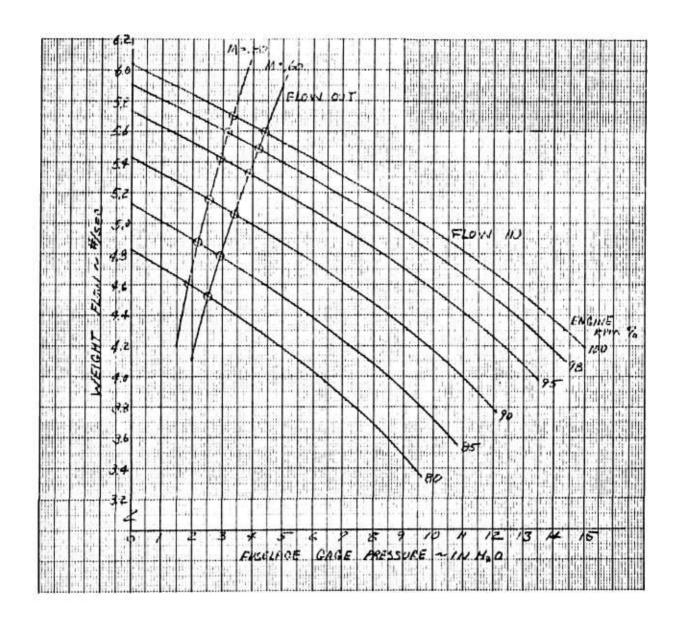


Figure 9.103 Cooling Air Weight Flow - Balance of Flow Into and Out of the Lower Fuscinge Vs Fuscinge Pressure and % RPM - Conventional Flight Mode, Standard Day, Sea Level, Mach No. = 0.6 and 0.8

9.4 THERMAL ANALYSIS

The structural boundaries, functions, and operation of the areas discussed in this section have been described in Section 3.0. The procedures used in this analysis were taken mainly from References 12, 14, and 15.

9.4.1 Cockpit Air Temperatures

The cockpit is ventilated by air drawn through gaps at the canopy closure. In the turbojet mode, cockpit air is made up largely of boundary layer air. In lift fan mode, it is made up largely of locally induced environmental air. As a result, cockpit inlet air temperatures are affected by climatic conditions (day and altitude), by aircraft flight speed and/or ingestion effects. Short of some form of air conditioning, there is no practical way of reducing cockpit air temperatures in the conventional mode. In the fan mode, relocation of cockpit air inlet may permit cockpit inlet air to approach ambient air temperatures.

Cockpit heat loads include inputs from the following: solar irradiation, crew and equipment aerodynamic heating, hot gas ingestion, heat transfer from walls, floor, and bulkheads.

Figures 7.78 and 7.79 present estimated cockpit temperatures vs aircraft speed altitude and day for conventional operation. Estimated temperatures were calculated as follows:

$$\Delta t_{AH} = \Delta T_{AH} = \frac{k-1}{2} r M^2 T_{AMB}$$

where for r = 0.89 and k = 1.4

$$\Delta t_{AH} = 0.178 M^2 T_{AMB}$$

Additional Heating

Solar heat constant = 270 Btu/hr ft²

Projected area of the canopy = 20 ft²

 Δt_{SC} = Temperature rise due to solar energy and the crew

$$\Delta t_{SC} = \frac{q_{SOLAR} + q_{CREW}}{W_a C_{p_a}^{3600}} = \frac{6.9}{W_a}$$

W_a = Cooling Air Flow rate; lb/sec

Total Temperature Rise, Δt_{sc}

$$\Delta t_{T} = \Delta t_{AH} + \Delta t_{SC}$$

$$t_c = t_{AMB} + \Delta T_t$$

Example:

Conditions: Hot Day, Sea Level, Mach = .6

$$T_{AMB} = 103^{\circ} F + 460 = 563^{\circ} R$$

$$W_{g} = 1.10 \text{ lb/sec}$$

$$\Delta t_{AH} = \left(\frac{1.4-1}{2}\right) (.89) (.6)^2 (103 + 460) = 36$$

$$\Delta T_{t} = \Delta T_{AH} + \Delta T_{SC} = 36 + 6.3 = 42.3$$

$$t_c = t_{amb} + \Delta T_T = 103 + 42.3 = 145.3^{\circ} F$$

9.4.2 Cooling Fan Compartment Inlet Port Air Temperature - Turbojet Mode

The free stream air passing the inlet is sucked into the cooling fan compartment when the high speed stream is brought to near stagnation condition:

$$\Delta t_{AH} = 0.178 \text{ M}^2 \text{ T}_{AMB}$$

and for hot day sea level conditions at M = 0.6

$$\Delta t_{AH} = 36^{\circ} F \text{ as above}$$

9.4.3 Cooling Fan Compartment Air Temperature

The cooling air enters the cooling fan compartment from the cockpit, fuselage ports and generators. Assuming complete mixing of the air, the resultant temperature is a function of the weight flow and temperature of each flow. A plot of cooling fan compartment temperature vs aircraft speed is presented in Figure 7.82.

Setting C_{p_a} equal for all flows

$$W_{G} t_{G} + W_{c} t_{c} + W_{p} t_{p} = (W_{G} + W_{c} + W_{p}) t_{m}$$

since $t_G = f(t_m)$

$$(t_G^{-t}m) = \frac{q_G}{W_G^C_{p_a}}$$

$$W_G t_G = W_G t_m + \frac{q_G}{C_{p_a}}$$

$$\frac{\mathbf{q}_{\mathbf{G}}}{\mathbf{C}_{\mathbf{p}_{\mathbf{a}}}} + \mathbf{W}_{\mathbf{c}} \mathbf{t}_{\mathbf{c}} + \mathbf{W}_{\mathbf{p}} \mathbf{t}_{\mathbf{p}} = (\mathbf{W}_{\mathbf{c}} + \mathbf{W}_{\mathbf{p}}) \mathbf{t}_{\mathbf{m}}$$

$$t_{m} = \frac{q_{G/Cp_{a}} + W_{c}t_{c} + W_{p}t_{p}}{W_{c} + W_{p}}$$

Example:

Hot Day, sea level,
$$M = .6$$

Cockpit Air

See cockpit air temperature analysis

$$t_c = 145.3^{\circ} F$$
 $W_c = 1.10$

Fuselage Port Air

See fuselage port inlet air analysis

$$t_p = 139^{\circ} F$$
 $W_p = 3.62$

Generator Air Temperature

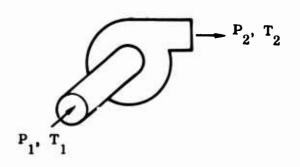
See Generator air temperature analysis

$$t_{m} = \frac{q_{G}/c_{p} + W_{c}t_{c} + W_{p}t_{p}}{W_{c} + W_{p}} = \frac{21.04 + 1.10 (145.3) + 139 (3.62)}{4.72}$$

$$t_{m} = \frac{21.04 + 159.83 + 503.14}{4.72} = 144.9^{\circ}F$$

9.4.4 Temperature Rise Across the Cooling Fans

The minimum temperature rise across the fan is approximated by assuming a reversible adiabatic compression process. Thus



$$T_2 = \left(\frac{\frac{P_2}{P_1}}{P_1}\right)^k T_m \cdot R = \left(\frac{\frac{P_2}{P_1}}{P_1}\right)^{285} T_m$$

This is a minimum value.

$$T_2 = \left(\frac{15.84}{14.69}\right)^{.285} T_m = 1.0216 T_m$$

$$T_2 - T_m = .0216 T_m$$

Large Cooling Fan

$$T_2 = \left(\frac{16.20}{14.69}\right)^{.285} T_m = 1.0283 T_m$$
.
 $T_2 - T_m = .0283 T_m$

Example:

Hot Day, Sea level, Mach = .6

Small Cooling Fan

$$\Delta t = \Delta T = .0216 T_{m} = 13.03 \text{ since } T_{m} = t_{m} + 460$$

$$T_{m} = 460 + 144.9 = 604.9^{\circ} R$$

$$t_{2} = t_{m} + \Delta t = 144.9 + 13.0 = 157.9$$

Large Cooling Fan

$$\Delta t = .0283 T_{m} = 17.08$$

$$t_{2} = 144.9 + 17.1 = 162.0$$

A plot of the cooling fan exhaust temperature vs aircraft speed is presented in Figure 7.81.

9.4.5 Generator Air Temperature

A constant power of 165 amps at 30 volts is available between 80% and 100% engine RPM per generator.

Generator Efficiency = 65%.

$$165 \times 30 = 4.95 \text{ KW/GEN} = 4.69 \text{ Btu/sec.} = q_{G}$$

$$q_G = 4.69 \text{ Btu/sec.}$$

$$q_{Gi}$$
 .65 = 4.69 Btu/sec.

$$q_{Gi} = 7.22 \text{ Btu/sec.}$$

$$\Delta q_G$$
 = Heat rejected = $(q_{Gi} - q_G)$ = 7.22 - 4.69 = 2.53 Btu/sec

$$\Delta q_G = W_G^C_{p_a} \Delta t$$

$$\Delta t_{G} = \frac{2.53}{W_{G} C_{p_{A}}} = \frac{10.52}{W_{G}}$$

Example:

Hot Day, sea level, M = .6

$$W_G = .70/2 = .35 lbs. air/generator$$

$$\Delta t_G = \frac{10.52}{.35} = 30 \text{ deg.}$$

A plot of generator discharge temperature vs aircraft speed is presented in Figure 7.80.

9.4.6 Temperature Rise Across the Hydraulic Oil Cooler

From: Stewart-Warner Corporation 10-12-62
Performance - 8407 C Oil Cooler

The effectiveness factor (E) is given by the relationship

$$\frac{T_{OIL\ IN} - T_{OIL\ OUT}}{T_{OIL\ IN} - T_{AIR\ IN}} = E$$

For E = 0.9,
$$\Delta T_{OIL}$$
 = $T_{OIL IN}$ - $T_{OIL OUT}$
= .90 ($T_{OIL IN}$ - $T_{AIR IN}$)

$$q_{AIR} = q_{OIL}$$

$$W_a^C_{p_A}^{(T_{AIR\ OUT} - T_{AIR\ IN})} = W_o^C_{p_o}^{(T_{OIL\ IN} - T_{OIL\ OUT})}$$

$$\Delta T_{A} = \frac{W_{o}}{W_{a}} \frac{C_{p_{o}}}{C_{p_{a}}} (.90) (T_{OIL\ IN} - T_{AIR\ IN})$$

$$C_{p_0} = .455$$
 $C_{p_a} = .24$

$$\Delta T_A = 1.706 \frac{W_o}{W_A} (T_{OIL\ IN} - T_{AIR\ IN})$$

Example:

Hot Day, sea level, Mach = .6

$$W_0 = 1 \text{ gal./min} = 7.15 \text{ lb/min}$$

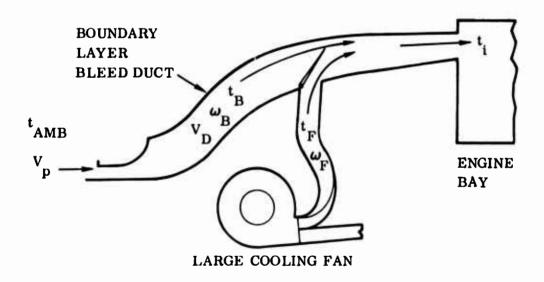
$$W_{p} = .88 \text{ lb/sec/Hyd Oil Cooler}$$

T_{AIR IN} = 155° F from Temp Rise Across the Small Fans

$\Delta T_A = 1.706 \left(\frac{\cdot}{\cdot}\right)$	$\left(\frac{119}{88}\right)$ (T _{OIL IN}	v - 155) = .230 (T _{OIL})	_{IN} - 155)
Set TOIL IN	ΔT_{A}	TAIR OUT	
200	10	165	
250	21	176	
300	33	188	

9.4.7 Engine Bay Inlet Air Temperature

The temperature of the engine bay inlet at the top, inboard, forward corner is a result of the mixed air from the boundary layer bleed duct and the large cooling fan as shown in the schematic below. A plot of engine bay inlet time vs aircraft speed is presented in Figure 7.87.



$$t_i = \frac{\sum_{wt} wt}{\sum_{w}} = \frac{W_B t_B + W_F t_F}{W_B + W_F}$$

For Large Cooling Fan exhaust temperature, t_F, see Section 9.4.4.

Boundary layer bleed duct, t_B

$$t_{B} = t_{AMB} + \Delta t_{AH} = t_{AMB} + 0.178M^{2} T_{AMB}$$

Example:

Conditions: Hot Day, Sea Level, Mach = $0.6 \text{ at } \text{M} = 0.6 \text{ W}_{\text{F}} = 0$

$$t_{i} = t_{B}$$
 $t_{AMB} = 103^{\circ} F$
 $t_{B} = 103 + 0.178 (.6)^{2} (563)$
 $= 103 + 36 = 139^{\circ} F$

9.4.8 Center Fuselage Air Temperature Analysis - Lift Fan Mode

During VTOL mode, hot gases flow through the fan supply ducts and leave the wing fans on the inboard quadrants. The hot gases leaving the fans impinge on the lower section of the center fuselage.

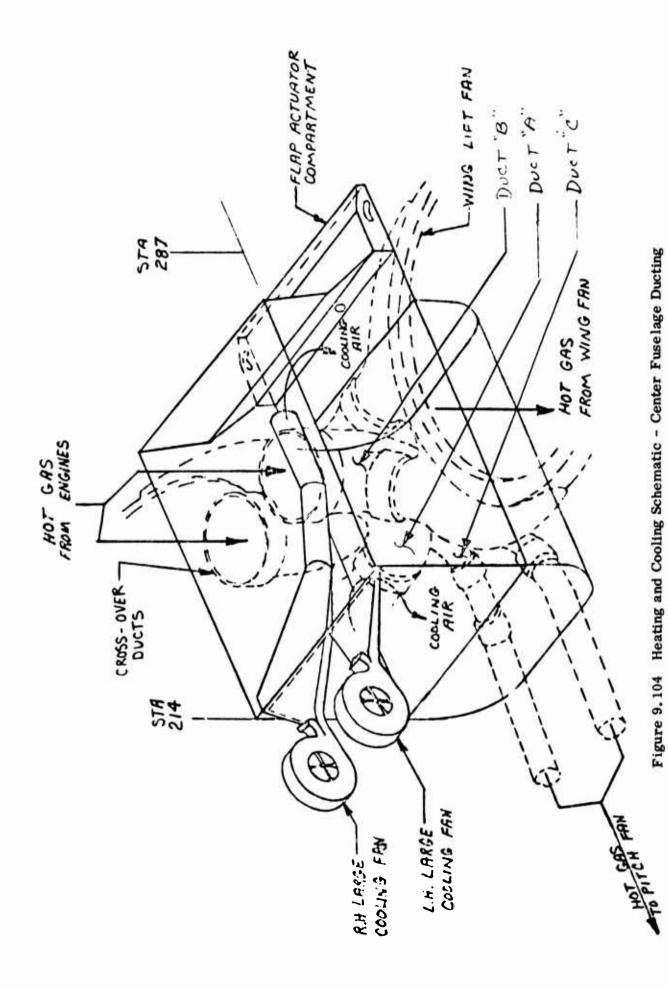
The center fuselage air will be heated by the following:

- 1. Heat Transfer from the supply ducts to the air.
- 2. Heat Transfer from the canoe to the air.
- 3. Mass transfer of the fuselage air recirculating between the duct and shroud.
- 4. Mass transfer of wing fan hot exhaust gas into the fuselage.
- 5. Mass transfer of hot gases from the supply duct joints.

Example:

Standard Day, Sea Level

100% RPM, Lift Fan Mode, Static Condition



392

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Assume complete mixing of all gases.

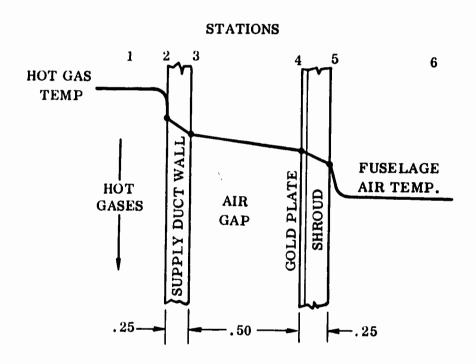
1. Heat Transfer from the Supply Ducts to the Air

Duct Area* A = 23.6 ft²/Both engines

 $B = 9.0 \text{ ft}^2/\text{Both engines}$

 $C = 21.0 \text{ ft}^2/\text{Both engines}$

*See Figure 9.104.



Overall Heat Transfer

$$\frac{\mathbf{q}}{\mathbf{A}} = \frac{\mathbf{t_1} - \mathbf{t_6}}{\frac{1}{\mathbf{h_{1-2}}} + \frac{\mathbf{X_{2-3}}}{\mathbf{k_{2-3}}} + \frac{1}{\mathbf{h_{3-4}}} + \frac{\mathbf{X_{4-5}}}{\mathbf{k_{4-5}}} + \frac{1}{\mathbf{h_{5-6}}}}$$

$$k_{4-5} = 2.1 \frac{BTU IN}{HR ft^2 \circ F} \frac{X_{4-5}}{k_{4-5}} = \frac{.025}{2.1} = 1.19 \times 10^{-2}$$

$$k_{2-3} = 135 \frac{BTU IN}{HR PT^2 \circ F} \frac{X_{2-3}}{k_{2-3}} = \frac{.025}{135} = 1.84 \times 10^{-4}$$

$$\frac{h}{\rho C_{p} V_{m}} = \frac{.0384 \left(\frac{R_{e_{d}}}{e_{d}}\right)^{-\frac{1}{4}}}{1 + \left(1.5 P_{r}^{-1/6}\right) \left(\frac{R_{e_{d}}}{e_{d}}\right)^{-1/8} \left(P_{r}^{-1}\right)}$$
 Reference 16

$$\rho = .047 \text{ lb/ft}^3$$

$$C_{p} = .27 BTU/lb ° F$$

$$P_{r} = .70$$

$$(P_r)^{-1/6} = 1.061$$

$$V_{\rm m} = 645 \, \rm ft/sec$$

$$R_{ed} = \frac{D\rho V}{\mu} = \frac{.92 (.047) (645)}{2.68 \times 10^{-5}} = 1.037 \times 10^{6}$$

$$\binom{R_{e_d}}{}^{-1/4} = .0313$$

$$\left({\rm Re}_{\rm d} \right)^{-1/8} = .177$$

$$\rho C_{p}V_{m} = (.047) (.27) (645) = 8.18$$

$$h_{1-2} = \frac{8.18 (3.84) (3.13)}{1+1.59 (.177) (-.3)} (3.6 \times 10^3 \frac{SEC}{HR}) = 38.5$$

$$\frac{1}{h_{1-2}} = \frac{1}{38.5} = .026$$

$$h_{3-4} = h_{c_{3-4}} + h_{r_{3-4}}$$

h = Convective Heat Transfer Coeff.

 $h_r = Radiation Heat Transfer Coeff.$

$$h_{r_{3-4}} = \sigma F_{A_{3-4}} \frac{\left[T_{D}^{4} - T_{S}^{4}\right]}{T_{D} - T_{S}} \text{ where } \sigma = 1730 \times 10^{-12}$$

$$F_{A_{3-4}} = \frac{1}{\frac{1}{\epsilon_3} + \frac{A_3}{A_4} \left(\frac{1}{\epsilon_4} - 1\right)} = \frac{1}{\frac{1}{\cdot 9} + \cdot 91 \left(\frac{1}{\cdot 1} - 1\right)} = .107$$

$$h_{r_{3-4}} = 185 \frac{\left| (T_{D/1000})^4 - (T_{S/1000})^4 \right|}{T_D - T_S}$$

$$Set T_{D} = 1610 \, ^{\circ}R$$

$$T_S = 1460 \, ^{\circ} R$$

$$h_{r_{3-4}} = 185 \frac{(6.72 - 4.54)}{150} - 2.69$$

$$LOG \frac{hc_{3-4}}{kc_{3-4}} = \Phi (N_{Gr})$$

$$N_{Gr} = \frac{\gamma^2 \beta g}{\mu} D_1^3 (t_1 - t_2)$$

$$\beta = \frac{1}{T^{\circ}R} = 7.8 \times 10^{-4} \circ R^{-1}$$

$$\mu^2 = 5.86 \times 10^{-10} \, \text{lb}^2/\text{sec}^2 \, \text{ft}^2$$

$$\gamma^2 = 3.6 \times 10^{-3} \text{ lb}^2/\text{ft}^2$$

$$g = 32.2 \text{ ft/sec}^2$$

$$D_1^3 = 7.8 \times 10^{-1} \text{ ft}^3$$

$$\Delta T = 100$$

$$N_{Gr} = \frac{(3.6 \times 10^{-3}) (7.8 \times 10^{-4}) (3.22 \times 10) (7.8 \times 10^{-1})}{5.86} \times 10^{12}$$

$$= 1.20 \times 10^{7}$$

$$LOG \frac{h_{c3-4}}{k_{c3-4}} = .04$$

$$\frac{h_{c3-4}}{k_{c3-4}} = 1.10 \qquad h_{c3-4} = 1.10 (.36) = .8$$

$$\frac{1}{h_{3-4}} = \frac{1}{2.69 + .8} = .286$$

$$h_{5-6} = h_{c_{5-6}} + h_{r_{5-6}}$$

$$h_{c_{5-6}} = .27 \left(\frac{\Delta T}{D}\right)^{1/4} = .27 (4.73) = 1.28$$

$$h_{r_{5-6}} = \sigma F_{A_{5-6}} \frac{\left[\left(T_{s_{0/1000}} \right)^{4} - \left(T_{o_{/1000}} \right)^{4} \right]}{T_{s_{0}} - T_{o}}$$

$$F_{A_{5-6}} = \frac{1}{\frac{1}{\epsilon_5} + \frac{1}{\epsilon_6} - 1} = \frac{1}{\frac{1}{\cdot 36} + \frac{1}{\cdot 8} - 1} = .33$$

$$h_{r_{5-6}} = \frac{571 (2.518 - .254)}{550} = 2.35$$

$$h_{5-6} = 1.28 + 2.35 = 3.63$$

$$\frac{1}{h_{5-6}} = .275$$

$$\frac{q}{A} = \frac{t_1 - t_6}{.026 + .0018 + .286 + .012 + .275} = \frac{t_1 - t_6}{.599}$$

$$t_1 - t_6 = 1150$$

$$A = 53.6 \text{ ft}^2$$

$$q = 102, 912 BTU/HR = 28.6 BTU/SEC$$

$$\Delta t = \frac{q}{W_a C_{p_a}} = \frac{28.6}{3.78 (.24)} = 32^{\circ} F$$

where $W_a = 3.78$ from Figures 11.68 and 11.69, at Fuselage Press = $5''H_9O$

2. Heat Transfer from Canoe Panel to Air

Area of Canoe = 36 ft^2

$$h = .19 (\Delta T)^{1/3} = 1.11$$

$$q = hA\Delta T = 8000 Btu/HR = 2.2 Btu/SEC$$

$$\Delta T = \frac{2.2}{3.78(.24)} = 3^{\circ}$$

3. Heat Transfer to Fuselage Air by Recirculation Between Ducts and Shroud

(q to fuse air) =
$$(W_{recirculation}) C_{p_a} \Delta t_a$$

$$\Delta T = 900-137 = 763$$

$$\Delta t_a = \frac{(Q \text{ to Fuse Air})}{W_{\text{Fuse Air}} C_{p_a}}$$

$$q = .1 (.24) (763) = 18.3 Btu/Sec$$

$$\Delta t_{FusAir} = \frac{18.3}{3.78(.24)} = 20^{\circ}$$

See Figure 9.105

4. Mass Transfer of Hot Wing Fan Exhaust Air Into The Center Fuselage

$$t_{m} = \frac{W_{F} t_{F} + W_{H} t_{H}}{W_{F} + W_{H}}$$
 F = Fuse. Air
H = Hot Gas

at
$$T_F = 100^{\circ} F$$
 $W_H = .3$

$$W_{F} = 3.78$$
 $t_{H} = 300$

$$t_{\rm m} = 114^{\circ}$$

$$\Delta t = 14^{\circ}$$

5. Duct Joint Leakage

The duct joint leakage rate cannot be predicted, therefore the fuselage air temperature must be plotted against duct joint leakage in % engine hot gas flow.

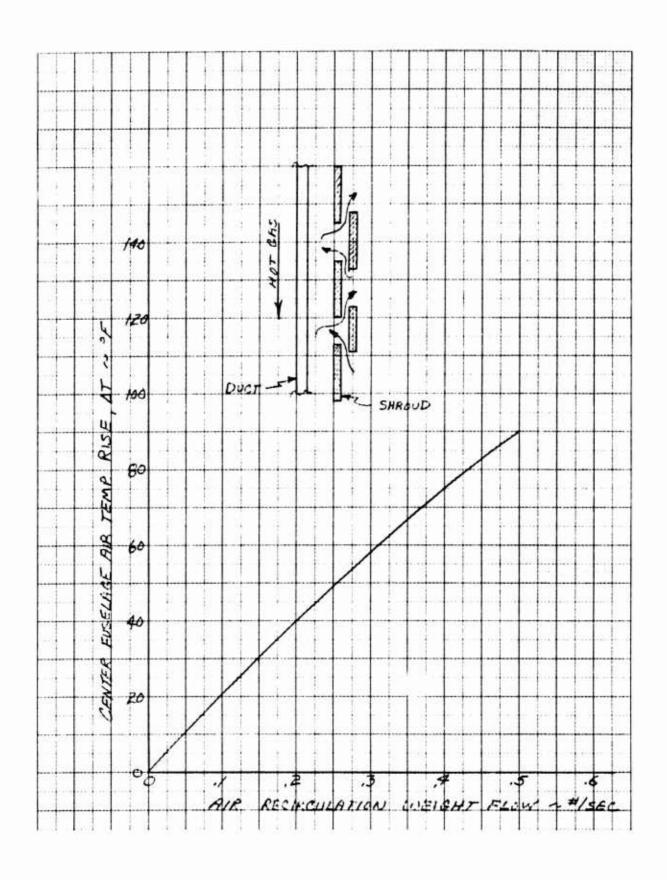


Figure 9. 105 Center Fuselage Air Temperature Rise Vs Recirculation of Fuselage Air Between Supply Duct and Shroud - Fan Mode

The temperature rise is a function of the temperature and weight flow of the hot gas leakage.

$$t_{m} = \frac{W_{F}t_{F} + W_{L}t_{L}}{W_{F} + W_{L}}$$

See Figure 7-77 for results.

9.4.9 Lift Fan Cavity Air Temperature - Turbojet Mode

Wing Fan

The wing cavity temperature is a function of the mixing of cooling air and hot diverter valve leakage. The cool air comes from the fuselage and pitch fan cavity.

$$t_{M} = \frac{\sum_{wt}}{\sum_{w}} = \frac{w_{p}t_{p} + w_{F}t_{F} + w_{H}t_{H}}{w_{p} + w_{F} + w_{H}}$$

Example:

Standard Day, sea level, 100% RPM

M	W _H	t H ° F	W t H H	w ** p lb/sec	t p ° F	Wtpp
0	. 304	1240	377	0	60	0
.1	. 354	1241	439	.28	60	16.8
. 2	. 361	1244	449	. 63	60	37.8
. 3	. 370	1247	461	1.02	61	62.2
.4	. 381	1247	475	1.45	62	89.9
.5	. 396	1248	494	1.91	63	120
. 6	.412	1247	514	2.39	64	153
.7	. 435	1239	539	2.75	66	182
.8	. 428	1230	526	2.87	68	195

^{*} $W_{H} = 0.8\%$ of engine air flow at the diverter valve inlet.

^{**} $W_{\rm p}$ = 0.5 sum of flow rates read from Figures 7.60 and 7.63.

М	w _F *	t °F	$\mathbf{w_F^t_F}$	$\sum_{\mathbf{w}}$	\sum_{Wt}	t _m
0	. 32	90	28.8	. 624	405.8	650
. 1	. 32	90	28.8	. 954	484.8	508
. 2	. 33	90	29.7	1.321	516.5	391
. 3	. 37	90	33.3	1.760	556.5	316
. 4	. 45	90	40.5	2.281	605.4	265
. 5	. 54	90	46.6	2.846	660.6	232
. 6	. 60	90	54.0	3.402	721.0	312
. 7	. 64	90	57.6	3.825	778.6	203
. 8	. 67	90	60.3	3.968	781.3	197

^{*} $W_F = 0.5$ value read from Figure 7.5%.

Calculate and plot $t_{\rm m}$ vs Mach No. for various RPM's and terminate each RPM curve at stable flight condition.

Nose Fan

Calculate the pitch fan cavity temperature in the same manner as the wing fan.

$$t_{m} = \frac{W_{o o}^{t} + W_{F}^{t} + W_{H}^{t}}{W_{o}^{t} + W_{F}^{t} + W_{H}^{t}}$$

9.4.10 Wing Fan Ejector Air Temperature During Forward Fan Flight

The static pressure at the wing fan ejectors will vary with respect to the cross flow at the fan during forward flight as shown by the plots of $(P_s-P_a)/q^s$ vs T_c^s and β_v in Figures 9.106 and 9.107 (Reference 17)

where

$$T_c^s = \frac{T_{ooo}/A_F}{T_{ooo}/A_F + q_o} = Slip Stream Thrust Coefficient$$

and

$$q^{S} = \frac{T_{OOO}}{A_{F}} + q_{O} = Slip Stream Dynamic Pressure$$

At trimmed flight for any velocity and β_V value, $T_C{}^S$ and q^S are given, therefore $P_S{}^-P_a$ at the ejector is known. With a system pressure differential known, the weight flow of cooling air is obtained from the system performance.

During operation in the fan mode, a scroll leakage of .2% W_g may occur into the cooling air. The temperature of the mixed flow is a function of the weight flow and temperature of the two flows.

$$t_{\mathbf{M}} = \frac{\sum_{\mathbf{W}t} \mathbf{t}}{\sum_{\mathbf{W}} \mathbf{t}} = \frac{\mathbf{W}_{\mathbf{c}}\mathbf{t}_{\mathbf{c}} + \mathbf{W}_{\mathbf{H}}\mathbf{t}_{\mathbf{H}}}{\mathbf{W}_{\mathbf{c}} + \mathbf{W}_{\mathbf{H}}}$$

Example:

Hot Day, 2500 feet

$$\alpha = 0$$

$$\beta_s = 6^\circ$$

C.G. at Sta. 246

GW = 9200 lbs.

Trimmed Flight Conditions:

v _p	${}^{\beta}_{\mathbf{v}}$	T ^s	q ⁸
Knots	Deg.	c	#/ft ²
0	0	1.0	210
35	10	. 984	224
54	20	. 965	243
70	30	. 942	247
86	40	. 922	278
95	45	. 914	308

Scroll leakage

 $W_{gas} = 38.5 lb./sec.$

13% To Pitch Fan = 5 lb./sec.

 W_{H} To Wing = 33.5 lb./sec.

 W_{H} / Side of Wing Fan = 16.7 lb./sec.

Leakage at .2% = 16.7 (.002) = .033 lb./sec. at 1140° F

 $W_H^{t}_H = 37.6$

 $Set t_{c} = 150^{\circ} F$

AFT AIR EJECTOR

_	V p Knots	W _c	W _c t	\sum_{wt}	$\sum_{\mathbf{w}}$	t m ° F
	0	.310	46.5	84.1	. 343	245
	20	.298	44.7	82.3	. 331	249
	40	. 273	40.9	78.5	. 306	256
	60	.239	35.8	73.4	. 272	269
	80	.200	30.0	67.6	. 233	290
	95	.180	27.0	64.6	. 213	303

FORWARD AIR EJECTOR

V p Knots	W _c	W _c t _c	\sum_{wt}	\sum_{w}	t M ° F
0	.280	42.0	79.6	. 313	254
20	.290	43.5	81.1	. 323	251
40	. 304	45.6	83.2	. 337	246
60	. 326	48.9	86.2	. 359	240
80	. 346	51.9	89.5	. 379	236
95	.360	54.0	91.6	. 393	233

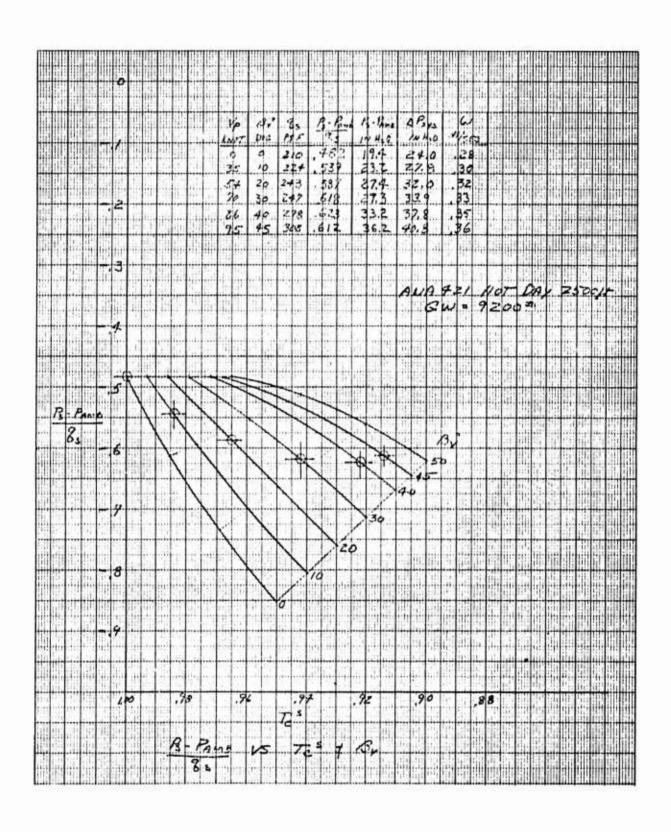


Figure 9.106 Wing Fan Forward Air Ejector - $T_c^s V_s \frac{P_s - P_s}{q_s}$

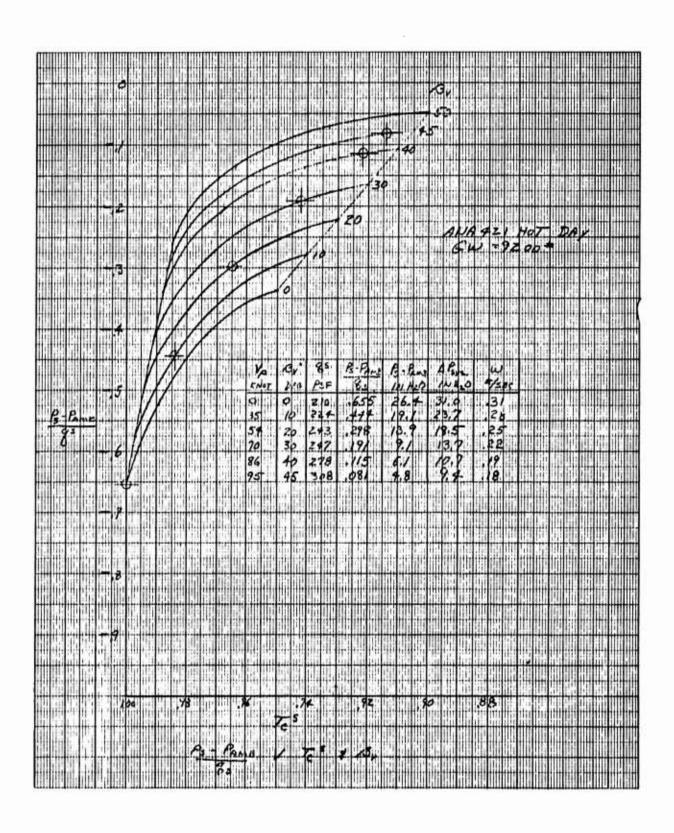


Figure 9.107 Wing Fan Aft Air Ejector - $T_c^s V_s = \frac{P_s - P_s}{q_s}$

9.4.11 Engine Bay Heat Transfer Analysis

Each engine is enclosed by a bay from the turbine casing to the tailpipe (see Figure 9.108). The engine has three distinct components in the engine bay, the turbine casing, diverter valve, and bellows. The turbine casing has a step temperature drop at the turbine blades, therefore the turbine casing may be analyzed as two units.

Turbine Casting, Section 1 and 2

The heat balance schematic is shown in Figure 9.109. It is assumed that no heat flow occurs through the forward bay enclosure.

Basic Heat Transfer Equations

$$q_{i} = U_{i}A_{T} (T_{G}^{-}T_{T}^{-})$$

$$q_{C_{T-A}} = h_{T}A_{T} (T_{T}^{-}T_{A}^{-})$$

$$q_{R_{P}} = \sigma F_{P}A_{f_{P}} (T_{T}^{-}T_{P_{i}}^{-})$$

$$q_{R_{W}} = \sigma F_{W}A_{f_{W}} (T_{T}^{-}T_{P_{i}}^{-})$$

$$q_{R_{F}} = \sigma F_{F}A_{f_{F}} (T_{T}^{-}T_{F}^{-})$$

$$q_{R_{X}} = \sigma F_{X}A_{T} (T_{T}^{-}T_{X}^{-})$$

$$q_{C_{W-A}} = h_{W}A_{W} (T_{W}^{-}T_{A}^{-})$$

$$q_{C_{P-A}} = h_{P}A_{P} (T_{F}^{-}T_{A}^{-})$$

$$q_{C_{F-A}} = h_{F}A_{F} (T_{F}^{-}T_{A}^{-})$$

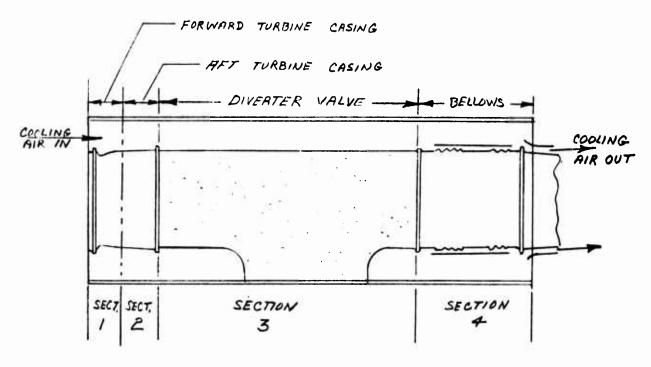


Figure 9. 108 Heating and Cooling Schematic - Engine Bay

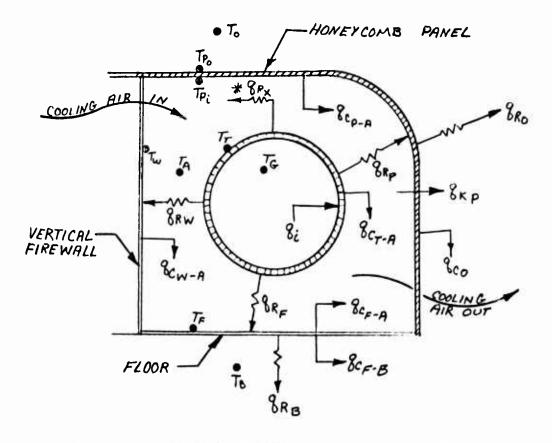


Figure 9. 109 Forward and Aft Turbine Casing Heat Flow Schematic

$$q_{K_{P}} = \frac{K_{P}}{1} A_{P} \left(T_{P_{O}}^{-1} - T_{P_{i}}^{-1} \right)$$

$$q_{R_{B}} = \sigma F_{B} A_{F} \left(T_{F}^{-1} - T_{B}^{-1} \right)$$

$$q_{C_{F-B}} = h_{B} A_{F} \left(T_{F}^{-1} - T_{B}^{-1} \right)$$

$$q_{C_{O}} = h_{O}^{A_{P}} \left(T_{P_{O}}^{-1} - T_{O}^{-1} \right)$$

$$q_{R_{O}} = \sigma F_{O}^{A_{P}} \left(T_{P_{O}}^{-1} - T_{O}^{-1} \right)$$

*qRX - Only 20 percent of the radiation leaving the turbine casing and reflecting from the walls will return to the turbine casing to be absorbed or reflected. The other 80 per cent will reflect from the walls to the diverter valve and bellow sections. See Figure 9.108.

Areas - ft²:

	Section 1	Section 2	
A _T	1.12	. 93	
^А т А _W	. 62	. 51	
${f A}_{f F}$.50	. 42	
$\mathbf{A}_{\mathbf{p}}$.90	.74	
$\mathbf{A_f_p}$. 64	. 64	
A _f w	. 42	. 42	
$\mathbf{A_f}_\mathbf{F}$. 34	. 34	

$$\epsilon_{\mathbf{p_0}} = .80$$

$$\epsilon_{\rm w}$$
 = .15

$$\epsilon_{\rm F} = .15$$

$$\epsilon_{\rm B} = .80$$

Turbine Casing Overall Transfer Coefficients

Section 1

$$U_i = 40$$

Section 2

Example:

Standard Day

Sea Level

Static Condition

100% RPM

$$T_G = 1320^{\circ} F = 1780^{\circ} R$$

$$T_{A} = 70^{\circ} F = 530^{\circ} R$$

$$T_{B} = 150^{\circ} F = 610^{\circ} R$$

$$T_{O} = 60^{\circ} F = 520^{\circ} R$$

$$h_{T} = \frac{.0194 \text{ (PV)}^{.6}}{T^{.17} D^{.4}} = \frac{.0194 \text{ [2116 x 3]}^{.6}}{(530)^{.17} (1.416)^{.4}} = 1.14$$

$$h_F = .27 \left(\frac{\Delta T}{X}\right)^{1/4} = .27 \left(\frac{430}{2.0}\right)^{1/4} = 1.03$$

$$h_{W} = .29 \left(\frac{\Delta T}{X}\right)^{1/4} = .29 \left(\frac{530}{2.47}\right)^{1/4} = 1.10$$

$$h_{p_i} = .29 \left(\frac{\Delta T}{X}\right)^{1/4} = .29 \left(\frac{330}{3.58}\right)^{1/4} = 1.01$$

$$h_{B} = .12 \left(\frac{\Delta T}{X}\right)^{1/4} = .12 \left(\frac{300}{2}\right)^{1/4} = .42$$

$$h_0 = .29 \left(\frac{\Delta T}{X}\right)^{1/4} = .29 \left(\frac{90}{3.58}\right)^{1/4} = .65$$

$$F_X = .9 (.80) = .72$$

$$F_0 = .8$$

$$F_W = \frac{1}{\frac{1}{\epsilon_T} + \frac{1}{\epsilon_W} - 1} = \frac{1}{1.25 + 5.66} = .145$$

$$F_{p} = \frac{1}{\frac{1}{\epsilon_{T}} + \frac{1}{\epsilon_{p}} - 1} = \frac{1}{1.25 + 9.0} = .097$$

$$F_F = \frac{1}{\frac{1}{\epsilon_T} + \frac{1}{\epsilon_F} - 1} = \frac{1}{1.25 + 5.66} = .145$$

$$F_B = \frac{1}{\frac{1}{\epsilon_F} + \frac{1}{\epsilon_B} - 1} = \frac{1}{1.25 + 5.66} = .145$$

K_p for the Honeycomb Panel = .43 Btu/hr. ft² • F/in.

$$\ell = 1 \text{ in.}$$

$$\frac{K}{p} = .43$$

Set
$$T_x = 300^{\circ} F = 760^{\circ} R \left(\frac{T_x}{1000}\right)^4 = .334$$

Heat Transfer Equations

q = heat flux BTU/HR

SECTION 1	SECTION 2
$q_i = 44.8 (T_G - T_T)$	$q_i = 81.8 (T_G - T_T)$
$q_{C_{T-A}} = 1.28 (T_T - T_A)$	$q_{C_{T-A}} = 1.06 (T_T - T_A)$
$q_{R_{p}} = 120 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{p}}{1000} \right)^{4} \right]$	$q_{R_{p}} = 100 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{p}}{1000} \right)^{4} \right]$
$q_{R_W} = 119 \left[\left(\frac{T_T}{1000} \right)^4 - \left(\frac{T_W}{1000} \right)^4 \right]$	$q_{R_{\mathbf{W}}} = 93 \left[\left(\frac{T_{\mathbf{T}}}{1000} \right)^4 - \left(\frac{T_{\mathbf{W}}}{1000} \right)^4 \right]$
$q_{R_{F}} = 96 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{F}}{1000} \right)^{4} \right]$	$q_{R_{F}} = 75 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{F}}{1000} \right)^{4} \right]$
$q_{R_X} = 1395 \left(\frac{T_T}{1000}\right)^4 - 466$	$q_{R_X} = 1158 \left(\frac{T_F}{1000}\right)^4 - 387$
${}^{\mathbf{q}}\mathbf{C}_{\mathbf{w}-\mathbf{A}} = .68 (\mathbf{T}_{\mathbf{w}} - \mathbf{T}_{\mathbf{A}})$	$q_{C_{W}-A} = .56 (T_{W} - T_{A})$

SECTION 1	SECTION 2
$q_{CP-A} = .91 (T_{P_i} - T_A)$	$q_{CP-A} = .75 (T_{P_i} - T_A)$
$q_{CF-A} = .52 (T_F - T_A)$	$q_{CF-A} = .43 (T_F - T_A)$
$q_{KP} = .39 (T_{P_i} - T_{P_o})$	$q_{KP} = .32 \left(T_{P_i} - T_{P_o} \right)$
$q_{R_{\overline{B}}} = 125 \left[\left(\frac{T_{\overline{F}}}{1000} \right)^4 - \left(\frac{T_{\overline{B}}}{1000} \right)^4 \right]$	$q_{R_{B}} = 105 \left[\left(\frac{T_{F}}{1000} \right)^{4} - \left(\frac{T_{B}}{1000} \right)^{4} \right]$
$q_{C_{F-B}} = .21 (T_F - T_B)$	$q_{C_{F-B}} = .18 (T_{F} - T_{B})$ $q_{C_{O}} = .48 (T_{P_{O}} - T_{O})$
$q_{Co} = .58 (T_{Po} - T_o)$	$q_{Co} = .48 (T_{P_o} - T_o)$
$q_{R_0} = 1246 \left[\left(\frac{T_{P_0}}{1000} \right)^4 - \left(\frac{T_0}{1000} \right)^4 \right]$	$q_{R_0} = 1024 \left[\left(\frac{T_{P_0}}{1000} \right)^4 - \left(\frac{T_0}{1000} \right)^4 \right]$

Balanced Equations

Turbine Casing

$$q_i = q_{C_{T-A}} + q_{RP} + q_{Rw} + q_{R_F} + q_{RX}$$

Section 1

$$44.8 (T_{G}^{-T}T) = 1.28 (T_{T}^{-T}A) + 120 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{P_{i}}}{1000} \right)^{4} \right]$$

$$+ 119 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{W}}{1000} \right)^{4} \right] + 96 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{F}}{1000} \right)^{4} \right]$$

$$+ 1395 \left(\frac{T_{T}}{1000} \right)^{4} - 466$$

(Eq. 1)

$$120 \left(\frac{T_{P_i}}{1000}\right)^4 + 119 \left(\frac{T_W}{1000}\right)^4 + 96 \left(\frac{T_F}{1000}\right)^4 = 1.28(T_T - T_A)$$

$$+ 1730 \left(\frac{T_T}{1000}\right)^4 - 44.8(T_G - T_T) - 466$$

Section 2

$$81.8(T_{G}^{-}T_{T}^{-}) = 1.06(T_{T}^{-}T_{A}^{-}) + 100 \left[\left(\frac{T_{T}^{-}}{1000} \right)^{4} - \left(\frac{T_{P_{i}^{-}}}{1000} \right)^{4} \right]$$

$$+ 93 \left[\left(\frac{T_{T}^{-}}{1000} \right)^{4} - \left(\frac{T_{W}^{-}}{1000} \right)^{4} \right] + 75 \left[\left(\frac{T_{T}^{-}}{1000} \right)^{4} - \left(\frac{T_{F}^{-}}{1000} \right)^{4} \right]$$

$$+ 1158 \left(\frac{T_{T}^{-}}{1000} \right)^{4} - 387$$

$$100 \left(\frac{T_{P_i}}{1000}\right)^4 + 93 \left(\frac{T_W}{1000}\right)^4 + 75 \left(\frac{T_F}{1000}\right)^4 = 1.06 (T_T - T_A)$$

$$+ 1426 \left(\frac{T_T}{1000}\right)^4 - 81.8 (T_G - T_T) - 387$$

Vertical Wall

$$q_{R_W} = q_{C_{W-A}}$$

Section 1

119
$$\left[\left(\frac{T_T}{1000} \right)^4 - \left(\frac{T_W}{1000} \right)^4 \right] = .68 (T_W^- T_A)$$

(Eq. 3)

$$175\left(\frac{T_{T}}{1000}\right)^{4} + 529 = 175\left(\frac{T_{W}}{1000}\right)^{4} + T_{W}$$

Section 2

$$93\left[\left(\frac{T_{T}}{1000}\right)^{4} - \left(\frac{T_{W}}{1000}\right)^{4}\right] = .56 (T_{W} - T_{A})$$

(Eq. 4)

$$166 \left(\frac{T_{T}}{1000}\right)^{4} + 530 = 166 \left(\frac{T_{W}}{1000}\right)^{4} + T_{W}$$

Floor

$$q_{R_F} = q_{C_{F-A}} + q_{R_B} + q_{C_{F-B}}$$

Section 1

$$96\left[\left(\frac{T_{T}}{1000}\right)^{4} - \left(\frac{T_{F}}{1000}\right)^{4}\right] = .52 (T_{F} - T_{A}) + 125\left[\left(\frac{T_{F}}{1000}\right)^{4} - \left(\frac{T_{B}}{1000}\right)^{4}\right] + .21 (T_{F} - T_{B})$$

(Eq. 5)

$$132\left(\frac{T_{T}}{1000}\right)^{4} + 577 = 303\left(\frac{T_{F}}{1000}\right)^{4} + T_{F}$$

Section 2

$$75\left[\left(\frac{T_{T}}{1000}\right)^{4} - \left(\frac{T_{F}}{1000}\right)^{4}\right] = .43 (T_{F} - T_{A}) + 105\left[\left(\frac{T_{F}}{1000}\right)^{4} - \left(\frac{T_{B}}{1000}\right)^{4}\right]$$

$$+ .18 (T_{F} - T_{B})$$

(Eq. 6)

$$123 \left(\frac{T_T}{1000}\right)^4 + 577 = 295 \left(\frac{T_F}{1000}\right)^4 + T_F$$

Honeycomb Panel - Inside

$$q_{R_{\mathbf{p}}} = q_{K_{\mathbf{p}}} + q_{C_{\mathbf{p}-\mathbf{A}}}$$

Section 1

$$120 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{P}}{1000} \right)^{4} \right] = .39 \left(T_{P_{i}}^{-T_{P_{o}}} \right) + .91 \left(T_{P_{i}}^{-T_{A}} \right)$$

(Eq. 7)

$$T_{P_0} = 2.33 (T_{P_i} - T_A) + T_{P_i} - 308 \left[\left(\frac{T_T}{1000} \right)^4 - \left(\frac{T_P}{1000} \right)^4 \right]$$

Section 2

$$100 \left[\left(\frac{T_{T}}{1000} \right)^{4} - \left(\frac{T_{P}}{1000} \right)^{4} \right] = .32 \left(T_{P_{i}} - T_{P_{0}} \right) + .75 \left(T_{P_{i}} - T_{A} \right)$$

(Eq. 8)

$$T_{P_0} = 2.34 (T_{P_i} - T_A) + T_{P_i} - 312 \left[\left(\frac{T_T}{1000} \right)^4 - \left(\frac{T_P}{1000} \right)^4 \right]$$

Honeycomb Panel - Outside

$$q_{K_p} = q_{C_o} + q_{R_o}$$

Section 1

(Eq. 9)

$$.39 \left({}^{T}P_{i} - {}^{T}P_{o} \right) = .58 \left({}^{T}P_{o} - {}^{T}O_{o} \right) + 1246 \left[\left(\frac{{}^{T}P_{o}}{1000} \right)^{4} - \left(\frac{{}^{T}O_{o}}{1000} \right)^{4} \right]$$

Section 2

(Eq. 10)

$$.32 \left({^{T}P_{i}}^{-} {^{T}P_{o}} \right) = .48 \left({^{T}P_{o}}^{-} {^{T}O} \right) + 1024 \left[\left(\frac{{^{T}P_{o}}}{1000} \right)^{4} - \left(\frac{{^{T}O}}{1000} \right)^{4} \right]$$

Heat Transfer Balance - Section One

Assume
$$T_{T} = 1089.1 \,^{\circ} F = 1549.1 \,^{\circ} R, T_{T}^{4} = 5.760$$

(From Eq. 1)

$$120\left(\frac{T_{P_i}}{1000}\right)^4 + 119\left(\frac{T_W}{1000}\right)^4 + 96\left(\frac{T_F}{1000}\right)^4 = 1.28(1019.1) + 1730 (5.76)$$

$$- 44.8 (230.9) - 466$$

(Eq. 11)

$$120\left(\frac{{}^{T}P_{i}}{1000}\right)^{4} + 119\left(\frac{{}^{T}W}{1000}\right)^{4} + 96\left(\frac{{}^{T}F}{1000}\right)^{4} = 459$$

(From Eq. 3)

$$175 (5.76) + 529 = 175 T_{W}^{4} + T_{W}$$

$$1536 = 175 T_{W}^{4} + T_{W}$$

$$Set T_{W} = 1188 ^{\circ} R = 728 ^{\circ} F \left(\frac{T_{W}}{1000}\right)^{4} = 1.991$$

$$1536 = 1536$$

(From Eq. 5)

$$132 (5.76) + 577 = 303 T_{F}^{4} + T_{F}$$

$$1337 = 303 T_{F}^{4} + T_{F}$$

$$Set T_{F} = 1016 ^{\circ}R = 556 ^{\circ}F \left(\frac{T_{F}}{1000}\right)^{4} = 1.065$$

$$1337 = 1337$$

(From Eq. 11)

$$120 \left(\frac{T_{P_i}}{1000}\right)^4 + 119 (1.991) + 96 (1.065) = 459$$

$$120 \left(\frac{{}^{T}P_{i}}{1000}\right)^{4} = 120$$

$$T_{P_i} = 1000 \, ^{\circ}R = 540 \, ^{\circ}F$$

(From Eq. 7)

$$T_{P_O} = 2.33 (470) + 1000 - 308 (4.76)$$

$$T_{P_{O}} = 629 \, {}^{\circ}R = 169 \, {}^{\circ}F \left(\frac{T_{P_{O}}}{1000}\right)^{4} = .156$$

(From Eq. 9)

$$.39(371) = .58(109) + 1246(.083)$$

Temperature Summary - Section One

$$T_G = 1320 \, ^{\circ} F$$

$$T_{T} = 1089.1 \circ F$$

$$T_A = 70 \cdot F$$

$$T_{\mathbf{P_i}} = 540 \, ^{\circ} \mathbf{F}$$

$$T_{\mathbf{p}_{\mathbf{O}}} = 169 \, {}^{\circ}\,\mathbf{F}$$

$$T_W = 728 ° F$$

$$T_F = 556 ° F$$

$$T_B = 150 ° F$$

$$T_O = 60 ° F$$

Heat Transfer Balance - Section Two

Assume
$$T_T = 1133.6 \text{ °F} = 1593.6 \text{ °K}, \left(\frac{T_T}{1000}\right)^4 = 6.449$$

(From Eq. 2)

$$100\left(\frac{T_{P_i}}{1000}\right)^4 + 93\left(\frac{T_W}{1000}\right)^4 + 75\left(\frac{T_F}{1000}\right) = 1.06 (1063.6) + 1426 (6.449)$$

$$-81.8 (116.4) - 387$$

(Eq. 12)

$$100\left(\frac{T_{P_i}}{1000}\right)^4 + 93\left(\frac{T_W}{1000}\right)^4 + 75\left(\frac{T_F}{1000}\right)^4 = 415$$

(From Eq. 4)

$$166 (6.449) + 530 = 166 \left(\frac{T_W}{1000}\right)^4 + T_W$$

$$T_W = 1226 \, {}^{\circ}R = 766 \, {}^{\circ}F \left(\frac{T_W}{1000}\right)^4 = 2.259$$

(From Eq. 12)

$$100\left(\frac{{}^{T}P_{i}}{1000}\right)^{4} + 93(2.259) + 75(1.143) = 415$$

$$100 \left(\frac{T_{P_i}}{1000}\right)^4 = 119 \qquad \left(\frac{T_{P_i}}{1000}\right)^4 = 1.190$$

$$T_{P_i} = 1045 \, ^{\circ}R = 585 \, ^{\circ}F$$

(From Eq. 8)

$$T_{P_O} = 2.34 (515) + 1045 - 312 (5.259)$$

$$T_{P_O} = 609 \, {}^{\circ}R = 149 \, {}^{\circ}F \, \left(\frac{T_{P_O}}{1000}\right)^4 = .137$$

(From Eq. 10)

$$.32 (436) = .48 (89) + 1024 (.064)$$

$$139 \approx 108$$

Temperature Summary - Section Two

$$T_G = 1320 \circ F$$

$$T_{T} = 1133.6 \, ^{\circ}F$$

$$T_A = 70 \circ F$$

$$T_{P_i} = 585 \, ^{\circ}F$$

$$T_{\mathbf{P}} = 149 \, ^{\circ} \mathbf{F}$$

$$T_W = 766 \,^{\circ} F$$

$$T_F = 574 \,^{\circ} F$$

$$T_B = 150 \, ^{\circ} F$$

$$T_{O} = 60 \circ F$$

Diverter Valve - Section 3

The heat transfer analysis of the diverter valve section is similar to the turbine casing analysis. Only the value of the coefficients areas and temperatures are changed. The Diverter Valve is insulated and has an overall heat transfer coefficient of 1.2 Btu/hr.ft² • F. See Figure 9.110.

Emissivities

$$\epsilon_{\rm T} = .50$$

$$\epsilon_{P_i} = .10$$

$$\epsilon_{\rm p_o} = .80$$

$$\epsilon_{\rm w} = .15$$

$$\epsilon_{\rm F}$$
 = .15

$$\epsilon_{\rm B} = .80$$

Bellows - Section 4

The bellow section is similar to the turbine casing section except for a shroud around the tailpipe that allows cooling air to flow between the shroud and tailpipe, see Figure 9.111. The heat flow from the shroud to the walls is the same as in the turbine casing section, $\mathbf{q_i}$ is equal to the total heat input into the shroud.

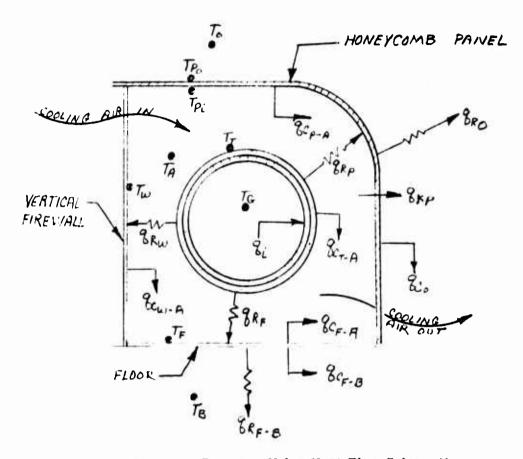


Figure 9.110 Diverter Valve Heat Flow Schematic

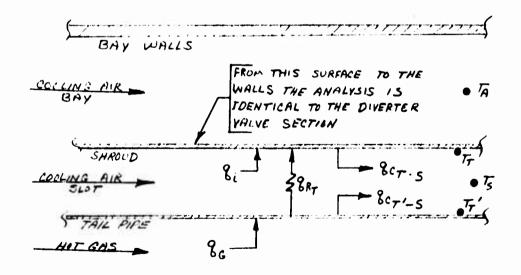


Figure 9.111 Bellows Heat Flow Schematic

$$q_G = q_{R_T} + q_{C_T}^1 - S$$

$$h_{G} A_{T'} \left(T_{G}^{-}T_{T'}\right) = \sigma F_{T}^{A} F' \left(T_{T'}^{4}T_{T}^{4}\right) + h_{T'} A_{T'} \left(T_{T'}^{-}T_{S}\right)$$

 T_G and T_S are known

Set T_T'

Define coefficients and areas

Find T_T

$$q_i = q_{R_T} - q_{C_T} - S$$

$$q_{i} = \sigma F_{T}A_{T'} \left(T_{T'}^{4} - T_{T}^{4}\right) - h_{T}A_{T} \left(T_{T} - T_{S}\right)$$

Define coefficients and areas

$$T_{T'}$$
, T_{T} , and T_{S} are known

Find q

With q_i and T_T known, analyze the whole system as outlined in the turbine sections. Pick various values of $T_{T'}$ until the whole system is balanced.

9.4.12 Aft Fuselage Heat Transfer Analysis

The aft fuselage is heated by two turbojet engine tailpipes passing diagonally through the section. The tailpipes are shrouded and cooling air is pumped through the annulus formed by the tailpipe and shroud. There is no cooling air flowing between the shrouds and fuselage skin, therefore the free convective heat transferred from the shrouds must enter the fuselage skin by free convection. The tailpipe, shroud, and fuselage skin materials are very thin, therefore the temperature across the material is assumed to be uniform. The aft fuselage is divided into

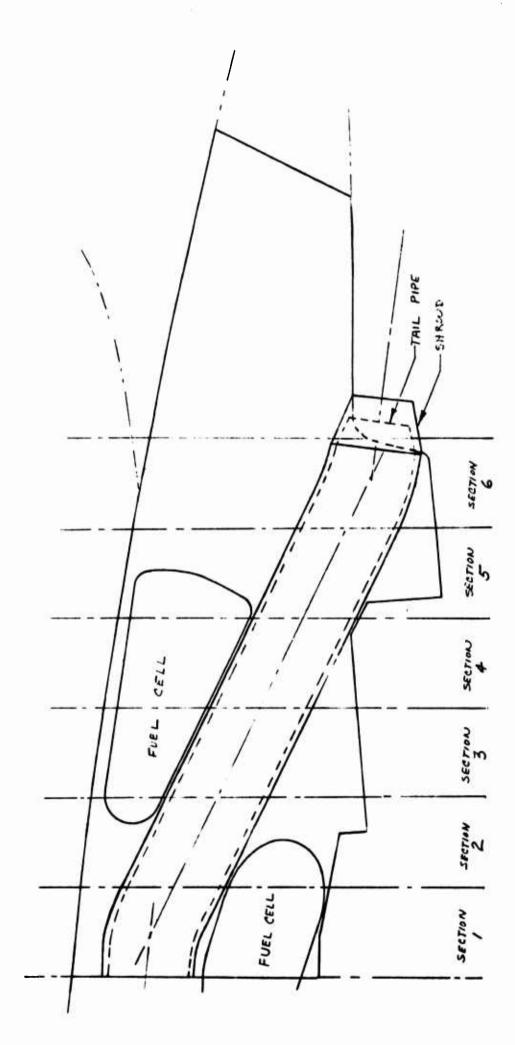


Figure 9.112 Heating and Cooling Schematic - Aft Fuselage

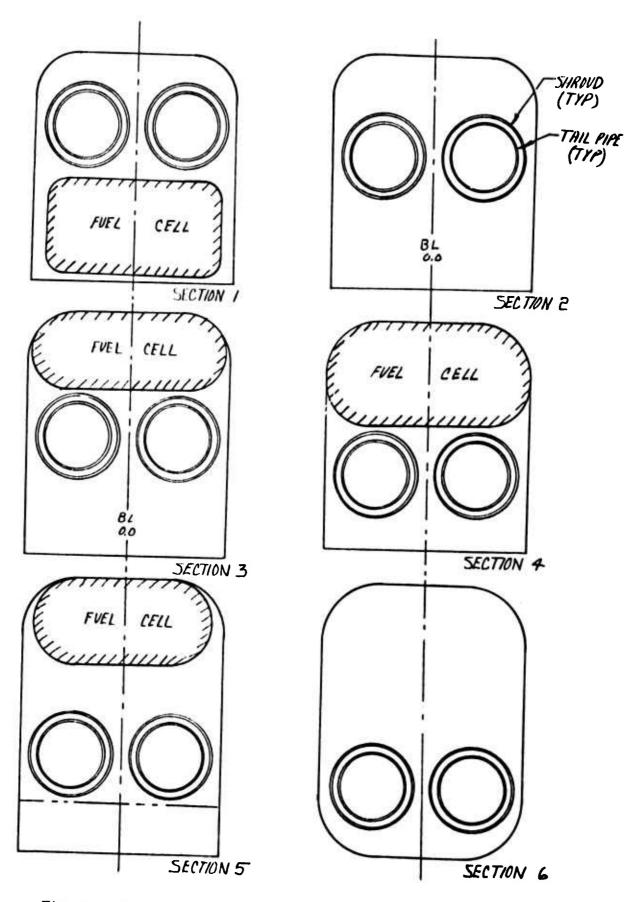


Figure 9.113 Heating and Cooling Schematic - Aft Fuselage Sections

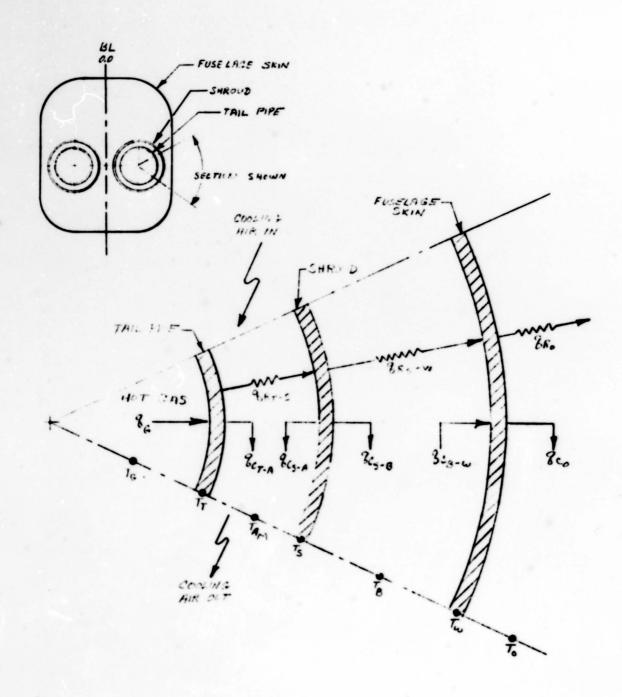


Figure 9, 114 Aft Fuselage Heat Balance Schematic

six sections as presented in Figures 9.112 and 9.113. The heat balance across a section is schematically shown in Figure 9.114.

Basic Heat Transfer Equations

$$q_{G} = h_{G} A_{T} (T_{G}^{-1}T)$$

$$q_{R_{T-S}} = \sigma F_{T} A_{T} (T_{T}^{4} - T_{S}^{4})$$

$$q_{C_{T-A}} = h_{T} A_{T} (T_{T}^{-1}T_{AM})$$

$$q_{C_{S-A}} = h_{S_{i}} A_{S} (T_{S}^{-1}T_{AM})$$

$$q_{R_{S-W}} = \sigma F_{S} A_{f} (T_{S}^{4} - T_{W}^{4})$$

$$q_{C_{S-B}} = h_{S_{i}} A_{S} (T_{S}^{-1}T_{B})$$

$$q_{C_{B-W}} = h_{W_{i}} A_{W} (T_{B}^{-1}T_{W})$$

$$q_{R_{O}} = \sigma F_{O} A_{W} (T_{W}^{4} - T_{O}^{4})$$

$$q_{C_{O}} = h_{W_{i}} A_{W} (T_{W}^{-1}T_{O})$$

Balanced Equations

$$q_G = q_{R_{T-S}} + q_{C_{T-A}}$$

$$q_A = q_{C_{T-A}} + q_{C_{S-A}}$$

$$q_{R_{S-W}} + q_{C_{B-W}} = q_{R_O} + q_{C_O}$$

$$q_{C_{S-B}} = q_{C_{B-W}}$$

$$q_{G} = q_A + q_{R_O} + q_{C_O}$$

Example

The following example is an analysis of the heat transfer in section one for a condition at standard day, sea level, static operation, 100% RPM, and turbojet mode.

Known Conditions

$$T_{G} = 1250^{\circ} F = 1710^{\circ} R$$

$$T_{O} = 60^{\circ} F = 520 \circ R$$

Air Temp. into the section = 88° F = 548 ° R

$$A_{T} = 6.48 \text{ ft}^2/\text{section}$$

$$A_S = 7.65 \text{ ft}^2/\text{section}$$

$$A_{f} = .863$$

Section	A _w
1	9.0
2	13.0
3	11.0
4	9.7
5	11.0
6	13.5

Emissivities

$$\epsilon_{\mathrm{T}} = .40$$
 $\epsilon_{\mathrm{S}} = .12$ $\epsilon_{\mathrm{W}} = .90$

$$\epsilon_{\mathrm{S}_{1}} = .10$$
 $\epsilon_{\mathrm{W}_{1}} = .90$

Calculations

 $\mathbf{h}_{\widehat{\mathbf{G}}}$ Tailpipe hot gas transfer coefficient

$$\frac{h_{G}}{\rho c_{p} V} = \frac{.0384 {R_{e_{d}}}^{-1/4}}{1 + 1.5 (P_{r})^{-1/6} {R_{e_{d}}}^{-1/8} (P_{r}^{-1})}$$

$$\mu = 2.68 \times 10^{-5} \text{ lb/sec.ft}$$

$$\rho = .047 \text{ lb ft}^2$$

$$P_r = .70$$

$$c_{p_a} = .27 \text{ Btu/lb} \cdot \text{F}$$

$$A = 1.48 \text{ ft}^2$$

$$D = 1.375 \text{ ft.}$$

$$W = 44 lb/sec.$$

$$V = \frac{W}{A\rho} = 632 \text{ ft/sec.}$$

$$R_{e_d} = \frac{D\rho V}{\mu} = 1.523 \times 10^6$$

$${\binom{R_{e_d}}{}}^{-1/4} = .028 {\binom{R_{e_d}}{}}^{-1/8} = .168 {\binom{P_r}{}}^{-1/6} = 1.061$$

$$h_{G} = \frac{\text{(.047) (.27) (632) (.0384) (.028)}}{1 + 1.5 (1.061) (.168) (-.3)} \left(3600 \frac{\text{sec}}{\text{hr}}\right)$$

$$h_G = 34 \frac{Btu}{hr-ft^2- F}$$

Annulus:

 h_{T} = Outside surface of the tailpipe

 h_{S_i} - Inside surface of the shroud

$$\frac{h}{c_{p_b}^{G}} \left(\frac{c_p^{\mu}}{k}\right)_b \left(\frac{\mu w}{\mu_b}\right)^{14} = \frac{.023}{\left(\frac{D_H^G}{\mu_b}\right)^{2}}$$

Subscript b = bulk

$$c_{p_b} = .24 \text{ Btu/lb } ^{\circ} \text{ F}$$

$$G = 3390 lb/hr-ft^2$$

$$\mu_{\rm b} = 5.08 \times 10^{-2} \, \rm lb/hr-ft$$

$$\mu W_{S_i} = 5.01 \times 10^{-2} \text{ lb/hr-ft}$$

$$\mu W_{T} = 8.71 \times 10^{-2} \text{ lb/hr-ft}$$

$$D_{H} = .25 \text{ ft.}$$

$$h = \frac{.023}{\left(\frac{D_{H}^{G}}{\mu_{b}}\right)^{.2}} \frac{c_{p_{b} G}}{\left(\frac{\mu_{w}}{\mu_{b}}\right)^{.14} \left(\frac{c_{p}^{\mu}}{k}\right)^{2/3}}$$

$$h_{s_i} = 3.43 \text{ Btu/hr-ft}^2 \circ \text{F}$$

$$h_{T} = 3.18 \text{ Btu/hr-ft}^2 \circ F$$

h - Outside surface of shroud

$$h_{s_1} = .27 \left(\frac{P}{14.7}\right)^{1/2} \left(\frac{\Delta T}{D}\right)^{1/4} = .24 \Delta T^{1/4}$$

 $\mathbf{h}_{\mathbf{W_i}}$ - Inside surface of the fuselage skin

 $h_{\begin{subarray}{c} \mathbf{w}_{\mathbf{l}} \end{subarray}}$ - Outside surface of the fuselage skin

In terms of ΔT , $h_{w_i} = h_{w_f} = .29 \left(\frac{P}{14.7}\right)^{1/2} \left(\frac{\Delta T}{D}\right)^{1/4}$

Section	${f w}_{f i}^{ m or h}_{f w}$
1	. 452 ΔT ^{1/4}
2	$.499 \ \Delta T^{1/4}$
3	$.478 \ \Delta T^{1/4}$
4	$.461\ \Delta T^{1/4}$
5	$.478\Delta T^{1/4}$
6	$.501 \ \Delta T^{1/4}$
$F_{T} = \frac{1}{\frac{1}{\epsilon_{T}} + \left(\frac{A_{T}}{A_{S}}\right) \left(\frac{1}{\epsilon_{B_{I}}} - 1\right)}$	= .099

$$F_{S} = \frac{1}{\frac{1}{\epsilon_{S_{l}}} + \left(\frac{A_{S}^{A_{f}}}{A_{T}}\right) \left(\frac{1}{\epsilon_{W_{i}}} - 1\right)} = .118$$

$$F_{O} = .80$$

Heat Transfer Equations

$$q_G = 220 (T_G - T_T)$$

$$q_{R_{T-S}} = 1110 \left[\left(\frac{T_T}{1000} \right)^4 - \left(\frac{T_S}{1000} \right)^4 \right]$$

$$q_{C_{T-A}} = 20.6 (T_{T} T_{A_{m}})$$

$${}^{q}C_{S-A} = 26.6 (T_{S} T_{A_{m}})$$

$$q_{R_{S-W}} = 1347 \left[\left(\frac{T_S}{1000} \right)^4 - \left(\frac{T_W}{1000} \right)^4 \right]$$

$$q_{C_{S-B}} = 1.84 (T_S - T_B)^{1.25}$$

$${}^{q}{}_{C}{}_{B-W}: \underline{SECTION} \qquad {}^{q}{}_{C}{}_{B-W} \qquad {}^{q}{}_{C}{}_{O}$$
 and 1 4.07 $({}^{T}{}_{B}-{}^{T}{}_{W})^{1.25}$ 4.07 $({}^{T}{}_{W}-{}^{T}{}_{O})^{1.25}$ ${}^{q}{}_{C}{}_{O}$ 2 6.48 $({}^{T}{}_{B}-{}^{T}{}_{W})^{1.25}$ 6.48 $({}^{T}{}_{W}-{}^{T}{}_{O})^{1.25}$ 3 5.26 $({}^{T}{}_{B}-{}^{T}{}_{W})^{1.25}$ 5.26 $({}^{T}{}_{W}-{}^{T}{}_{O})^{1.25}$

^q C _{B-W} :	SECTION	${}^{\mathbf{q}}\mathbf{c}_{\mathbf{B-W}}$	${\bf q_{C^{O}}}$
and	4	4.47 (T _B -T _W) ^{1.25}	4.47 (T _W -T _O) ^{1.25}
^q C _O	5	5.26 (T _B -T _W) ^{1.25}	5.26 (T _W -T _O) ^{1.25}
(Cont)	6	6.76 (T _B -T _W) ^{1.25}	6.76 (T _W -T _O) ^{1.25}
q _{R_O} :	SECTION	$\frac{\mathbf{q}_{\mathbf{R}_{\mathbf{O}}}}{\mathbf{Q}}$	
	1	1.40 x $10^4 \left[\left(\frac{T_W}{1000} \right)^4 \right]$	$-\left(\frac{\mathbf{T_O}}{1000}\right)^4$
	2	$3.02 \times 10^4 \left[\left(\frac{T_W}{1000} \right)^4 \right]$	$-\left(\frac{\mathrm{T_{O}}}{1000}\right)^{4}$
	3	1.71 x $10^4 \left[\left(\frac{T_W}{1000} \right)^4 \right]$	$-\left(\frac{\mathrm{T_{O}}}{1000}\right)^{4}$
	4	1.51 x $10^4 \left[\left(\frac{T_W}{1000} \right)^4 \right]$	$-\left(\frac{\mathrm{T_{O}}}{1000}\right)^{4}$
	5	$1.71 \times 10^4 \left[\left(\frac{T_W}{1000} \right)^4 \right]$	$-\left(\frac{\mathrm{T_{O}}}{1000}\right)^{4}$
	6	$2.10 \times 10^4 \left[\left(\frac{T_W}{1000} \right)^4 \right]$	$-\left(\frac{\mathbf{T_{O}}}{1000}\right)^{4}$

Balance for Section One

Using the Balance equations

(1)
$$q_G = q_{T-S} + q_{T-A}$$

$$h_{G}^{A}_{T} (T_{G}^{-T}_{T}) = \sigma F_{T}^{A}_{T} (T_{T}^{4} - T_{S}^{4}) + h_{T}^{A}_{T} (T_{T}^{-T}_{A})$$

200
$$(T_G^{-T}T) = 1110 \left[\left(\frac{T_T}{1000} \right)^4 - \left(\frac{T_T}{1000} \right)^4 \right] + 20.6 \left(T_T^{-T}A_m \right)$$

- (2) Set $T_T = 1124.26 \, ^{\circ}F = 1584.26 \, ^{\circ}R$
- (3) Assume $\Delta T_a = 28^{\circ} R$
- (4) Solve for T_S

$$q_G = 27,663 Btu/hr$$

$$qC_{T-A} = 21,058 \text{ Btu/hr}$$

$$q_{R_{T-S}} = 27,663 - 21,058 = 6,605 \text{ Btu/hr}$$

$$1110 \left(\frac{T_S}{1000}\right)^4 = 110 \left(\frac{T_T}{1000}\right)^4 - 6.605$$

$$\left(\frac{T_S}{1000}\right)^4 = 6.300 - 5.950 = .350$$

$$T_S = 769 \,^{\circ} R$$

(5) Solve for
$${}^{q}C_{S-A}$$

$${}^{q}C_{S-A} = 26.6 \quad {}^{T}S^{-T}A_{m} = 5,506 \text{ Btu/hr}$$

(6) Solve for ΔT_a to check step (3)

$${}^{q}A = {}^{q}C_{S-A} + {}^{q}C_{T-A} = 21,058 + 5,506 = 26,564 \text{ Btu/hr}$$

$$\Delta T = \frac{q_A}{WC_p} = \frac{26,560 \text{ Btu/hr}}{1.10 \text{ lb/sec . 27 Btu/lb °F}} \left(\frac{\text{hr}}{3600 \text{ sec}}\right) = 28 \text{ °F}$$

(7) Solve for T_W

$${}^{q}C_{S-B} + {}^{q}R_{S-W} = {}^{q}R_{T-S} - {}^{q}C_{S-A} = 1099 \text{ Btu/hr}$$

$${}^{q}C_{S-B} + {}^{q}R_{S-W} = {}^{q}R_{O} + {}^{q}C_{O}$$

$$1099 = 4.07 (T_W^{-1}^{-1})^{1.25} + 1.40 \times 10^4 \left[\left(\frac{T_W}{1000} \right)^4 - \left(\frac{T_O}{1000} \right)^4 \right]$$

$$T_W = 575 \, ^{\circ} R$$

(8) Solve for TB

$${}^{q}C_{S-B} = {}^{q}C_{B-W}$$

1.84
$$(T_S - T_B)^{1.25} = 4.07 (T_B - T_W)^{1.25}$$

$$\left(\frac{T_S^{-T}_B}{T_B^{-T}_W}\right)^{1.25} = 2.21$$

(9) Using T_W , T_B , and T_S , Solve for

 $^{q}C_{S-B}^{}$ and $^{q}R_{S-W}^{}$ and check for the value obtained in step (7)

$$q_{C_{S-B}} = 1.84 (T_{S}^{-}T_{B}^{-})^{1.25} = 786 \text{ Btu/hr}$$

$$q_{R_{S-W}} = 1347 \left[\left(\frac{T_S}{1000} \right)^4 - \left(\frac{T_W}{1000} \right)^4 \right] = 324 \text{ Btu/hr}$$

$${}^{q}C_{S-B}$$
 ${}^{+}{}^{q}R_{S-W}$ = 1110 Btu/hr

This is close enough to the value 1099 Btu/hr obtained in step (7).

(10) Check overall system

$$q_G = q_A + q_{R_O} + q_{C_O}$$

$$27,663 \approx 27,659$$

(11) Summary of temperatures

$$T_G = 1250 \, ^{\circ} F$$

$$T_{T} = 1124.26 \, ^{\circ} F$$

$$T_S = 309 \cdot F$$

$$T_{A_m} = 102 \cdot F$$

$$T_B = 182 \, ^{\circ}F$$

$$T_{W} = 115 \, ^{\circ} F$$

$$T_O = 60 \, ^{\circ} F$$

The sections two thru six are analyzed in the same manner, using the values of q given in this section, decreasing T_B by the value of

$$\left(\frac{\mathbf{q}_{\mathbf{G}}}{\mathbf{w}_{\mathbf{G}}\mathbf{c}_{\mathbf{p}_{\mathbf{O}}}}\right)$$

and obtaining \mathbf{T}_{A_m} by using \mathbf{T}_a in as \mathbf{T}_a in + ΔT from the preceding section.

9.5 STRUCTURAL PROTECTION SYSTEM ANALYSIS

9.5.1 Insulation of Nose Fan Thrust Reverser Door

Upper Closure Longeron Assembly Part No. 143F003

A steady state heat transfer analysis of insulation requirements for Assembly Part No. 143F003 was made using the simplified model of Figure 9.115 below (see Figure 6.1 for reference).

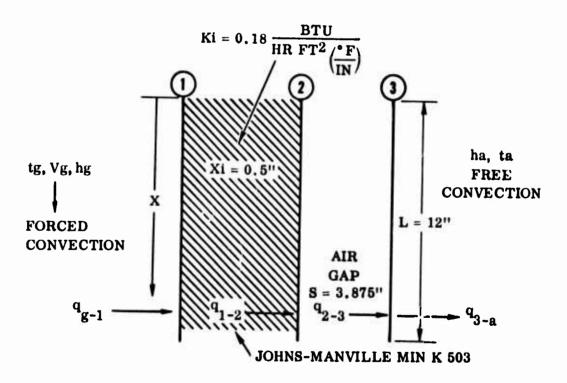


Figure 9.115 Part No. 143F003 Heat Transfer Model

For steady state

$$q_{g-a} = q_{g-1} = q_{1-2} = q_{2-3} = q_{3-a}$$

From which

$$\frac{1}{U_0} = \frac{1}{h_g} + \frac{X_i}{k_i} + \frac{1}{h_2} + \frac{1}{h_a}$$

The heat transfer coefficient hg is obtained from Equation 34 or 36 on Page I-25 of Reference 18; the choice depending upon whether or not

Re =
$$1.7 \times 10^5 \frac{V_g PX}{T_1^{1.75}} \stackrel{<}{>} 5 \times 10^5$$

The heat transfer coefficient h_2 between walls 2 and 3 of Figure 9.115 accounts for both convection (h_{2c}) and radiation (h_{2r}) .

The convective term h_{2c} may be read directly from Figure 1c-35 on Page 1-c-58 of Reference 12 at δ = 3.875" and an assumed Δt between the plates (Δt = t_2 - t_3).

The radiative term h_{2r} is determined from the equation

$$h_{2r} = \frac{1730 \text{ F}_{A\epsilon} \left[\left(\frac{T_2}{1000} \right)^4 - \left(\frac{T_3}{1000} \right)^4 \right]}{(T_2 - T_3)}$$

where T = t + 460.

Since neither \mathbf{T}_2 nor \mathbf{T}_3 are known, trial and error is required.

The heat transfer coefficient h_a likewise includes convective (h_{ac}) and radiative (h_{ar}) components so that $h_a = h_{ac} + h_{ar}$.

The convective term $h_{\rm aC}$ is determined from either equations 104 or 105, Page 1-c-56, Reference 13, depending upon the value of Gr Pr which is easily obtained from the equation

The radiative term h is obtained from the equation

$$h_{ar} = \frac{1730 \text{ F}_{A\epsilon} \left[\left(\frac{T_3}{1000} \right)^4 - \left(\frac{T_a}{1000} \right)^4 \right]}{(T_3 - T_a)}$$

Since the solution is by trial and error, it is convenient to use the fact that the ratio of component temperature differences to the total temperature difference is equal to the ratio of component thermal resistance to total thermal resistance so that

$$\frac{\frac{t_{g}-t_{2}}{t_{g}-t_{a}}}{\left[\frac{1}{h_{g}} + \frac{X_{i}}{k_{i}}\right]} = \frac{\left[\frac{1}{h_{g}} + \frac{X_{i}}{k_{i}}\right]}{\left[\frac{1}{h_{g}} + \frac{X_{i}}{k_{i}} + \frac{1}{h_{2}} + \frac{1}{h_{a}}\right]}$$

and

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$$\frac{\frac{t_g - t_3}{t_g - t_a}}{\frac{1}{t_g} - \frac{1}{t_g}} = \frac{\left[\frac{1}{h_g} + \frac{X_i}{k_i} + \frac{1}{h_2}\right]}{\left[\frac{1}{h_g} + \frac{X_i}{k_i} + \frac{1}{h_2} + \frac{1}{h_a}\right]}$$

Sample Calculation:

Initial Conditions

$$V_g = 300 \text{ ft/sec}, P = 2118 \text{ lbs/ft}^2, t_g = 700^{\circ} \text{ F}$$

$$X_i = 0.5'', k_i = 0.18'', X = 3'', t_g = 100^{\circ} \text{ F}$$

Assume
$$t_1 = 685^{\circ} F$$
, $t_2 = 244^{\circ} F$, $t_3 = 174^{\circ} F$ $F_{A\epsilon} = .9$

(Note $T = t + 460$)

 $R_e = 1.7 \times 10^5 (300) (2118) (3/12) / (685 + 460)^{1.75} = 1.17$
 $\times 10^5 < 5 \times 10^5$

Use Equation 34 Page I-25 Reference 18

$$h_{g} = 0.0077 \left(\frac{V_{g}P}{X}\right)^{1/2} = 0.0077 \left[\frac{(300)(2118)}{(3/12)}\right]^{.5}$$

$$= 12.2 \frac{Btu}{hr.ft._{o}^{2}F}$$

 h_{2c} = 0.43 from Figure IC-35 Page 1-c-58 Reference 12 at Δt = 70° F and δ = 3.875"

$$h_{2r} = (1730) (F_{A\epsilon}) \left[\left(\frac{T_2}{1000} \right)^4 - \left(\frac{T_3}{1000} \right)^4 \right] / (t_2 - t_3)$$

$$= (1730) (.9) \left[\frac{4}{.704} - \frac{4}{.634} \right] / (244 - 174) = 1.87$$

$$h_2 = h_{2c} + h_{2r} = 2.30$$

To obtain h_a = h_{ac} + h_{ar} check (Gr) (Pr) first.

$$Y = 9.2 \times 10^5 \text{ at } t = (t_3 + t_a) / 2 = 137$$

Then

GrPr =
$$Y(t_3-t_a)(L)^3 = (9.2 \times 10^5)(74)(1) = 6.8 \times 10^7$$

Use Equation 104 Page 1-c-56 Reference 12

$$h_{ac} = 0.29 \left[\frac{P}{(144) (14.7)} \right]^{1/2} \left(\frac{\Delta t}{X} \right)^{1/4}$$

$$h_{ac} = (0.29) (74)^{1/4} = .85$$

$$h_{ar} = (1730) (F\Delta\epsilon) \left[\left(\frac{T_3}{1000} \right)^4 - \left(\frac{T_a}{1000} \right)^4 \right]$$

$$= (1730) (.9) \left[\frac{4}{.634} - \frac{4}{560} \right] / 74 = 1.29$$

$$h_a = .85 + 1.29 = 2.14$$

$$\frac{1}{U_0} = \left[\frac{1}{h_g} + \frac{X_i}{k_i} + \frac{1}{h_2} + \frac{1}{h_a} \right]$$

$$= \left[\frac{1}{12.2} + \frac{.5}{.18} + \frac{1}{2.3} + \frac{1}{2.14} \right]$$

$$= .082 + 2.78 + .435 + .467 = 3.764$$

To check assumptions of $t_2 = 244$ and $t_3 = 174$

$$\frac{\frac{t_g - t_2}{g^2}}{\frac{t_g - t_a}{g^2}} = \frac{\frac{1}{h_g} + \frac{X_i}{g}}{\frac{1}{U_o}} = \frac{2.862}{3.764} = .761 = \frac{700 - t_2}{700 - 100}$$

$$t_2 = 700 - .761 (600) = 700-456 = 244 ok$$

$$\frac{\frac{t_g - t_3}{g - t_a}}{\frac{t_g - t_a}{g}} = \frac{\frac{1}{h_g} + \frac{X_i}{k_i} + \frac{1}{h_2}}{\frac{1}{U_0}} = \frac{3.297}{3.764} = .876 = \frac{700 - t_3}{700 - 100}$$

$$t_3 = 700 - 525 = 175$$
 close enough

Conclusion

0.5" Johns Manville Min K 503 insulation will keep the longeron assembly below the design load limit of 250° F. Edges should be sealed to prevent "blow-by" of hot gases behind the insulation.

9.5.2 Insulation Requirements for Local Aircraft Surface Areas

9.5.2.1 Method of Analysis

The following procedure is applicable to transient heat transfer analysis of aircraft surface insulation systems shown in Figure 2.3. It is based primarily on the numerical method of Dusinberre as presented in Reference 15. The one-dimensional heat transfer model, presented in Figure 9.116, consists of a thin metal plate protected by a relatively thick layer of insulation. The plate is assumed large enough that edge effects are negligible, assumes negligible contact resistance between the insulation and metal plate, and assumes that the metal plate is thin enough that it may be treated as if it had infinite thermal conductivity. Radiation from the hot and cold sides is neglected; thereby adding some conservatism to the method since more heat is added to the hot side and less heat is lost from the cold side than would be the case if it had been included.

The insulation of thickness X_i is divided into n slabs of thickness $\Delta X_i = X_i/n$. The general equation for determining the temperature t_i at the i^{th} interface for $1 \le i \le n-1$ at the $j+1^{th}$ time increment is

$$t_{i, j+1} = \frac{t_{i-1, j} + (M_A^{-2}) t_{i, j} + t_{i+1, j}}{M_A}$$

At the insulation surface where $t_i = t_0$, the equation

$$t_{0, j+1} = \frac{N_A}{N_A + 1} t_{g, j+1} + \frac{1}{N_A + 1} t_{1, j+1}$$

is used, except for the first time increment following the initial application of $t_{\rm g,\ i+1}$ to the system, where the approximation

$$t_{0,j} = (t_{0,j} + t_{g,j})/2$$
 is used. (Note the subscripts are correct.)

At the insulation-metal plate interface i = n an iteration step is required as follows

Assume
$$t_{p, j+1} = t_{n, j}$$

$$\Delta t_{p} = E t_{n-1, j+1} - F t_{p, j+1} + G t_{a}$$

and then a calculated value of $t_{p, j+1}$ is given by

$$t_{p, j+1} = t_{n, j} + \Delta t_{p}$$

Obviously the first assumption of $t_{p, j+1}$ is incorrect; so the next trial assumes the value of $t_{p, j+1}$ just calculated. This procedure is repeated until the assumed and calculated value agree within some specified limit at which time the statement is made that $t_{n, j+1} = t_{p, j+1}$.

Values of MA, NA, E, F, and G are defined below.

$$M_{A} = \frac{c_{p_{i}}^{\gamma_{i}} (\Delta X_{i})^{2}}{k_{i} \Delta \theta}$$

$$N_{A} = \frac{h_{g_{i}} \Delta X_{i}}{k_{i}}$$

$$E = \frac{(\triangle \theta) (k_i)}{(60) (144) (X_m) (\rho_m) (C_{p_m}) (\triangle X_i)}$$

$$F = \left(\frac{\Delta \theta}{(60) (144) (X_m) (\rho_m) (C_{p_m})}\right) \left(\frac{k_i}{\Delta X_i} + h_a\right)$$

$$G = \frac{\binom{h}{a} (\Delta \theta)}{(60) (144) (X_m) (\rho_m) (C_{p_m})}$$

The above equations were programed for digital computer use. The term M_A is arbitrarily set at $M_A = 2$ or greater to obtain the time increment $\Delta\theta$. Time varying boundary conditions together with any initial temperature distribution in the insulation may be handled with

TABLE 9.11
Insulation System Study Summary

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9				.500					T	1000					13.7
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11										600					
12		$\neg \neg$								400					
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15										715	T				
16	1	1								600	\top				

CASE		Insula	tion				Sk	in		T.	Boundar	Condition			Initial Condition	See
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27				1							715	1				
28	 			1		T					600	1				"

 $k = Btu/hr.ft.^2 (°P/in.); C_p = Btu/lb. °P; y = lb./in.^5; X = inches$

^{0 -} Minutes; t - °F; h = Btw/hr.ft.² °F

^{*} Insulation: Johns Munville Min K

Figure 9.116 Heat Transfer Model for Insulated Metal Skin

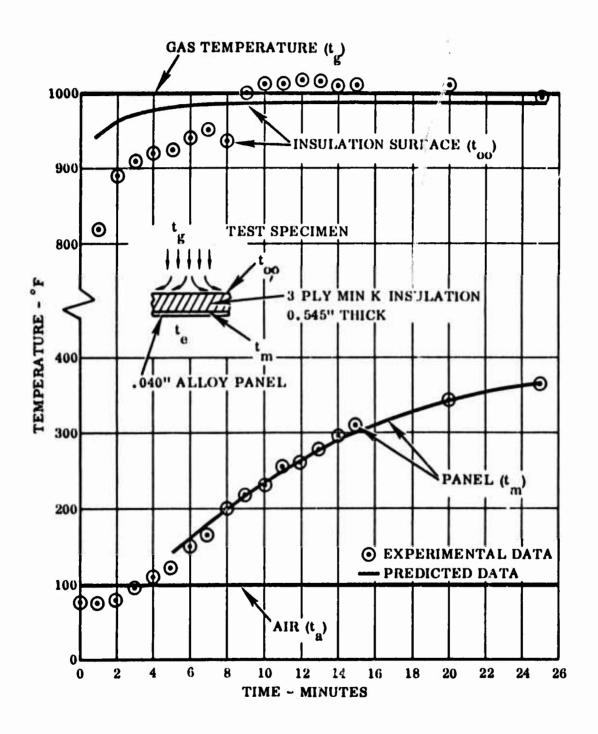


Figure 9.117 Comparison of Predicted and Experimental Insulated Panel Temperatures: 0.545" Min K Insulation on .025" Titanium

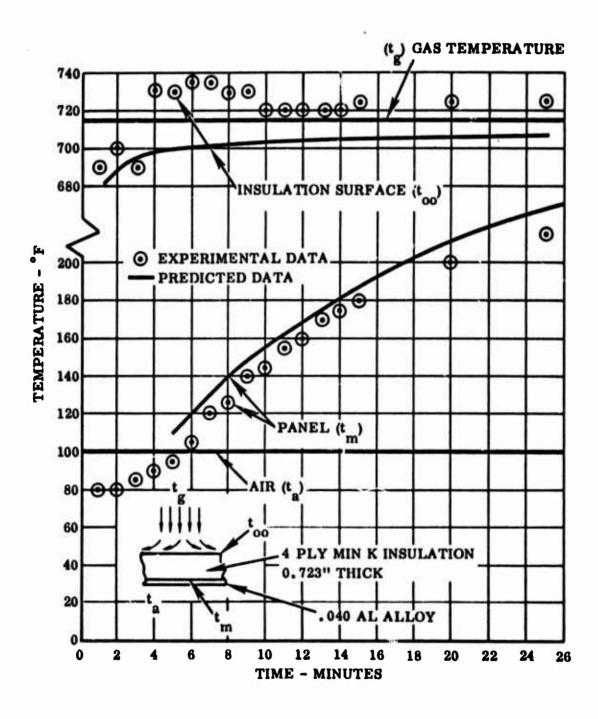


Figure 9.118 Comparison of Predicted and Experimental Insulated Panel Temperatures: 0.723" Min K Insulation on .025" Titanium

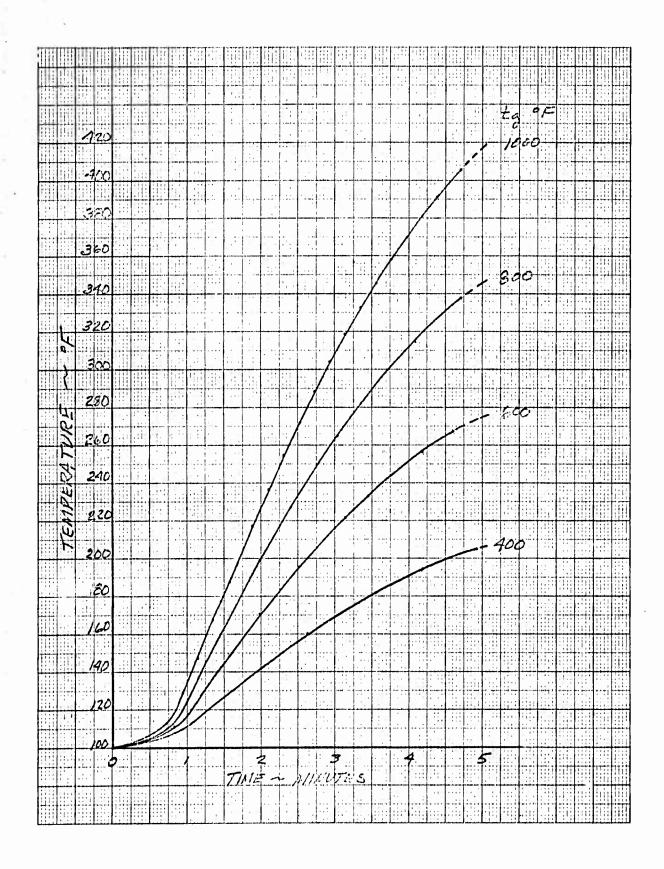


Figure 9.119 Skin Temperature-Time Profiles Vs Gas Temperature 0.25" Min K Insulation on .025" Titanium

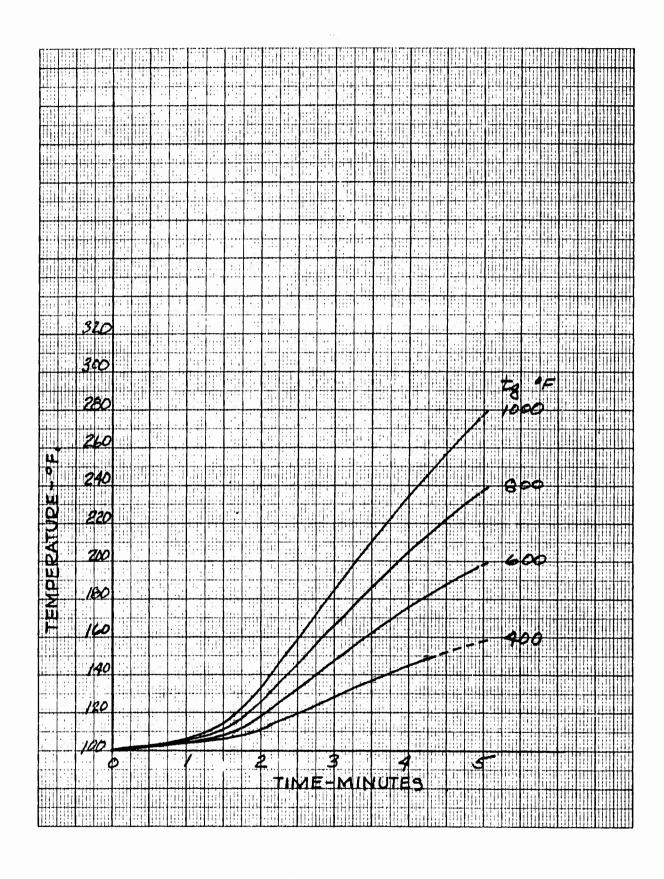


Figure 9. 120 Skin Temperature-Time Profiles Vs Gas Temperature 0.375" Min K Insulation on .025" Titanium

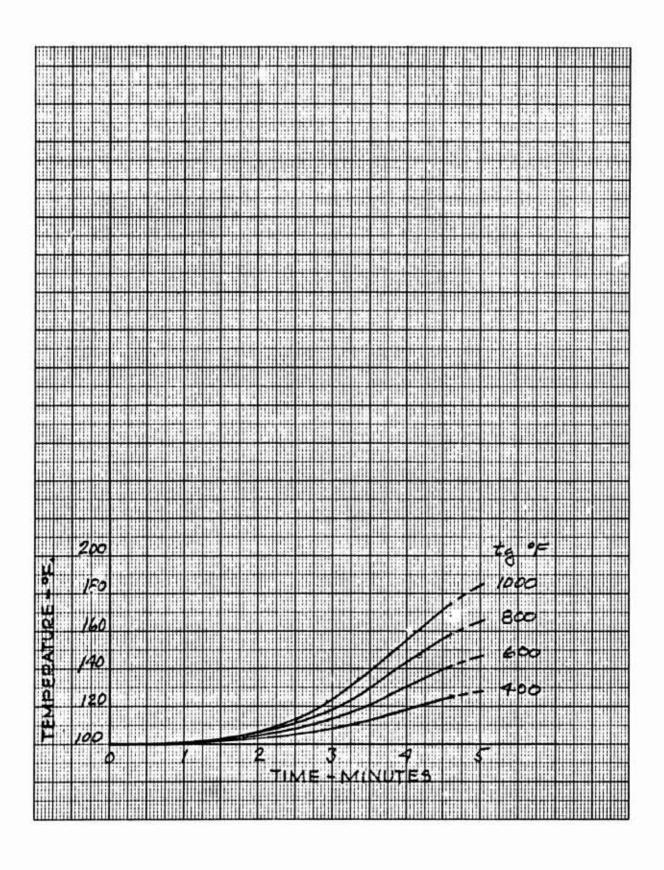


Figure 9. 121 Skin Temperature-Time Profiles Vs Gas Temperatures 0. 50" Min K Insulation on . 025" Titanium

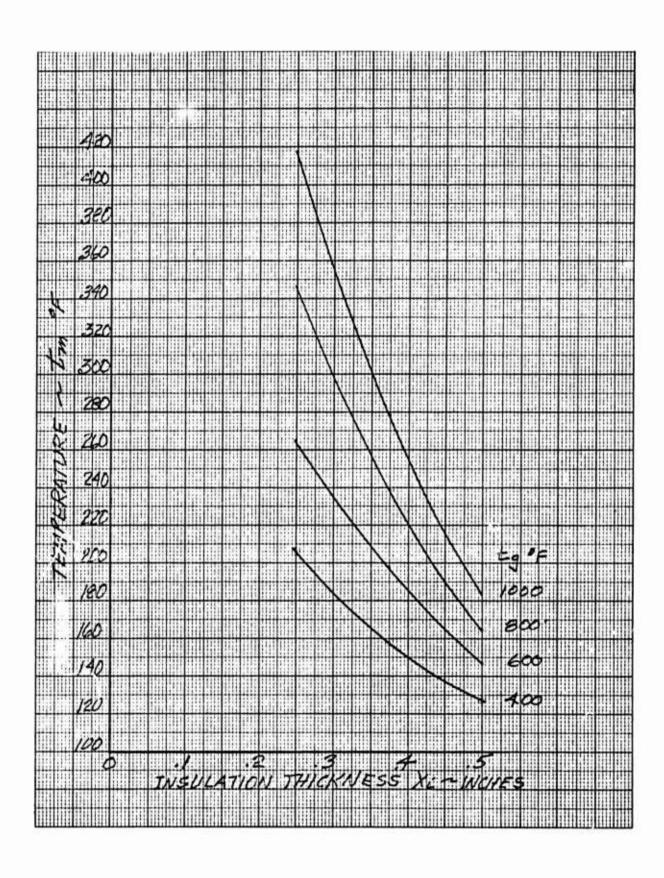


Figure 9. 122 Skin Temperature Vs Insulation Thickness and Gas Temperature After 5 Minutes Exposure

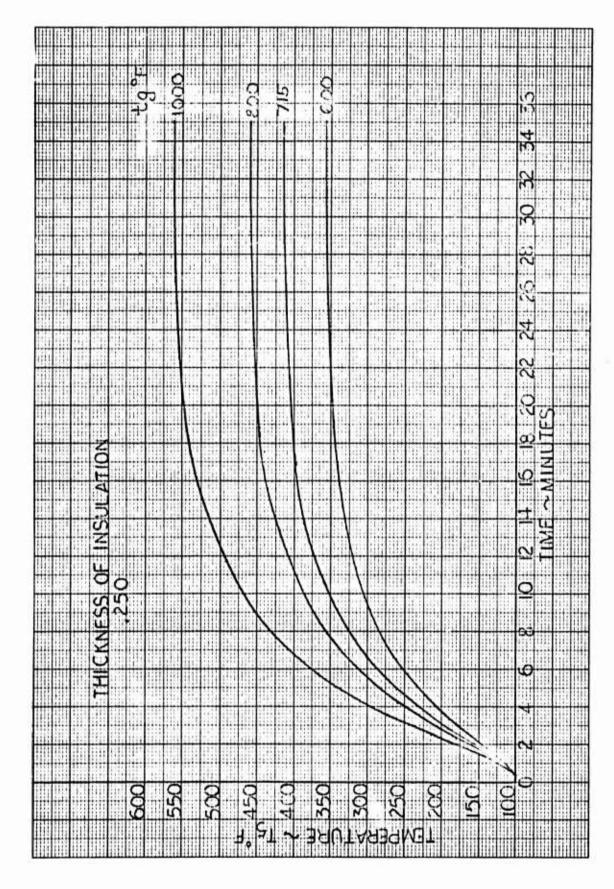


Figure 9. 123 Skin Temperature-Time Profiles Vs Gas Temperature 0. 25" Min K Insulation on . 040" Aluminum

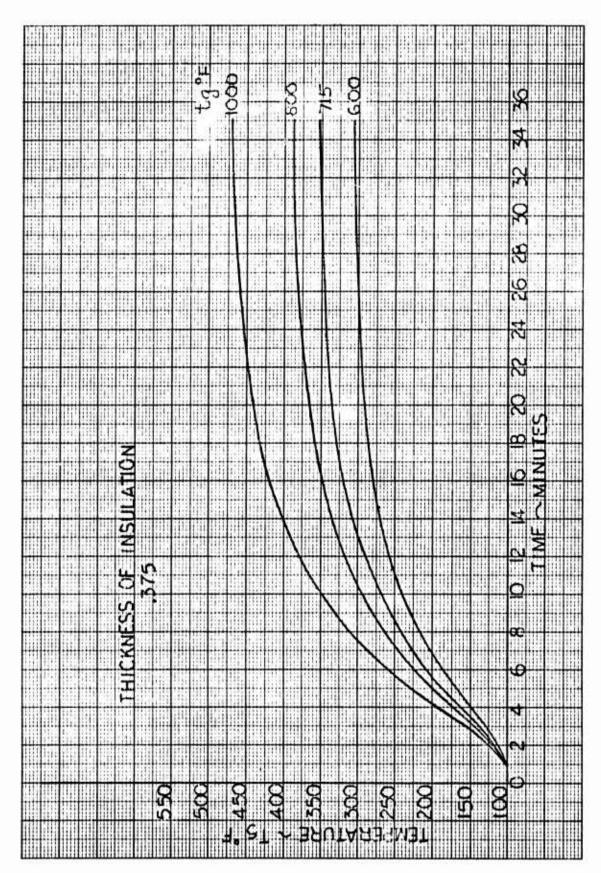


Figure 9.124 Skin Temperature-Time Profiles Vs Gas Temperature 0.375" Min K Insulation on .040" Aluminum

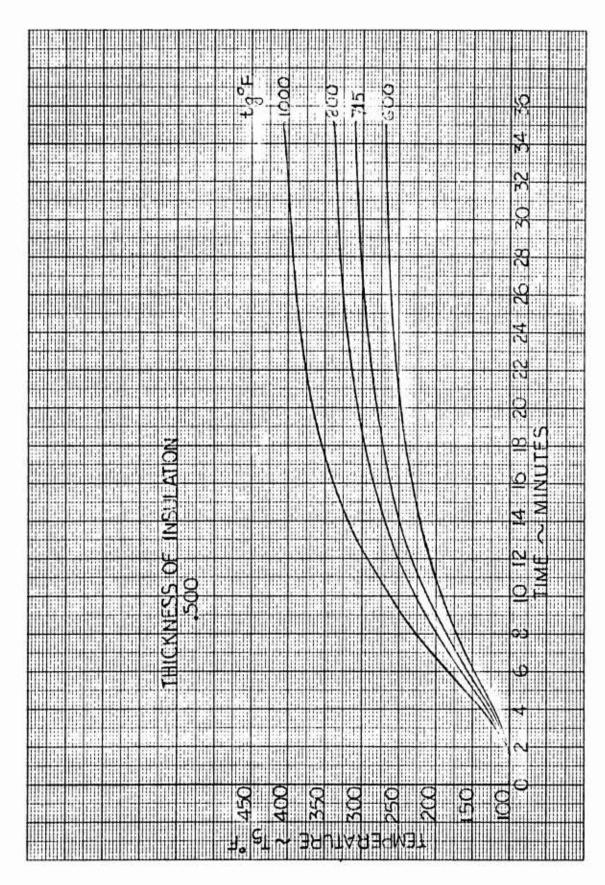


Figure 9. 125 Skin Temperature-Time Profiles Vs Gas Temperature 0. 500" Min K Insulation on . 040" Aluminum

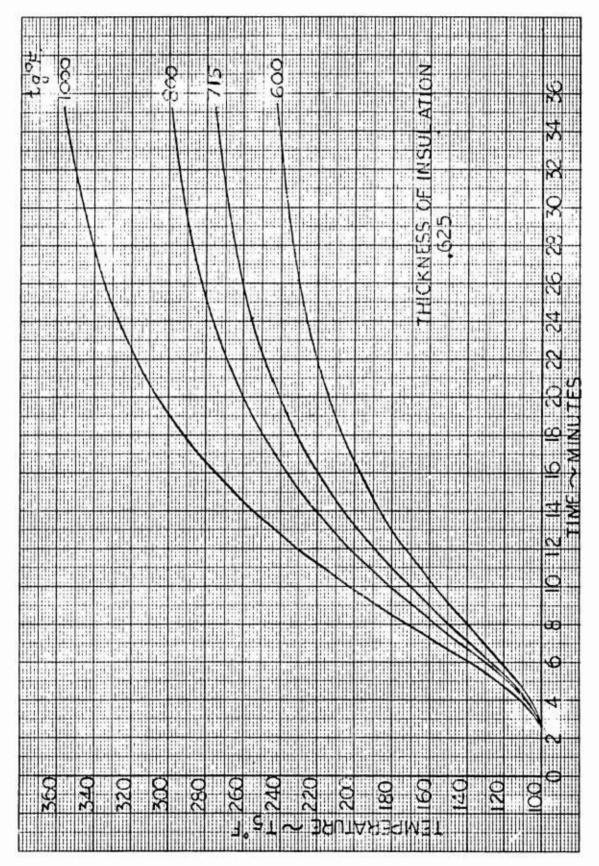


Figure 9.126 Skin Temperature-Time Profiles Vs Gas Temperature 0.625" Min K Insulation on .040" Aluminum

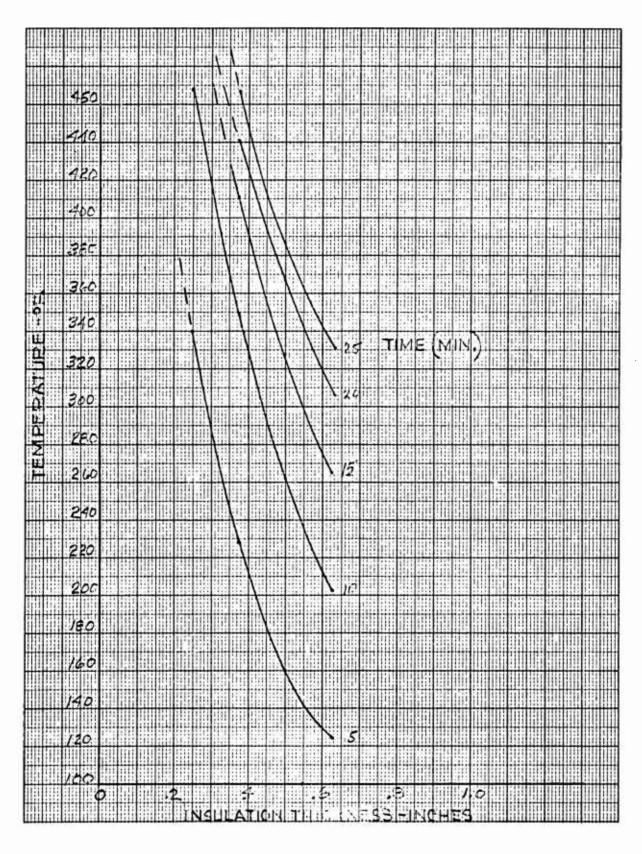


Figure 9. 127 Skin Temperature Vs Insulation Thickness and Exposure Time; Gas Temperature 1000°F, and Aluminum Skin

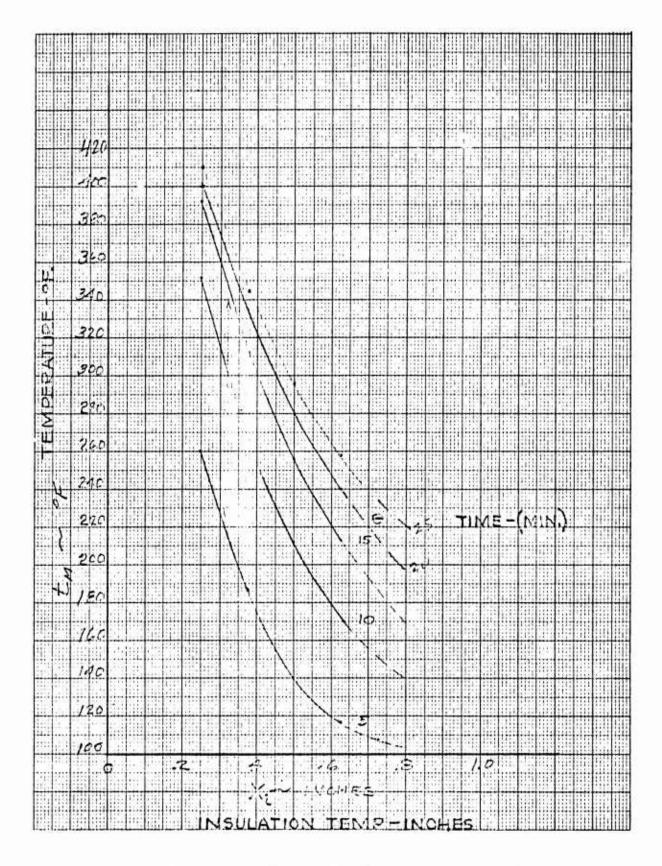


Figure 9. 128 Skin Temperature Vs Insulation Thickness and Exposure Time; Gas Temperature 715° F, Aluminum Skin

ease. The validity of this method of transient analysis was established by comparison of predicted and experimentally determined temperaturetime profiles as shown in Figures 9.117 and 9.118.

As an aid to selection of insulation thickness, temperature-time profiles were calculated for the series of 28 cases summarized in Table 9.11. These data and convenient cross-plots are presented in Figures 9.119 to 9.128.

9.6 NASA-AMES DATA FOR FULL SCALE XV-5A MODEL TEST 177

This section presents available data obtained from full scale XV-5A Model tests during NASA-Ames Test 177 conducted between 6 December 1962 and 18 January 1963. Test 177 was conducted primarily to obtain the aerodynamic characteristics of a full scale XV-5A model which are presented in Reference 19. Therodynamic considerations, particularly structural and evnironmental temperatures, were of secondary concern; however, approximately 24 temperature recording channels were available for gathering the test data summarized in Sections 9.6.5 through 9.6.7. A few unidentified installation photographs are presented in Section 9.6.1. Various other interpretive and supporting data are also presented. In all cases data is fragmentary, however. it represents the best data available at the time critical aircraft design decisions were being made. Mostly, the temperature data was used as recorded, but in a few instances corrections were required as outlined. Conversion of data from one set of operating conditions to another was accomplished by the correlating method of Section 5.3.5.2. The Test 177 data are presented in the following sections without further discussions.

9.6.1 Run Schedule NASA-Ames Test 177

This briefly indicates the test conditions established for the runs of Test 177. Rune 1 - 53 were conducted in the 40' x 80' Wind Tunnel at the NASA-Ames Research Center, Moffett Field, California. Runs 54 - 56 were outside ramp tests conducted at the same facility.

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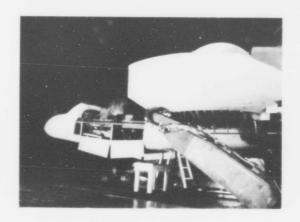
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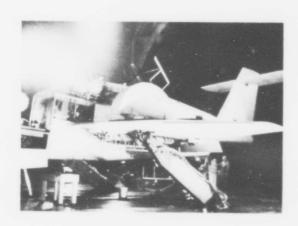
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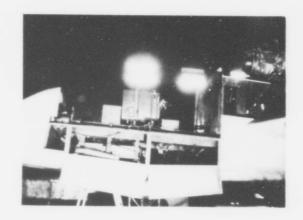
9.6.2 Installation and Model Photographs

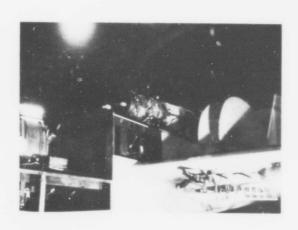
The installation and model photographs presented in this section, included primarily for documentation purposes, show various aspects of the full scale XV-5A Model including the two J85 gas generators for driving the wing fans, the T58 gas generator for driving the nose fan, method of model support, wing fan butterfly doors, louvers, and actuators, flap, tailpipes, thrust spoilers, landing gear environment thermocouple lattice and operators' console.

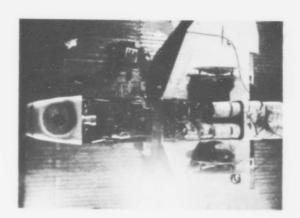








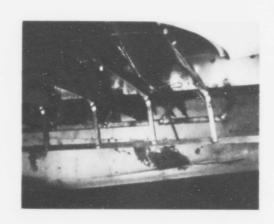


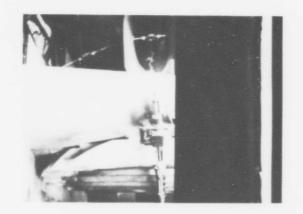










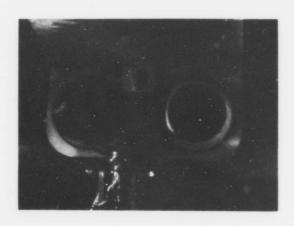




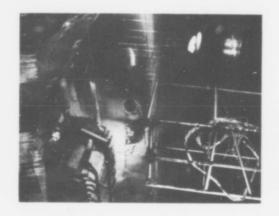


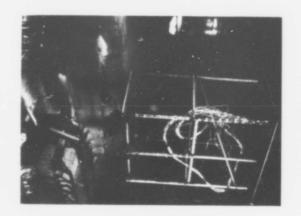


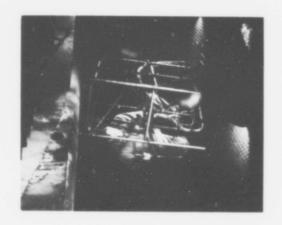


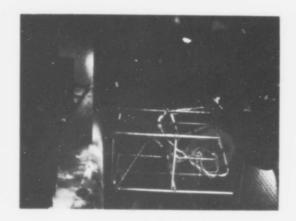


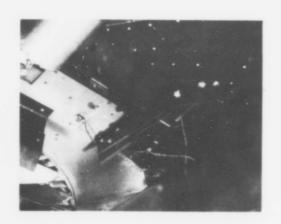


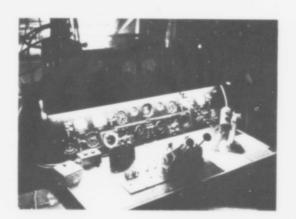












9.6.3 Engine Data

Operators' console data are presented in this section for the J85 and T58 gas generators used to drive the wing and nose fans, respectively. Where no data are presented, it generally means the particular engine(s) was (were) not operating. This may be verified by checking the Run Summary of Section 9.6.1.

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5											1	1					1
9																	
7	-																
8											1						
6																	
9											1						
7												1					
2										1	1	1					
13						٦					1	1					
7											7			İ			
2																	
78																	
17																	
7																	
2											1						
8																	
21																	
2																	
E																	
2																	
Ø																	
8														•			
21														1			
8																	
8						Ī											
8						-											

AMES RAMP RUNS - ENGINE DATA

Date	D 4	Left E	ngine	Right E	ngine	Т-	58	m/n	0	_
Date	Run #	RPM	EGT	RPM	EGT	RPM	EGT	T/R	β_8	β _V
1-14-63	54	79	530	83	480	81	490	40	0	0
	1 1	80	500	81	480	31	480			
	i i	86	500	86.5	430	87	490		į	
1-15-63	55	93	560	95	470	96	620	40	0	0
	56			1		100	630	0	i	
1-15-63] }	94.5	580	95.5	490	93	590	40	G	0
	1	95	620	97.5	580				0	0
	1 1	95	620	97.5	560			Ì	10	
	}	95	620	97.5	560	1			20	
	1	95	620	98	560			i	24	
	1 1	95	620	98	600	1			0	
	!	95	620	97.5	600		l	1	10	ŀ
	1 1	94.5	620	95	560				0	0
	ł I	86	560	86.5	500					1
	1	79	530	41.5	510			ļ		ì
	1 1	-	-	79	450	i			<u>}</u>	1
	1 1	-	-	87	450		1		1	1
		-	-	95	520					1
	l i	-	-	96.5	500					ł
	1	80	490	-						1
	ł	87	500	-						[
		93.5	580	-				ĺ	ľ	l
		94.5	600	-					İ	
1-15-63	57	78	550	81	460				l	ļ
		87.5	560	88	475			1		ł
		95	610	95, 5	500			Ī	İ	
] [96	620	97	510					
1-16-63	58	77	480						l	
	1 1	83.5	500							
	!	92.5	580					1		
		93	580						i	
	1	79	480	80	460					1
]]	84	490	85	960					
		93	600	94	480					
		93.5	620	95.6	550			}		
				78.5	440			{		
				85	450					
				94	530					
	_		.	95	570					ł
1-16-63	59	1 80	520							(
		2 87	520							
		3 96	630							
		4 97	640	40.	440					
		5		81	440					
	l I	6	1 1	86	440			l		1

AMES RAMP RUNS - ENGINE DATA (Cont.)

Date 1-17-63	Run #	7 8 9 10 1 2	78 86.5	EGT 500	94.5 96.5	EGT 480	RPM	EGT	T/P	βε	β _V
1-17-63	60	8 9 10 1	86.5		96.5		THE NO				
1-17-63	60	9 10 1	86.5								
1-17-63	60	10 1	86.5			500	E-2003				
1-17-63	60	1			82	450			8.41		
1-17-63	60		60	550	86.5	460					
		9	80	520	80	500	77.5	480	40	0	0
			85.5	520	86		84.5				1972
		3	94		95	1.5					S 400
		4	81	530	81	500					
		5	88.5	550	88. 5	520					
		7	77	520			2				
		8	85.5	630					T-10		
		9	94	600		- 1					
	1.00	10	94.5	620			1000				
	7 3 1	11			79	440					
		12	1		85.5	430	and the				
		13			94.5	480		10.00			1 -
		14			96.5	500					
1-18-63	61						1.76	14.4			
T . = 46°	- 7	1	78	520	78.5	460	78	430	100		
T _{amb} = 46°		2	85	520	85	460	89.5	500			12 - 1
		3	94	600	94	480	99	640			
		4	94	600	97	500	89,5	640			
		5	94	600	98	500	101	630			
		6	94	600	98.5	500	101	630			
		7	78		78		101	630			
		8	86	560	86	500	7.4				
		9	94	600	96.5	580					
	45.4	10	78	460	80	440	7			1	
		11	85	500	85.5	450		1 11			
		12	93.5	600	95	460					
		13	94	620	96	480					
1-18-63	62	-							()		
2.1	·-	1	78	500	80	440					
Tamb = 50°		2	85	500	86	470					1.0
		3	93	570	94.5	540					100
		4	94	600	96.5	600					
		5	76	610	30.0	500	1				
		6	84.7	520							
		7	93.5	580		4 77	7.5				
0.00			94	600	M. C.	21.					
		8		000	78.5	440					
		10			86	440					
1818		11			94.5	490			1		
		12			95.5	520					
	4	14			70. 0	020					

TIME AFTER RUN #62 1-18-62

Left Engine	37:19
Left Fan	33:31
Right Engine	36:42
Right Fan	33:00
T-58 Pitch Fan	17:34

T-58 PITCH FAN DATA

Run#	N T-58	EGT	NPF	T ₂	
54	70	480	1550	50	
1-14-63	75	460	1680		Bleeds cut off
	80	480	2000		
10.1	81	490	2050	40	
	87	495	2400		Shut down for fuselage panel repair
55	74	400	1820		
56	95	630	3350		
57	96	630	3400		
	95	620	3350	100	The second second second
	96	620	3400		
	99	620	3750		Pitch fan by itself with no ingestion

9.6.4 Test 177 Summary - Thermocouple Locations and Identification

The information included herein is applicable to the reduced temperature data of Section 9.6.5.

Test 177 consisted of approximately 62 runs, some of which involved simultaneous operation of lift and pitch fans. Only one data point (point 4 of run #41) was made at wing lift fan speeds above 1750 rpm - for this point the speeds were both 2410 rpm.

Data have been reduced for runs 18, 19, and 21, (16 and 41 partially). At the present, two rolls of temperature records are in the San Diego plant, but these are Ames property and will be returned soon. These records represent the temperatures of runs 25 and on.

The data available from these records are: gas temperatures in wing voids around both fans, internal fuselage gas temperatures, gas temperatures on the under side of the wing and flap (L/H), J85 engine inlet temperatures, wing fan inlet temperatures and on the last run in which the thrust deflector was installed, seven gas temperatures from the aft fuselage.

Thermocouple Locations for Ames Test 177

Run 11

Description		STA. L.	W.L.	B.L.
	[16	296(296)*	96(96)*	32
	17	296(296)	96(96)	46
Gas temperatures below	18	310(302)	98(97)	32
L/H wing surface with	19	310(302)	98(97)	46
45° flap	20	319(312)	91(99)	32
	21	319(312)	91(99)	46

* Values in parentheses are STA. L. and B. L. for thermocouples if the flap was retracted.

Gas temperatures	1	271	+13	31
Landing	2	286	+13	31

Gear	3	301	+13	31
Rake	4	271	-7	31
	5	286	-7	31
$(h/_{D}=1.7)$	6	301	-7	31
	7	271	+13	51
	8	286	+13	51
	9	301	+13	51
	10	271	-7	51
	11	286	-7	51
	12	301	-7	51
Run 18				
Description		STA.L.	W.L.	B.L.
Gas temperatures below				<u>D. L.</u>
L/H wing surface with	ide	ntical to Run	11 data	
45° flap				
Gas temperatures	1	271	63	31
Landing Gear	2	286	63	31
Rake	3	301	63	31
	4	271	43	31
$(h/_{D}=1.0)$	5	286	43	31
	6	301	43	31
	7	271	63	51
	8	286	63	51
	9	301	63	51
	10	271	43	51
	11	286	43	51
	12	301	43	51

A Y N 64B017

Desm	1	0
Run	4	0

		STA.L.	W.L.	B. L.
	16	182	156	15R
	17	182	147	0.0
Engine Inlet Duct	18	182	156	15L
	19	182	147	15L
	20	182	156	0.0
	21	182	147	15R
	12	208	134	0
	13	218	147	0
Fuselage Temperatures	14	257	110	0
	15	269	126	0
Wing and flap lower	1			
surface surface gas	iden	tical to Ru	11 data	
temperatures	}			
Run 21				
Landing	(1	271	-23	31
Gear h/d =2.2	2	286	-23	31
Rake	3	301	-23	31
L/H wing	6	216	100	61
Gas	7	236	100	41
(Internal)	8	278	100	41
	l 9	294	100	61
Fuselage Temperatures	ider	tical to Ru	19	
Engine Inlet Temperatures	ider	itical to Ru	1 9	

Run 41				
Engine Inlet Temperatures	ide	ntical to Ru	n 19	
L/H Wing Gas Temperatures	ideı	ntical to Ru	n 21	
Wing and flap lower surface Gas temperatures	ideı	ntical to Ru	11	
Run 45				
		STA.L.	W.L.	B. L.
Aft fuselage gas temperatures	6	430	100	T.P.Q
(about 1/2" clearance from	16	394	94	22
skin)	17	430	110	19
	18	394	110	22
	19	376	110	23
Д	20	406	110	21
	21	376	94	23
Run 56				
L/H Flap-Ext.	6	324(324)	87(100)	25
	16	315(207)	92(99)	43
	17	324(324)	87(100)	61
	18	308(303)	98(98)	43
	19	308(303)	102(103)	43
	20	324 (324)	87(100)	61
	21	315(207)	92(99)	43
L/H Wing - Fwd.	3	214	106	61
	16	214	94	43
	17	214	107	43

L/H Wing Int. -

identical to Run

Fuselage - Internal

10 132 89 0 11 230 114 0 12-15 identical to Run

Addenda to Thermocouple Location

Run 11

Landing gear thermocouples - the beads were separated from the frame by 1/4" to 1/2". Two comments - 1) possible error due to radiation, 2) high gas velocity brought about rapid temperature changes and rapid attainment of equilibrium

Wing gas temperatures - 16 and 17 were separated from skin by 1/4", 16 under wooden skin and 17 under steel skin. 18-21 were separated from steel skin by 1/16 asbestos paper, but heat-sinked by brass mounting screw to skin. 18-21 were washer-type thermocouples

Fan Inlet temperatures - G. E. had 8 thermocouples installed on each fan, 2 on the upper side of each strut. These fan inlet temperatures should be considered suspect for two or three reasons. First, they were mounted inside a shield intended to give them the stagnation temperature, however, the shield assumed the same temperature as the hot strut causing a radiation error. Note that the temperatures rise steadily. Second, instrumentation technicians did not keep ice in the reference bath, so the data are also in error due to reference drift. Third, the technicians did not always recognize individually each of the eight thermocouples. Much later (for the ramp test) these thermocouples were replaced by true gas temperature couples, but this data is available only from G. E., because the data were recorded on an oscillograph to be reduced at Evandale.

Engine Inlet - Six inlet thermocouples were recorded, but detailed interest was expressed after these data had been reduced. Where a max, or min, is listed, it refers to the six temperatures.

Fuselage Maximum and Wing Maximum - The same can be said for these measurements as was said for engine inlet.

Run 18 No change

Run 19 The engine inlet duct temperatures are recorded separately.

The beads extended into the throat of the inlet about 2" (from the side) at a point about 10" aft of the inlet mouth.

Fuselage Temperatures - Bare thermocouple beads supported by ceramic separators and held 1.5" to 3" away from metal structure. All couples (10-15) are on the ship centerline inside the fuselage.

Wing and flap temperatures - same as Run 11.

Run 21 Landing gear rake - Same as Run 11.

L/H wing gas - listed separately for the first time. These couples were bare heads mounted midway between upper and lower surfaces and about 2" to 6" from the fan.

Fuselage temperatures - Same as paragraph above.

Engine inlet - Same as Run 18.

Run 41 Engine inlet - Same as Run 18.

L/H fan - Same as paragraph above.

Wing and flap - Same as Run 11.

- Run 45 Aft fuselage temperatures 7 couples were bare beads mounted 1/4" to 1/2" from skin.
- Run 55
 and 56

 L. Flap Ext. No. 6, 16, 17, 18, and 19 were bare
 bead 1/4" from steel skin. No. 20 and 21 were beads that
 were clamped between steel skin and the aluminum clip that
 held the associated air temperature couple. Although these
 beads were heat-sinked to the skin and were probably
 closer to skin temperature than air temperature, they were
 not as accurate as a washer with a large contact surface
 would have been.

9.6.5 Reduced Temperature Data

The reduced temperature data presented in this section are identified in Section 9.6.4 and are augmented by the data in the following Section 9.6.6.

CORRECT

TILE LIVINGE Tie. E/Hi ers)

253¥	192 177	392 385	230	23%	302	3	303	355	126	533	325	260 472 322	0,	275	342	12	7,7	959	321	327	523	343 323	300 199 397 345	
SASA CASA	13		494 233	12 154 212 236	194 192 508	324 236 210 216 244 246 234	265 220 255 281 283 491	307-38	3	5	229 159 165 161 5071325	472	77	245.206.235.447.273	422	20 240 422 350	256 412	39	63	7.5	327		597	1
7 1/2	101	35 55		7	192	577	253	13			0	C92	32	3	212	340	33		273	3.5	35;	50	661	
343	511	38	122 116	77	8	3	182	53	551 03	413 139	63		12	18		à.	30	20	3		53	3	3	3
TEMPS BELOW WING SURFACE 7 16 19 1232	93 115 107	102	0	126	3	9	35	5			R	3	162	45	742	736	286 336	282 370 235	ECS 6351325	349 419	352 453	726	7	3
5 S S	3				8	3	100	22	235 155	308	23	72	10	221	8	9	262	8	150	5.	24.6	7	25C 254	į
TEMP WING 17 16	8	105 165	1	2	3	100	153	305 225 283 305			4	ह	3	1 12	244 294 242 244	4	3	239	37.2	65 2		2		1
SAS S	5	3	234 114 172	254 133 152	315,180,195	24.2	351 2	353	261 215	SS 33	207 4	355 304,224 260,254	286 754 775 762 754 754	175.67	378.2	47.4 744 Sae	446 736	782 2	513 237 237	568 265 243	EE+393	576 276 274 182 444 195	432,257	1
ι <u>υ</u>	-	- - -	12	12	"!	ļψ.	10	16.	13	.,5	12	177	12	1	4,	7	**	- T	·v	N	3	.v	4-	1
. 12	3	112	8	3	2	32-	Ł	8	4	72	76	26	98	8	8	Ş	3	4)	6,1	76	50	75	8	1
		4	50.20	S T	9	35	3	2	74 74	72	63		8	42.2	3		3	43	1/2	76/	000	15	8	1 8
0	212 54 64 64 170 186 264 64 64	2	ま	GE 182 274 220 152 192 242 262 140 80 136 190 184	SIE OIC OT	230-225-82	246 144 84	280.116	4	72	140 165 184	8	35. 336176 256 234 116 110 232 100 102 196	342 42 342 350	8	3	8		****	1	78	79.	3	6. 6. 6. 6.
6	3			2	28	22.2	52 2	34	8	8	38	26	32.1	23	3	8	3	3.5	75 140 76	114 145 77		3	5	
. 0	8	3	8	8		82	8	8	78		2		3	20	8		3	\$2	9	4	ē	23		T:
	2	92 196 260 122	8	3	7	33	98	8		178 74	8	R	8	1	3	G. C.	000	l qg	76	8	114 112 176 172	3	8	
PS PAKE	7	20	220	42.2	33.2	30.2	8	72	75 186	2	8	2	1	122	3	8	3	9	12	76	4	78.5	000	
M \ V \ \	4	\$	222	32.2	25, 23, 294. 30	310-230:238	12.2	315.172.190	8	72.7	78.2	2	55.2	32.2	0	3.	3	3	2.	12	40	78.7	3	
CAND SAS TEMPS	4	2	8	1:25	74.2		230 256-312 230 185			80 74 72 72	Ω	77	16.2	FS1 222 255 TOS	3	Se : Se	3	ii.	76	77 7	29	76.7	23	5
102	12 6	3	22	20.16	*	18.2	200	304 533	40 82	S	8	8	18	1965	8	82 8	42	10 10 10			2	727	3	-
S	2 02	150 244	242	742	X	3	8		i	4	72.2	523	3	67	3	8	38	12.5	1.7.146	172.15	12.	80.122	3	-
3	3	3	106 264 232 106 122 202 206 725 68	32.28	232.254 154 174	350-304 178, 224	34 28	196: 94	95 36	6:74	21C 272 230 KD 178 285 235 715	352 222 356 244 222 376 155 92	262 €	100	8	(a)	52 3	52.	76	53.	155 162 100	33		EO 199131 60 62 63 61 61 103
12		+	5	9		.0	17	72.2	5.	75:76	7.6 2	77	73 2	73 3	3	(4) (4)	3	20	15	5 5	3	79 8	8	0
PUGT NIET NEX KIN	67.62	12 2	25 65	10	109	50	78	79.7	52.7	4-0- 00	* ***		8		643	83	83 8	85.8	5	22 7	÷40		40	0
	8	56 7		X	9					7:0	63.63	7.85		7 100	4				3		8	15 8		-
RICET	E	121 3	90 99	8	43 106	147 108	52 106	13	51113	= 8	32.1	100 117	230 147	243 147	203104	208 IOB	216117	216.139	56.04	52 54	69	32 111	134 113	6
O.	35.		27	=	Ā	7	3	1 2	3	50 156 117 53	30 182 119	r.	0	12:2	35 2	çi.	2	7	=	3	<u></u>	F	5	1 0
Υ.		0	4	ق	10	Q	5	4	0.	l in	(4)	=	-	-	4	O	4	9	6	0	0	4	0-	U
	*	** ****	4		100						-				1.					9				-
TUN TUN: (大野) (野)	3	. 66	5	3	5	۲	-72	73	4	75	2	11	. 78	8	ō	22	8	33	7.9	6	8	8	62	2
375	3-		-) - 10 100 100 100							<u></u>	-1	8			_		1	<u> </u>	-		
デ/25									-				:	- • -	romort a								+	
TANA NA	0-																							
RIGHT ZPN	0071	1700	000	Š	5	STS	1680	S.30	3	1740	1710	28	5	1720	1705	8	$\bar{8}$	8	8	9	8	01/1	1700	1220
FAN	1700	01.51	069: O1Li	01.	1700 1675	1700 675	3	Sc		1720 1740	017	00T1 00T1 . T201	0571 0571 2,0011	1700 1720	2071 CETI 217011	COT! 0271		0071 2571 2, 2111	00T1 00C1	.0071 0691	OOT! OIT!	1730	0021 0221	24 12534 17501730
		7,210	1		1046'2	7, 1701			0221 7,750				2,0		7,4	1	11101/2 1720	2,2						17.6
4 I	2	3	<u>₹</u>	1045	ō		640	1050	0	1059	50			1103		011			1245	1246	1247	1249	:252	3
POINT	-	2	'n	4	'n	19	7	10	_O	Ö	=	2	5	4	7	3	5	$\overline{\omega}$	ō	20	5	22	23	C

RUN 11 $H/D_f = 1.7$

RYAN EST

Rdg.	α	β	v _p	N _f	N _{Pf}
42	0	-12	30	1700	0
43		0			4.75
44		15			
45		30			
46		40			
47		50			
14		-12	40		
13		0			
12		15			
11		30			9
9		40			
10		50			
41	3 1	-12	60		
40		0			
39		15			
38		30			
36		40			
37		50			
27		0	80	4.7	
26		15			
25		30			4 10
23		40			100
24	- 2	50			

RUN 11

Bv	Temp.	Rdg.		6		17		18		19	2	0	13	21
								T						ĭ
12	-65°	42	292	227	282	217	244	179	250	185	232	167	238	173
0	-62	43	306	244	276	214	240	178	252	190	230	168	236	174
15	-46	44	408	362	304	258	244	198	264	218	250	204	252	206
30	-45	45	260	215	242	197	242	197	220	175	208	163	192	147
40	-57	46	140	83	130	73	242	185	130	73	98	41	96/	39
50	-64	47	600+	546+	328	274	374	320	112	58	276	222	86	32
-12	-50	14	258	208	250	200	200	150	224	174	206	156	212	162
0	-63	13	272	209	270	207	212	149	248	185	240	177	240	177
15	-75	12	386	311	302	225	222	147	258	183	252	177	258	183
30	-70	11	204	134	138	68	226	156	156	86	182	112	168	88
40	-75	9	256	181	210	135	230	155	150	75	144	69	130	55
60	-90	10	600+	510+	328	238	306	216	160	70	410	320	136	46
-12	-95	41	262	167	266	171	256	161	252	157	246	151	248	153
0	-95	40	272	177	272	177	250	185	256	161	244	149	250	165
15	-75	39	256	181	144	69	246	171	200	125	156/	81	178	103
30	-85	38	396	311	170	85	282	197	274	189	226/	141	240	155
40	-90	36	474	384	222	132	274	184	270	180	270	180	208/	118
60	-106	37	600+	494+	344	238	362	256	376	270	574	468	430/	324
0	-41	27	252		172/	133	268	227	208	167	144	103	174/	133
16	-44	26	292	248	178/	134	288	244	256	212	190	146	214/	170
30	-52	25	398	346	230	78	302	250	342	290	326	274	296	244
40	-50	23	428	378	254	204	280	230	250/	200	296/	246	196	146
50	-75	24	600+	525+	356	281	304	229	400	375	566	491	412/	335

RUN 11

	ð	tCORR.	16	17	18	19	20	21	α
	1	-35	153	81	139	81	93	85	-4
	2	-48	124	70	130	66	120	70	0
40V _K	3	-59	187	67	125	63	75	70	+4
	4	-61	235	89	133	77	93	105	6
0 =	5	-69	259	121	139	103	125	133	8
$\beta_{\rm V} = 35^{\circ}$	6	-80	252	164	138	144	172	174	10
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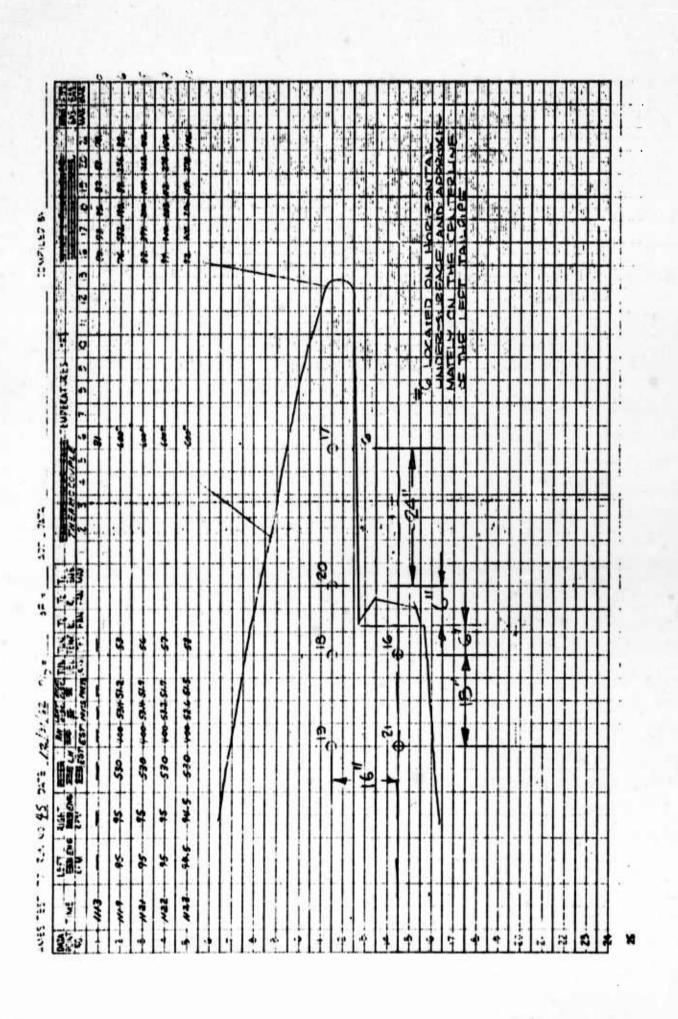
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9.6.6 Aircraft Temperature During XV-5A Model Tests

This section reproduces unpublished data NASA-Ames Test 177 temperature presented in an Evaluation Memorandum prepared by General Electric Company.

EVALUATION MEMORANDUM

TAE - E.M. #110

SUBJECT: AIRCRAFT TEMPERATURES DURING

XV-5A MODEL TESTS

DATE: FEBRUARY 13, 1963

AUTHOR: B.C. alford pel

cc: AP Adamson ED Alderson

DE Clark & Staff

RT Hacnel LC Jensen

WR Morgan & Staff

RH Goldsmith & Staff

JT Kutne"

WB Campbull

Aircraft temperature data gathered during the XV-5A model tests at Ames are

The data from Runs 11, 16, 18, 19, 21, 41, 45, 55 and 56 were collected and reduced by Don Fisher of Ryan. Additional data are included from Runs 59 through 62.

The following statements explain the tabulation of data. (see Figure 1)

Fan Inlet - Obtained from G.E. fan inlet thermocouples. Run 11:

> Engine Inlet - Maximum & minimum temperature from six inlet thermocouples (#16-21) are presented.

Landing Gear Rake - Numberang is as follows: (see Figure 2) on the in....d side, the upper deck is numbered from forward to aft 1, 2, 3 and the lower deck 4, 5 and 6; on the outboard side, the upper dock is numbered from forward to aft 7, 8, 9 and the lower deck 10, 11 and 12. Note that at a model heighth of 1.7, this rake is not in a true landing gear position.

Loft Hand Wing - Six thermocouples were placed on the wing and flap as shown in the diagram on the data shoet for Run 19. Numbers 16 and 17 were free air temporatures, but the remaining four on the flap were slightly sinked to the flap sink by their mounting screws.

Wing Gas, Maximum - Maximum of the nine internal wing gas temperatures (#1-9). The number recorded was usually 5 or 9.

Fuselage Gas, Maximum - Maximum of six internal fuselage gas temperatures (#10-15). The number recorded was usually 14 or 15.

Hun 16: Tomperature-Plate survey as diagramed on data sheet.

TiL and TiR - Taken from analog sheet Ti Eng - average of engine Run 18: inlets 16-21. (see data sheet for Run 19)

> Landing Gear Rake - Same as for Run 11 with the addition of 13 which is the temperature at the outboard-aft-flogr corner of the rake. At H/D = 1.0, these data represent true landing gear temperatures

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E.M. #110
Page 2
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Wing Surface; Wing Gas, Maximum: Fuselage Gas, Maximum -Same as Run 11.

Selected data points as shown. Inlet duct and fuselage gas Run 19: temperatures are recorded individually as noted.

Includes: Three landing goar rake temperatures - Four L/H Run 21: wing gas gemperatures. Two fuselage temperatures. Six engine inlet temperatures.

Run 41: Data presented must be completed from the Ames run sheets.

Primarily aft fuselage temperature data. Run 45:

Ramp Test. Figure 3 shows location of the following types of Run 55: thermocouples.

5 - Type A - Gas temperature forward, L/H fan; - Type A - Gas temperature forward, L/H fan; exterior of wing.

- Type B - Gas temperature interior left wing.

- Type C - Gas temperature interior fuselage.
- Type D - Gas temperature exterior, L/H flap and trailing edge.

- Type E - Flap skin temperature.

4 - Type F - Gas temperature interior right wing.

Run 56: Same as Run 55.

Run 59: through Run 62: The following table explains the data.

5 - Type A - Gas temperature forward, L/m fan; exterior of wing.
 4 - Type B - Gas temperature interior left wing.

Type C - Gas temperature interior fuselage.
Type D - Gas temperature exterior, L/H flap and trailing edge.
Type E - Flap skin temperature.

4 - Type F - Gas temperature interior right wing.

NOTE: There are only (3) copies available of the following:

Figures 1 and 3 Data sheets for Runs 11, 16, 18, 19, 21, 41, 45, 55 and 56

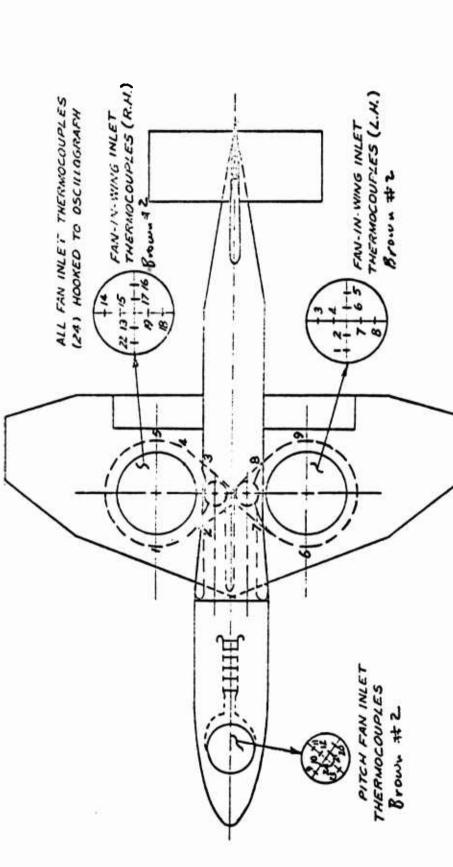
If you would like to see these tabulations, please see G.C. Alford, J.D. Corbett or R. H. Goldsmith

	BROWN #1	BROWN #3
THERMOCOUPLE #	(0-300°F)	(0-600°F)
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₹2	F	-
3	A	-
N 4	-	P .
☆ 5	•	F _
6 · · · · ·	63 int. wrig	6 D) Fleep P B int. wing 9 B)
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9	•	9 B) '
- 10	c)	•
- 11	c c int. fus,	•
-12	c (•
-13	6)	
14	Recorder Temperature Compensator	if c}intifus
15	Model Terminal Strip Temperature	
_ 16	A) .	16 D)
_ 17·	a) ext. wing	17 DIFlas
18, 19.	3	18 D(FEE
10· 20	<u>^</u> !	9 D
20 21	• -	24 2
22	Table In the Commonstrate	21 E/
23	T-58 Inlat Temperature	_
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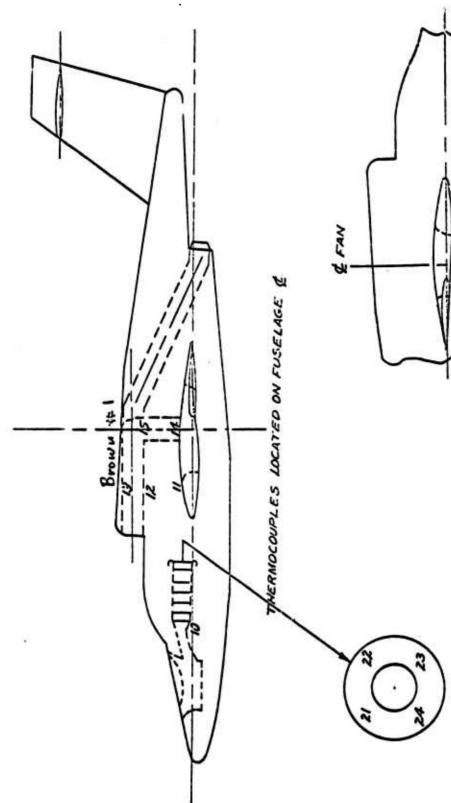
Note 1: All temperatures read on Brown #1 should be corrected by adding ΔT , where; $\Delta T = 15_{(1)} - 11_{(1)}$.

Note 2: Temperatures on Brown #3 should be doubled, then corrected by ΔT added as above.

LANDING GEAR RAKE THERMOCOUPLES "ES TO "35 HOOKED TO BROWN RECORDER



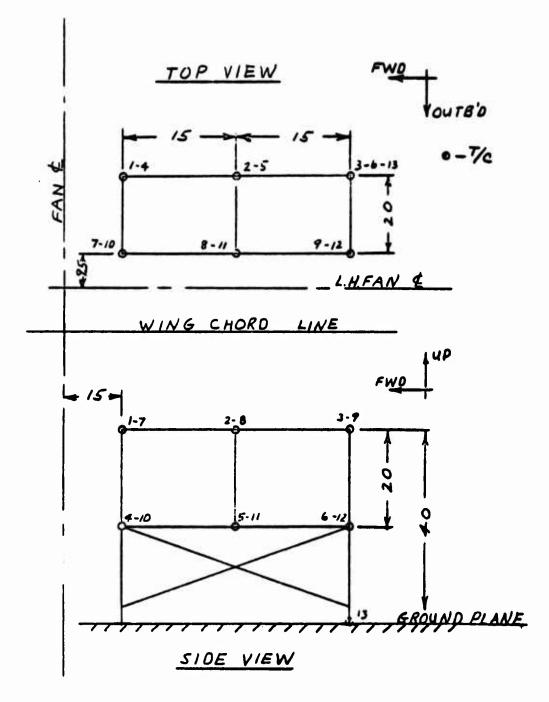
THERMOCOUNTES LOCATED IN WING I 40 AIR SPACE From # !



TISE INLET THERMOCOUPLES

Brown-th. |

THERMOCOUPLES LOCATED ON FUSELAGE SKIN UNDER FRN (RT. SIDE ONLY)



LANDING GEAR TEMP. SURVEY - XV-SA MODEL

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9.6.7 40' x 80' Wind Tunnel and Aircraft Operational Control Data

Test Control data are presented in this section together with occasional summaries of temperature data. Temperature data are identified from Section 9.6.6.

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Run 11 (Sheet 3 of 6)

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Run 11 (Sheet 5 of 6)

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Run 12 (Sheet 1 of 1)

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